

A MEMS thermal shear stress sensor produced by a combination of substrate-free structures with anodic bonding technology

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By combining substrate-free structures with anodic bonding technology, we present a simple and efficient micro-electro-mechanical system (MEMS) thermal shear stress sensor. Significantly, the resulting depth of the vacuum cavity of the sensor is determined by the thickness of silicon substrate at which Si is removed by anisotropic wet etching process. Compared with the sensor based on a sacrificial layer technique, the proposed MEMS thermal shear-stress sensor exhibits dramatically improved sensitivity due to the much larger vacuum cavity depth. The fabricated MEMS thermal shear-stress sensor with vacuum cavity depth as large as $525\mu m$ and vacuum of 5×10^{-2} Pa exhibits a sensitivity of 184.5 mV/Pa and a response time of $180 \mu s$. We also experimentally demonstrate that the sensor power is indeed proportional to the 1/3-power of the applied shear stress. The substrate-free structures offer the ability to precisely measure the shear stress fluctuations in low speed turbulent boundary layer wind tunnels.

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The wall shear stress, also known as skin friction, is defined as the force per unit area that a moving fluid exerts on a surface or wall. Accurate measurement of this parameter is of fundamental importance for the study of fluid dynamics on a solid boundary and the control strategies of aerodynamic flow.¹⁻³ Several approaches have been developed to accomplish this goal. They can be categorized into two basic types, direct and indirect methods.⁴ Each of them has advantages and disadvantages. For the former such as Pitot tube, oil film interferometry and floating element sensors, they are able to measure the tangential force on the wall directly and are easy to use. However, they suffer from contamination by dust and moisture. For the latter such as thermal shear-stress sensors based on changes of physical measurands closely related to the shear stress, they are usually more robust than the former.

Inspired by the rapid development of micro-electro-mechanical system (MEMS) technology, MEMS-based thermal shear stress sensor has emerged as a promising candidate for replacing other conventional techniques in a better understanding of unsteady flow behaviors. This is not only because it can provide higher spatial and temporal resolution, and minimized flow interference, but also it has low cost and high reliability when batch fabricated using silicon planar technique. Over the past two decades, continuous efforts have been devoted to develop various MEMS-based thermal shear-stress sensors employing the sensing principle of heat transfer from a heated thin film element to the fluid flow. In particular, MEMS-based hot film sensors can accurately measure the fluctuating shear stress at a small region and are nonintrusive to the flow. However, they suffer from low frequency and non-uniform phase responses. This is because they are very sensitive to parasitic losses of the supporting substrate. To overcome this problem, the combination of hot film and vacuum cavity structure was commonly used to reduce the total communication between the hot film sensor element and the underlying substrate. To create a thermal barrier which avoids heat to conduct away from the sensing element through the underlying substrate, a membrane with thickness in the micron range should be placed over the vacuum cavity.

The sensor performances depend sensitively on the volume of the fabricated vacuum cavity. Ideally, the depth of the vacuum cavity and the usable area of the membrane should be as large as the wafer thickness and the entire wafer area, respectively. Generally, this is obtained by using so-called sacrificial layer technique from the front side of silicon substrate. Photolithography and reactive ion etching are needed to create access holes in the sacrificial layer, which is then removed to release free-standing structures,



technique has led to diverse applications, the effective release of the sacrificial layer still remains a challenge due to the slow, diffusion-limited nature of chemical etch. The depth of the released cavity determined by the thickness of the sacrificial layer is usually less than 10µm.

In this Letter, we explore the feasibility of a simple and practical scheme for MEMS-based thermal shear-stress sensor with very large vacuum cavity. Instead of using sacrificial layer technique, we generate a substrate-free structure by conventional KOH-based wet chemical etching from the bottom side of silicon substrate, and combine it with anodic bonding technology to form a vacuum cavity. Such a vacuum cavity has a much larger volume than that fabricated by conventional sacrificial layer technique, leading to an increased sensitivity to the MEMS-based thermal shear-stress sensor. The dependences of the temperature change on the vacuum cavity depth were also investigated.

Our concept for the MEMS thermal shear stress sensor based on the combination of substrate-free structures and anodic bonding technology is schematically represented in Fig. 1(a). It consists of a vacuum cavity and a thermal resistor strip on the surface of the isolated thin film. The resistor strip is mounted into flow boundary layer while keeping the surface as flush as possible to reduce the influence of the flow field, where the air flow velocity changes from zero (at the wall) to the value of the mean stream velocity, as shown in Fig.1(b). The resistor is heated by an electric current, thus the heat lost in the air flow is dependent on the air flow velocity. The voltage across the thermal resistor can be measured with a read-out circuit. The measured values of the voltage are then used to calculate the shear stress associated with the mean flow velocity.



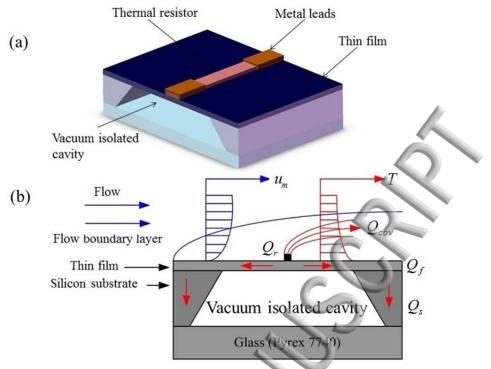


FIG. 1. (a) 3D view and (b)operation principle of the proposed shear stress sensor.

The total heat generated by the thermal resistor, Q_{total} , which stems from the heat stored in the resistor, the heat transferred to and stored on the insulation thin film, the heat transferred to and stored on the substrate and the convective heat transfer of air, can be expressed as 9

$$Q_{total} = h(\tau) A_r(T_r - T_0) + c_r m_r \frac{dT}{dt} + c_f m_f \frac{dT_f}{dt} + \frac{4\lambda_f d_f l(T_f - T_0)}{L}, \tag{1}$$

where l and L are the length of the resistor and the isolation cavity side, respectively, A and d are the effective area and thickness of the resistor strip, respectively, λ represents thermal conductivity of the resistor material, c denotes the specific heat, and m is the mass. The subscripts r and f represent the heat transfer to the resistor and the insulation thin film, respectively.

The convective heat transfer coefficient $h(\tau)$ is a function of shear stress τ , can be written as t^{10}

$$h(\tau) = 0.807 \left(\frac{c_a \lambda_a \rho_a \tau}{w \mu}\right)^{1/3},\tag{2}$$

where w is the characteristic linear dimension (here defined as the width of the resistor strip), w is the viscosity coefficient, and ρ_a is the air density. The subscript a represents the heat transfer to the air.

For simplicity and clarity, here a steady-state approximation is used, i.e., after the resistor is heated, the insulation thin film and the substrate have the same temperature change (ΔT). According to the so-called



blishing ohmic heating formula, and substituting Eq.(2) into Eq.(1), Q_{total} is then given by

$$Q_{total} = U^2 / R_s = 0.807 \left(\frac{c_a \lambda_a \rho_a \tau}{w \mu}\right)^{1/3} \Delta T A_r + \frac{\lambda_r A_r \Delta T}{d_r} + \frac{\lambda_f A_f \Delta T}{d_f} + \frac{\lambda_a A_f \Delta T}{d} + \frac{4\lambda_f d_f l \Delta T}{L}, \tag{3}$$

where U is the voltage across the resistor, R_{c} is the thermal resistor, and takes the form

$$R_{s} = R_{0} \left[1 + \alpha \left(T - T_{0} \right) \right], \tag{4}$$

where R_0 and T_0 are the resistance and temperature under the ambient condition, respectively, and α is the temperature coefficient of resistance (TCR). In addition, the thermal response time is an important figure of merit for the sensor. By applying Laplace transformation to Eq. (3), we obtain

$$t = \frac{c_f m_f d_f}{\lambda_f A} + \frac{c_r m_r d_r}{\lambda_r A} + \frac{c_f m_f L}{4\lambda_f d_f l},\tag{5}$$

Here, Platinum (Pt) was selected as the thermal resistor material due to its high TCR value of 0.39%. The temperature changes versus the vacuum cavity depth are plotted in Fig.2 for the same input power of 0.02W. This figure clearly shows that the temperature change will become larger as the depth of the vacuum cavity increases. Furthermore, the temperature rise of the vacuum cavity is about 20° C higher than that of the air cavity, which means that the vacuum cavity is more desired for the reduction of the heat loss. For the sensor with cavity side length of $200\mu m$, the thermal response time is calculated to be $153.8\mu s$.

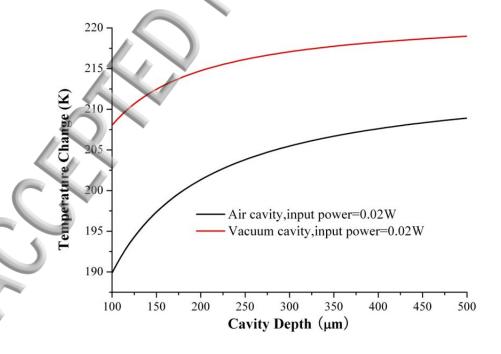




FIG. 2. Relationship between the temperature change and the vacuum cavity depth at the same input power.

As stated above, to avoid the complex sacrificial layer process, substrate-free structures were employed to form the isolated cavity with very large volume. Fig.3 shows the major fabrication steps. The sensor fabrication started with a 4 inch (100) orientated p-type double-side-polished silicon with thickness of 525µm. Each sensor covers an area of about 3 square millimeters, and above 2500 sensors can be obtained per wafer. A SiNx layer with thickness of 1.5 µm was firstly deposited onto both sides of a silicon substrate by low pressure chemical vapor deposition[Fig.3(a)]. Proximity optical lithography and reactive ion etching were then used to open a window on the back side of the silicon substrate[Fig.3(b)]. A Pt resistor strip structure was then formed by standard lift-off process with ultraviolet exposure of photoresist on the front side of the silicon substrate [Fig.3(c)]. Subsequently, aluminum leads with a thickness of 0.3 μm were formed by another standard lift-off process on the front side of the silicon substrate[Fig.3(d)]. It should be noted that a proper alignment is required to assure that the Pt resistor strip and aluminum leads are positioned right during the ultraviolet exposures. The silicon exposed by the back side window was etched away using KOH aqueous solution, creating a SiNx membrane with thickness of 1.5µm[Fig.3(e)]. To protect the aluminum-lead alloy structures during the anisotropic wet etching process, the front side of the substrate was covered by a glass plate and mounting adhesive (Crystalbond™ 509, Aremco Products Inc). After wet etching the backside of the silicon substrate, the residual adhesive was removed by acetone. Finally, the silicon wafer was anodically bonded (450 °C for 60 min with a ramp rate of 30°C /min, a bonding voltage of 1500 V and a base pressure of 5×10⁻² Pa) on a borosilicate glass wafer (Pyrex 7740), covering all of the isolated cavity[Fig.3(f)].



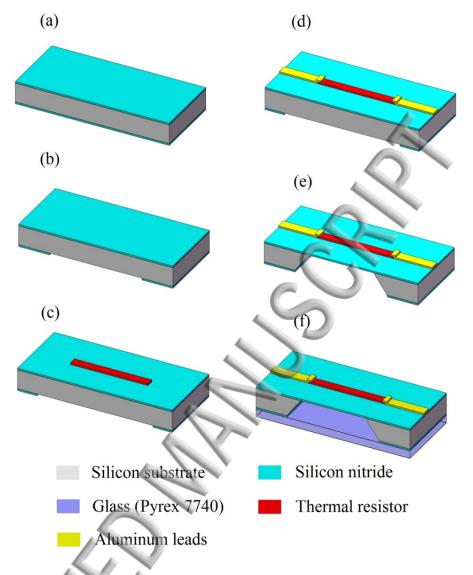


FIG. 3. Schematic of the major fabrication steps.

The pressure of about 5×10⁻² Pa was obtained for the fabricated isolated cavity. Finally, the sensors were packaged in a standard dual-in-line package as shown in Fig. 4(a), and were assembled with a printed circuit board as shown in Fig. 4(b). The schematic diagram of the printed circuit board is illustrated in Fig. 4(c). Fig.4(d) and 4(e) show optical microscopy image and scanning electron microscopy (SEM) image of the fabricated sensor, respectively, clearly demonstrating that no KOH solution was found to corrode the front side of the aluminum-lead alloy structures.



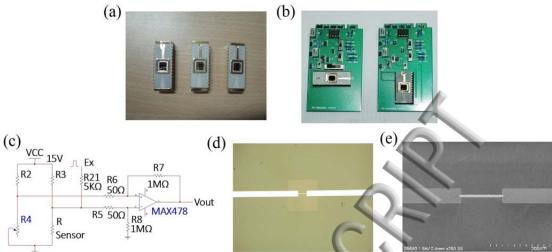


FIG. 4. (a) The packaged sensors. (b) The fully assembled sensors with test circuit boards. (c) Schematic diagram of the printed circuit board. (d) Top view image of the fabricated sensor. (e) SEM image of the fabricated sensor.

To find the contribution of the cavity to the current-voltage (I-V) characteristics, here we investigated three different types of sensors: (i) with vacuum isolation cavity(5×10⁻² Pa);(ii) with air-filled cavity; and (iii) with no cavity. With Keithley 4200-SCS semiconductor characterization system and cascade RF-1 microwave probe station, the currents passing through the resistors were recorded at the same applied voltage. The I-V curve measurement results are shown in Fig.5. At the same applied voltage, the current of the sensor with a vacuum cavity is smaller than those of the two other sensors, indicating that the sensor with a vacuum cavity has the largest resistance value. The I-V curve of the sensor with a vacuum cavity also exhibits a large nonlinearity, indicating that the vacuum cavity has a profound impact on the ohmic heating. For the sensor with no cavity, the I-V characteristic curve is a linear function. This can be attributed to the fact that the heat is mainly transferred within the thermal resistor placed on the silicon substrate. To further understand the effect of thermal isolation, here we define a factor θ affecting the thermal efficiency as $\theta = P/\Delta T$, where P is the input power determined by the equation P = IV, ΔT is the temperature rise in the resistor. The calculated θ values for these three different types of sensors with vacuum isolation cavity, with air-filled cavity and with no cavity are 8.0×10^3 °C/W, 6.8×10^3 °C/W and 2.0×10³ °C/W, respectively. These results demonstrate that the sensor with vacuum isolation cavity offers the best thermal isolation. This is due to the fact that this sensor has a much larger vacuum cavity than those of the two other sensors.



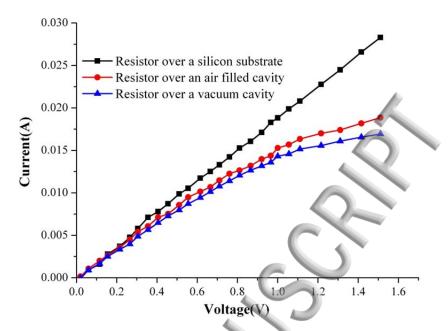


FIG. 5. I-V curves of three different types of fabricated sensors.

Electronic test signals were also used in the determination of time constant of the sensor. By inputting a square wave voltage signal into the Ex terminal of the readout circuit presented in Fig. 4(c), the thermal response time was measured to be 180 μ s, as shown in Fig.(6), which is close to the calculated value of 153.8 μ s. Before applying to a shear stress measurement, the MEMS thermal shear stress sensor calibration was performed based on an empirical relationship between the Reynolds number and the shear stress. The calibration experiment was carried out in a wind tunnel, where the velocity of air flowing can be adjusted by the level of a power supply. Fig.7(a) shows the measured output voltage of the fabricated sensor with vacuum cavity depth of 525 μ m versus mean airflow velocity ranging from 5 to 15 m/s. According to the fluid theory, the shear velocity u is a function of the Reynolds number Re and the mean airflow velocity u. The shear stress can be expressed as u11

$$\tau = 0.332Re^{-0.5}\rho u^2. \tag{6}$$

A relationship was established between the shear stress and the measured output voltage of the sensor with the vacuum cavity, as shown in Fig.7(b). The sensitivity of the sensor is estimated to be 184.5 mV/Pa. To gain further insight into the sensor behavior, we analyzed and fitted the data using the least square



Publishing method. The dependence of the heating rate of the sensor on the applied shear stress in steady state, is given

by

$$U^2 = 0.807\tau^{1/3} + 0.334. \tag{7}$$

Fig. 7(c) shows the relation between the norm of output voltage and the cube root of the applied shear stress. One can observe that the experimental result is in good agreement with the classical theory that the sensor power is proportional to the 1/3-power of the shear stress. Thus, Eq.(3) can be used to correctly predict the trend of our MEMS thermal shear-stress sensor operation.

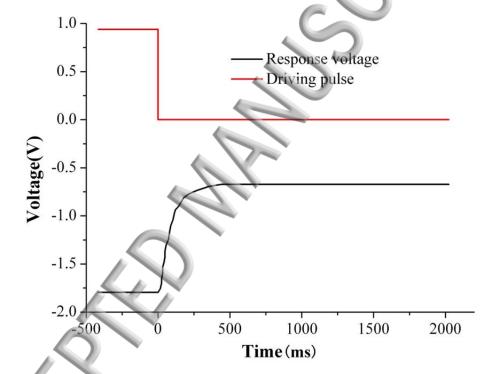


FIG. 6. The thermal response time measurement result of the sensor produced by a combination of substrate-free structures with anodic bonding technology.



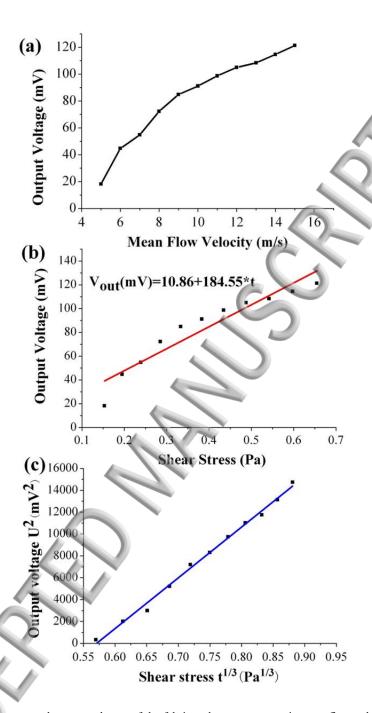


FIG. 7. (a)The measured output voltages of the fabricated sensor versus air mean flow velocity in the wind tunnel. (b) The calculated relationship between the measured output voltage and the applied shear stress. (c)The calculated relationship between the norm of output voltage and the cube root of the applied shear stress.

In conclusion, we have demonstrated theoretically and experimentally a MEMS thermal shear stress sensor with very large vacuum cavity. The basic idea is to use a combination of substrate-free structures



Publishing with anodic bonding technology, first to fabricate a substrate-free structure by anisotropic wet chemical etching of silicon, rather than the sacrificial layer technique, then to form a vacuum cavity with anodic bonding technology. Thus, a vacuum cavity with a depth as large as the thickness of the silicon substrate used can be obtained in a controlled manner, paving the way for the realization of robust MEMS thermal shear stress sensor with dramatically improved sensitivity. A prototype MEMS thermal shear-stress sensor with vacuum cavity depth as large as 525μm and vacuum of 5×10⁻² Pa was fabricated, exhibiting a sensitivity of 184.5 mV/Pa and a response time of 180 μs. Furthermore, our experimental data also confirm that the sensor power is indeed proportional to the 1/3-power of the applied shear stress. Such sensors should have great potential in applications such as the precise measurement of shear stress fluctuations for low speed turbulent boundary layer wind tunnels.

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IEEE J. Microelectromech. Syst.8(1),90(1999).

¹ J. M. Ginder and L. C. Davis, Appl. Phys. Lett. **65**(26), 3410 (1994).

²Z. X. Zhao, M. Shin, J. M. Gallman, and R. D. White, Sensors and Actuators A:Physical **205**, 133 (2014).

³ M. Shikida, Y. Yamazaki, K. Yoshikawa, and K. Sato, Sensors and Actuators A: Physical **189**, 212 (2013).

⁴Chang Liu, Jin-Biao Huang, Zhenjun Zhu, Fukang Jiang, Steve Tung, Yu-Chong Tai, and Chih-Ming Ho,

⁵ M. Bulut Coskun, Steven Moore, S. O. Reza Moheimani, Adrian Neild, and Tuncay Alan, Appl. Phys. Lett. **104**(15), 153502 (2014).

⁶L. Löfdahl and M. Gad-el-Hak, Meas. Sci. and Technol. **10**(8), 665 (1999).

⁷J. Miau, T.S. Leu, J.M. Yu, J.K. Tu, C.T. Wang, V. Lebiga, D. Mironov, A. Pak, V. Zinovyev, and K.M. Chung, Sensors and Actuators A: Physical, **235**, 1(2015).

⁸ Brian J Kim and Ellis Meng, J. Micromech. Microeng. **26**(1), 013001(2016).

⁹ Jonathan W Naughton and Mark Sheplak, Progress in Aerospace Sciences 38 (6), 515 (2002).



¹⁰ J.B. Huang, S. Tung, C.M. Ho, C. Liu, and Y.C. Tai, IEEE Trans. on Instrumentation and Measurement **45** (2),

570 (1996).

¹¹ P.K. Kundu, I.M. Cohen and D.R. Dowling, *Fluid Mechanics* (Academic Press, New York, 2011).

¹² R. J. Goldstein, *Fluid Mechanics Measurements* (Taylor & Francis, New York, 1996).



