# Law-of-the-wall for small-scale streamwise turbulence intensity in high-Reynolds-number turbulent boundary layers

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(Dated: October 13, 2018)

## Abstract

Following the dimensional analysis approach carried out in previous studies, it is hypothesized that the small-scale fluctuations should only depend on the inner scales, analogous to the Prandtl's law-of-the-wall for the mean flow. This allows us to examine the high frequency regime of the streamwise energy spectra where a "law-of-the-wall" in spectra would hold. Observations in high-Reynolds-number turbulent boundary layer data indicate that a conservative estimate for the start of this law-of-the-wall is  $f^+ = 0.005$  (which corresponds to 200 viscous time-units) across a range of wall-normal positions and Reynolds numbers. This is sufficient to capture the energetic viscous-scaled motions such as the near-wall streaks, which has a time scale of approximately 100 viscous units. This spectral collapse is consistent with the observations in internal flows and external flows in other studies. Furthermore, the spectral collapse leads to a universal scaling (based on skin-friction velocity and kinematic viscosity) for the small-scale streamwise turbulence variance (consistent with the hypothesis) across the entire boundary layer. A logarithmic variation of this small-scale variance is observed farther away from the wall.

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#### I. BACKGROUND

A number of previous efforts have focussed on proposing scaling laws for turbulent energy spectra of turbulent wall-flows. Perry & Abell [15] was perhaps the first to develop scaling laws for streamwise energy spectra using dimensional analysis approach. This work has been extensively extended in various subsequent studies [16–18]. The primary focus of these efforts were on the overlap scaling of streamwise energy spectra where the energy content is inversely proportional to the wall-normal position. The choice of different velocity and length-scales for the dimensional analysis in these previous studies were underpinned by the attached-eddy hypothesis, a model of wall turbulence that provides critical insights. Very recently, Zamalloa et al.[23] used dimensional analysis and proposed "model-free" scaling relations for the turbulent-energy spectra in different regions (near-wall, log and outer wake regions) of turbulent wall-flows. Specifically, they demonstrated the presence of law-of-the-wall in the high wavenumber regime of energy spectra in pipe/channel flows using experimental and DNS data over a limited range of Reynolds numbers.

In this paper, we present a re-interpretation of the dimensional analysis work carried out by Perry & Abell [15] and Zamalloa et al. [23] as well as observations of law-of-the-wall in small-scale turbulence intensity and in the high frequency regime of streamwise energy spectra in high Reynolds number turbulent boundary layers. We highlight some subtle differences (compared to those previous studies) in the choice of velocity and length scales (used in dimensional analysis) and justify these choices based on physical reasoning and experimental observations.

#### II. LAW-OF-THE-WALL FOR STREAMWISE ENERGY SPECTRA

This section is a recap of the dimensional analysis performed by [15] and [23]. However, we highlight the choice of different scaling variables and the reasoning required for it. Let  $\phi_{11}(k_1, y)$  be the power spectral density of the streamwise velocity fluctuation for a longitudinal wavenumber  $k_1$  at location y away from the wall. Then, the integral over all  $k_1$  of this power spectral density is equal to the turbulent energy of the streamwise velocity component at that wall-normal location.

$$\int_0^\infty \phi_{11}(k_1, y) dk_1 = u^2(y) \tag{1}$$

Now,  $k_1\phi_{11}(k_1, y)$  is the turbulent energy contained at a given wavenumber  $k_1$  at a certain wall-normal location. Following [15] and [23] and their dimensional analysis approach, it is clear that we need a velocity scale to non-dimensionalise  $k_1\phi_{11}$ , and potentially two different length scales for the wavenumber and wall-normal position.

$$\frac{k_1\phi_{11}(k_1,y)}{\tilde{U}^2} = F(kL_1,y/L_2) \tag{2}$$

where,  $L_1$  is the relevant length scale for wavenumber and  $L_2$  is the relevant length-scale for wall-normal position.  $\tilde{U}$  is the relevant velocity scale for the energy. We have to to use physical arguments to arrive at appropriate choices for these scales.

Recall that Prandtl's law-of-the-wall for mean flow postulates that at high Reynolds numbers, close to the wall ( $y \ll \delta$ , where  $\delta$  is the outer length scale), there is an inner layer in which the mean velocity is determined by the viscous scales, independent of outer length and velocity scales. Based on this, we arrive at an equation for the mean-flow, which is the law-of-the-wall:

$$\frac{U}{U_{\tau}} = f_w \left( \frac{yU_{\tau}}{\nu} \right) \tag{3}$$

 $U_{\tau}$  is the skin-friction velocity ( $U_{\tau} = \sqrt{\tau_w/\rho}$ , where  $\tau_w$  is the wall-shear-stress and  $\rho$  is the density of the fluid) and  $\nu$  is the kinematic viscosity of the fluid.

We can follow the same reasoning as for the mean profile to determine the appropriate values for  $\tilde{U}$ ,  $L_1$  and  $L_2$ . As an observer at the wall with "no knowledge" of the outer flow, there is only one choice for  $\tilde{U}$ , which is the viscous velocity scale  $(U_{\tau})$ . There may be two possible candidates for  $L_1$ : viscous length scale  $(\nu/U_{\tau})$  and wall-normal position (y). For  $L_2$ , there is only one possible candidate, which is the viscous length scale  $\nu/U_{\tau}$  (note that we will represent scales non-dimensionalised with the viscous scales with a superscript '+'). Then, the spectral scaling becomes,

$$\frac{k_1\phi_{11}(k_1,y)}{U_\tau^2} = F(k_1L_1,y^+) \tag{4}$$

[15] used their insights from attached-eddy hypothesis and chose  $L_1 = y$  that highlights distance from the wall scaling. They used  $U_{\tau}$  to be the velocity scale. Here, we take a

different approach to determine  $L_1$ . We postulate that there exists a law-of-the-wall for the small-scale velocity fluctuations, i.e. the integral of the spectrum over a certain high wavenumber range. In the near-wall region, the variance of the small-scales over a range of high wavenumbers is only influenced by inner velocity scale such that,

$$\frac{\overline{u_S^2}}{U_\tau^2} = g_w(y^+) \tag{5}$$

where,  $u_S^2$  is the variance of the streamwise velocity fluctuations in the high wavenumber regime that should conform to law-of-the-wall and  $g_w$  is the function that represents law-of-the-wall for these small-scale motions. This will result in a universal form that is only dependent on viscous scales and is independent of outer influence. Therefore, we can choose  $L_1$  as  $\nu/U_\tau$ . For these high wavenumbers, [15] used the classical Kolmogorov scaling of  $\eta$  (length scale) and  $v_\eta$  (velocity scale) that depends on the dissipation of turbulent kinetic energy ( $\langle \epsilon \rangle$ ). However, given that the Kolmogorov scales changes with wall-normal direction, especially in the near-wall region, the inner-scales are a much more stringent requirement compared to Kolmogorov scales. Additionally, it ensures that the small-scale turbulence intensity has the same scaling parameters as the mean flow.

This above assumption is consistent recent observations of the small-scale behaviour in wall-bounded flows. Recently, Ref.[20] showed the universality in the small-scales across different flows even in low Reynolds numbers. This was especially true farther away from the wall. Ref.[2] showed that the kinetic energy dissipation (and the Kolmogrov scale) in the near-wall region of a turbulent channel flow scales with the inner scales. They show excellent collapse of  $\eta^+$  versus  $y^+$  with increasing Reynolds numbers. Dissipation is a weighted integral of the energy spectrum, where the weighting is towards the higher wavenumbers. Therefore, the collapse in small-scale turbulence intensity is similar to the collapse of integrated dissipation spectra. The dissipation based length scale (i.e. the Kolmgorov scale) provide a good scaling parameter for boundary-free flows, however, in the presence of the wall, skin-friction velocity  $(U_{\tau})$  captures the integrated effects of the large-scales and small-scales and therefore represents a better scaling parameter. In fact, Ref. [2] built on previous efforts[5, 22] and showed that the probability density functions of instantaneous Kolmogorov scale can be collapse with an outer-length scale that is derived based on Townsend's attached Eddy model. This was necessary to capture the effect of large-scale on the small-scales Based on

these arguments, we set  $L_1$  to be  $\nu/U_{\tau}$ . Therefore,

$$g_w(y^+) = \int_{M^+}^{\infty} F(k_1^+, y^+) d[\ln(k_1^+)]$$
 (6)

is a universal function in the inner region across Reynolds numbers. At fixed values of  $y^+$  close enough to the wall (i.e.  $y \ll \delta$ ), the above equation indicates that there should be a universal value of  $M^+$  that is independent of  $y^+$ .

Zamalloa et al. [23] use y as their length scale  $L_1$ , however, they recognised that this scaling can be replaced with a scaling similar to the one mentioned above. They did not examine this in any detail. They used data and observed that the spectra collapses for high wavenumbers, if,  $k_1y \geq 10(y^+/Re_{\tau})$ . This translates to  $k_1^+ \geq 10/Re_{\tau}$  or  $k_1\delta \geq 10$ , which essentially means that there should be one order of magnitude difference in the wavenumber range compared to the large-scales of the flow. These observations were limited in Reynolds number range (only up to  $Re_{\tau} \approx 3000$ ) and were confined to internal flows. This limit of collapse might have an outer influence since the scale separation is not sufficiently large. However, the choice of  $U_{\tau}$  as the velocity scale should ensure that there is collapse in the high wavenumber regime. Therefore, it is important to examine this collapse at higher Reynolds numbers to determine if there exists an appropriate wavenumber cut-off that eliminates the dependence of distance from the wall (or Reynolds number). This is examined in the next section.

#### III. OBSERVATIONS IN TURBULENT BOUNDARY LAYERS

It is very difficult to obtain wavenumber spectra at high Reynolds number without invoking Taylor's hypothesis where we have to assume or model a frequency-wavenumber mapping function. Typically, this mapping function is assumed to just depend on the local mean velocity. This could lead to incorrect observations on the nature of collapse of spectra. Therefore, here we resort to using frequency spectra from hot-wire data without invoking Taylor's hypothesis (or other equivalent mapping). In this case, law-of-the-wall for spectra becomes,

$$\frac{f\phi_{11}(f,y)}{U_{\tau}^2} = F(fT_1, y^+) \tag{7}$$

where, f is the frequency and  $T_1$  is a suitable time-scale. Following the same arguments as in previous section, the appropriate time-scale should be the inner time-scale based on wall-shear stress and viscosity,  $T_1 = \nu/U_\tau^2$ . This makes the law-of-the-wall for streamwise energy spectrum in frequency domain,

$$\frac{f\phi_{11}(f,y)}{U_{\tau}^2} = F(f^+, y^+) \tag{8}$$

We now seek collapse in spectra across Reynolds numbers such that,

$$g_w(y^+) = \frac{u_S^2(y^+)}{U_\tau^2} = \int_{A^+}^\infty F(f^+, y^+) d[\ln(f^+)]$$
(9)

is a universal function in the inner region. Here,  $A^+$  is equivalent to  $M^+$  where at fixed values of  $y^+$  close enough to the wall (i.e.  $y \ll \delta$ ), there is a universal value of  $A^+$ .

Hot-wire measurements obtained in Melbourne's High Reynolds Number Boundary Layer Wind Tunnel (HRNBLWT) is used to compare spectra at similar wall-normal locations over a range of Reynolds numbers ( $Re_{\tau}=2800,\ 3900,\ 7300,\ 14150$ ). The datasets for  $Re_{\tau}=2800,\ 3900$  and 7300 are obtained from [8] and [13]. The dataset for  $Re_{\tau}=14150$  is from [7]. Figure 1 shows the energy spectra against inner-normalised frequency over a range of wall-normal locations including  $y^+=11\pm1,\ 16\pm1,\ 53\pm3,\ 103\pm4,\ 207\pm8$  and  $525\pm22$ . The largest  $y^+$  value is chosen to be in outer-edge of the log region at the lowest Reynolds number. It is difficult to obtain data at exactly the same  $y^+$  values across different Reynolds numbers. Therefore, the locations mentioned are average of the nearest wall-normal location over the range of Reynolds numbers and the error is based on twice the standard deviation of the wall-normal positions that are chosen across Reynolds numbers.

Spectra in figure 1 shows excellent collapse in the spectra at high frequencies where we expect the law-of-the-wall to hold. It should be noted the data at the highest Reynolds number might suffer from mild attenuation of energy due to spatial resolution of the hot-wire probe, especially, at these high frequencies [8]. Despite this, the collapse is within the statistical error. The collapse is obvious even at  $y^+ \approx 103$  down to a frequency of  $f^+ = 0.001$  (which is shown in dashed line in the figure). Below this frequency, the outer-influence becomes obvious and the spectrum for each Reynolds number peel off at different frequencies. The quality of the collapse is worse at  $y^+ = 525$ . At that location, wall-normal location is at the outer-edge of the log region at  $Re_{\tau} = 2800$ , while it is still in the log

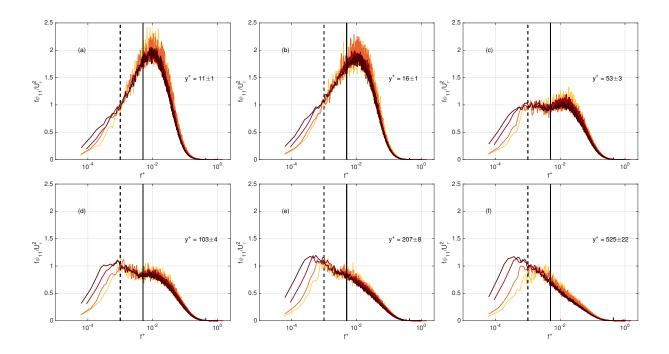


FIG. 1. (a) - (f) Inner-normalised energy spectra for 4 different Reynolds numbers at 6 different values of  $y^+$ . The figures show the value of  $y^+$  and the variance in that selected wall-normal location across the Reynolds numbers. Lines are coloured from lightest to darkest in order of increasing  $Re_{\tau}$  (= 2800, 3900, 7300 and 14150). The solid black line shows the  $f^+$  = 0.005, which is 200 inner time-units while the dashed black line shows  $f^+$  = 0.001, which is 1000 inner time-units.

region for the other Reynolds numbers. We expect the law-of-the-wall for the small-scales of the spectrum to diminish at these locations where the shear is minimal and therefore the spectra at the small-scales would collapse with Kolmogorov scales. Regardless, in the near-wall region (where  $y^+$  is small and  $y \ll \delta$ ), there is excellent collapse in the spectra at least up to  $f^+ = 0.005$  (shown as solid black line) and perhaps up to  $f^+ = 0.001$  (dashed line).

Based on the above observations, a conservative estimate for  $A^+$  in equation 9 can be 0.005. We can now integrate the power spectral density from this value of  $A^+$  up to infinity. This should give the near-wall small-scale turbulent energy, which obeys law-of-the-wall. The amount of energy contained in these scales will diminish farther away from the wall. In fact, farther away from the wall, the viscous scale may no longer be equal to inner-scales but could be replaced with Kolmogorov scales as in [15]. However, farther away from the

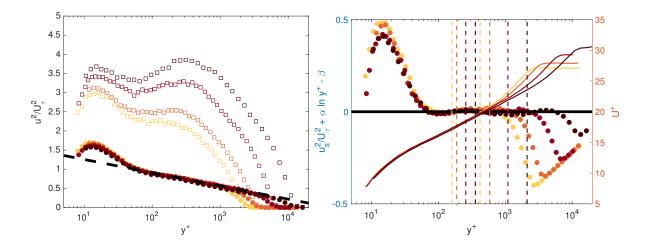


FIG. 2. (Left) Inner normalised large-scale (open squares) and small-scale (filled circles) variance of streamwise velocity for 4 different Reynolds numbers. The separation between large- and small-scale is based on a Fourier filter at  $f^+ = 0.005$ . The dashed black line shows the log decay of the small-scale variance beyond  $y^+ > 100$ . (Right) Compensated form of small-scale variance where the fitted log trend is subtracted from the data (symbols) and the log region in the mean flow (solid lines). The dashed lines show the extent of log region  $(3\sqrt{Re_{\tau}} \leq y^+ \leq 0.15Re_{\tau})$ . Increasing darkness in all symbols and lines represents increasing Reynolds number.

wall, it is difficult with current experimental data to delineate if this frequency should scale with viscous wall-units or Kolmogorov scales. From balancing production and dissipation in the logarithmic region, [18] found that the inner-normalised Kolmogorov time scale,  $t_{\eta}^{+}$ , should vary as  $t_{\eta}^{+} \approx \sqrt{(\kappa y^{+})}$  (where  $\kappa$  is the Karman constant). Therefore, the value  $A^{+}$  in terms of Kolmogorov scales will vary with wall-normal location as  $A_{\eta} \approx 0.005\sqrt{\kappa y^{+}}$ . For the higher Reynolds number considered here, this would translate to  $A_{\eta} \approx 0.06$  (at the lower end of the log region) and  $A_{\eta} \approx 0.14$  (at the upper end of the log region). It is difficult to discern collapse (or lack thereof) with the current data and does not preclude the presence of Kolmogorov scaling. However, it does suggest that inner-scaling provides a bound for the low frequency limit where there is indeed collapse of the small scales (that is consistent with the Kolmogorov scaling).

The impact of using  $A^+ = 0.005$  as the cut-off frequency is explored by computing the large-scale and small-scale variance of streamwise velocity based on this cut-off value. Figure 2 shows these two quantities in inner-scaling. The small-scale fluctuations across Reynolds

numbers collapse across the entire range of wall-normal locations (up to  $y^+ \approx 1000$ , which is well outside the log region for the lower Reynolds number). In fact, the small-scale variance appears to follow a logarithmic decay with wall-normal position for  $y^+ > 3\sqrt{Re_{\tau}}$  across all Reynolds numbers. This logarithmic decay extends well in to the outer region (beyond the traditional log region for the mean profile). This log decay has the form,

$$\frac{u_S^2(y^+)}{U_\tau^2} = -\alpha \ln(y^+) + \beta \text{ for } y^+ > 3\sqrt{Re_\tau}$$
 (10)

where,  $\alpha = -0.15 \pm 0.002$  and  $\beta = 1.59 \pm 0.01$  (the uncertainty estimates are based on the rms of the difference between the experimental data and equation 10 across all Reynolds numbers within the log region as identified in the mean flow).

This log trend is further clarified in figure 2(right) where the small-scale turbulence is plotted in compensated form (i.e. the log trend determined above was subtracted from the data with the given constants). This compensated form is similar to the indicator function (where the gradients are plotted), however, this compensated form is preferred here to avoid numerical differentiation of coarsely sampled (in y direction) experimental data. The figure clearly exhibits a residue of nearly zero over a wide range of wall-normal locations, which is a reflection of the presence of the log region of the above form. The figure also shows the logarithmic region exhibited by the mean flow and dashed lines show a conservative estimate of the log region (at each Reynolds number) in the mean flow following [12],  $3\sqrt{Re_{\tau}} \leq y^+ \leq 0.15Re_{\tau}$ . It is clear that the log decay in small-scale turbulence overlaps with the log region in the mean flow. In fact, the figure seems to indicate that the log region for small-scale turbulence is over a wider range compared to the mean flow.

The value of these constants are expected to depend on the choice of  $A^+$ . The values reported here are for  $A^+ = 0.005$ . The value of  $\alpha$  does not appear to change (for say  $A^+ = 0.001$ , however, the intercept does depend on the amount of energy retained in the filtering process. Regardless of the intercept, the above finding indicates that the rate of change of small-scale fluctuations with wall-normal location (i.e. gradient of wall-normal fluctuations with wall-normal location) is inversely proportional to the distance from the wall (far enough away from the wall).

The large-scale variance increases with increasing Reynolds number. This seems to confirm that the value of  $A^+$  should indeed be independent of wall-normal location (at least in the near-wall and logarithmic region). It should be noted that this value of  $A^+$  is a

conservative bound on the frequency. In time-scale, this corresponds to 200 wall-units. The physical significance of 200 wall-units as the cut-off time scale is obvious. The near-wall streaks in boundary layers/pipes/channels all last approximately 100 wall-based time-units (and is equivalent to 1000 wall-units length scale) and their features are very robust regardless of the type of flow (see reviews [11, 14, 21] and references therein). Therefore, the cut-off of 200 wall-units essentially captures the energy contained in these near-wall streaks that self-sustain themselves given a certain mean wall-shear-stress.

The quoted value of  $A^+$  should be considered as frequency bound above which spectral law-of-the-wall is expected to hold. It is possible that at increasing Reynolds numbers, spectral collapse is observed up to lower values of  $f^+$  (i.e. lower frequencies or equivalently wavenumbers - which is indeed the case in figure 1) due to increasing scale separation between inner and outer scales. In fact, there is also evidence that a cut-off wavelength of 7000 to 10000 wall-units results in the collapse of small-scale turbulence statistics in the near-wall region (see [10, 21]). This length-scale is derived from frequency spectra in liaison with Taylor's local hypothesis. Ref. [13] used a cut-off wavelength of 7000 wall-units, which approximately corresponds to  $f^+ \approx 0.001$ , to separate the inner and outer scales to measure amplitude modulation. Moreover, recently, Ref.[1] and Ref.[9] showed that the spectral coherence between two probes (one located in the near-wall region and another in the outer region) at high Reynolds numbers reduced to zero at around 7000 wall-units, indicating a value of  $A^+ = 0.005$  would also not have any coherence. Very recently, [19] performed measurements in the Melbourne facility using the NSTAP probes (where the measurements are fully resolved) and showed that the spectra indeed collapses with inner scaling across a range of wall-normal positions. In fact, they claim that the collapse is up to wavelengths of 10000 wall-units (that corresponds to approximately 1000 viscous time units) and most certainly up to wavelength of 2000 wall-units (200 time-units). Therefore, the proposed time-scale of 200 wall-units is a very conservative estimate for the cut-off frequency above which spectral similarity should hold.

The influence of outer region on the collapse of near-wall high-frequency spectra can be further examined by using data from experiments where the outer influence is artificially enhanced by the introduction of free-stream turbulence. Figure 3 shows spectra obtained at  $y^+ = 15$  in an experiment that was designed to increase the strength of outer influence [3, 4, 6]. The figure shows the premultiplied power-spectral density (normalised by  $U_{\tau}$ ) versus

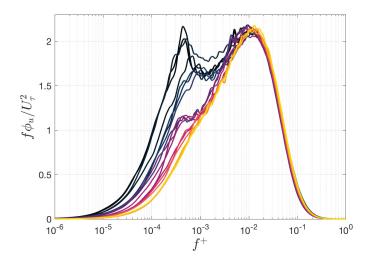


FIG. 3. Inner-normalised pre-multiplied spectra at  $y^+ \approx 15$  for different levels of freestream turbulence (FST). Lines are coloured from lightest to darkest in order of increasing turbulence intensity (the range is from 7% to 13%). The Taylor microscale of FST ranges from 300 to 760. The abscissa shows inner-normalised frequency  $f^+$ . Further details on the different turbulence intensities, integral length scales and boundary layer characteristics can be found in [6].

inner normalised frequency. In these experiments, the outer influence was exacerbated by using freestream turbulence and this turbulence intensity would penetrate the boundary layer and alter the mean skin-friction characteristics as well as spectral properties [3]. However, when this altered skin-friction is taken in to account, the collapse of the spectra at  $y^+ = 15$  over a wide range of external forcing is really clear (freestream turbulence levels up to 13%). It can be seen that for low freestream turbulence intensity, the collapse is excellent till  $f^+ = 0.001$ . However, as the FST levels increases, the energetic outer scales penetrate down to the wall and the collapse is worse. Yet, the spectra appear to collapse up to  $f^+ = 0.005$  regardless of the intensity of forcing in the outer layer.

### IV. CONCLUSIONS

The dimensional analysis approach of [15] and [23] to obtain spectral scaling is reinterpreted in the context of experimental observations in high Reynolds number turbulent boundary layers. A scaling relation for the high frequency regime of streamwise energy spectra (in frequency domain) in the near-wall region is proposed based on the hypothesis of law-of-the-wall for the small-scale turbulence intensity. It is noted that the inner scale and Kolmogorov scales are related to each other, yet, there are subtle important differences. The inner viscous scale based on  $U_{\tau}$  and  $\nu$  captures the effects of both large- and small-scales (in the integrated sense). The Kolmogorov scale  $(\eta)$  depends on mean turbulent-kinetic-energy dissipation  $(\langle \epsilon \rangle)$  and  $\nu$  where  $\langle \epsilon \rangle$  captures the effect of small- and large-scales (depending on how much dissipation resides in the low-wavenumbers or low-frequencies). The value of  $\langle \epsilon \rangle$  changes with wall-normal location while  $U_{\tau}$  does not. Therefore, the value of  $\eta$  and  $v_{\eta}$  changes with wall-normal location while the inner-scales remain fixed. Therefore, the choice of  $U_{\tau}$  as the relevant velocity scale and  $\nu/U_{\tau}$  as the length scale will situate the mean flow and the small-scale turbulence intensity in the same framework. This does not negate Kolmogorov scaling as other studies have found that the Kolmogorov scales ( $\eta$  and  $v_{\eta}$ ) exhibit universal behaviour with inner scaling across different Reynolds numbers.

We observe that when the energy spectra is scaled in inner-units, there is a collapse beyond a certain frequency. This value appears to be a universal inner-scaled value across Reynolds numbers. If this was scaled with  $\eta$ , then, this value would not be the same for all wall-normal locations. Moreover, the inner-scaled value has a physical significance in that the length-scale corresponding to this value is related to the size of the near-wall streaks. The observations here are consistent with the findings related to spectra and Kolmogorov scales in internal and external flows[2, 19, 23]. In frequency domain, the spectra at similar wall-normal positions are found to collapse down to  $f^+ \approx 0.005$  across a whole range of Reynolds numbers. The corresponding small-small turbulence intensity also appears to collapse across all wall-normal positions, confirming the law-of-the-wall hypothesis. The profile follows a logarithmic decay with wall-normal position farther from the wall, which is consistent with Kolmogorov scalings used by [18]. These observations were further confirmed using experimental data where the boundary layer is under the influence of free-stream turbulence and the collapse of the spectra up to the near-wall streaks appears to be robust.

#### ACKNOWLEDGMENTS

This study was a re-analysis of existing data and we are grateful to Prof. Marusic and Prof. Hutchins at the University of Melbourne for making the hot-wire data from HRNBLWT available. We thank Dr. Jason Hearst for providing Figure 3. We are thankful to

Prof. Marusic and the anonymous referees for their comments on the manuscript. Financial support of the European Research Council (ERC Grant agreement No. 277472), and the Engineering and Physical Sciences Research Council of the United Kingdom (EPSRC Grant Ref. No. EP/L006383/1) is gratefully acknowledged.

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