Numerical study of mobilized friction along embedded catenary mooring chains

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ABSTRACT

Understanding the soil resistance along an embedded anchor chain is imperative for efficient and economic design of an overall mooring system as it determines the magnitude and direction of the load at the padeye of the anchor. The tensioning process of an embedded chain for catenary moorings was modeled using a coupled Eulerian–Lagrangian (CEL) finite element approach simulating the large deformations of the chain as it cuts through the soil to form an inverse catenary. The analyses reveal that the configuration of the embedded chain and the relationship between tension and chain angle at the padeye show excellent agreement with previously published analytical predictions. However, the ratio of the tension at the padeye to that at the mudline obtained from CEL is significantly higher than the theoretical values, mainly due to partial mobilization of the frictional soil resistance along the length of the chain. The CEL results indicate that the partial mobilization is a result of the combined-loading effect during failure of the soil around the embedded chain as it cuts through the seabed, in contrast with the conventional assumption that the ultimate frictional and normal soil resistances are mobilized simultaneously. A new design approach is proposed for calculating the local equivalent coefficient of friction based on the yield locus for a deeply embedded chain and the normality rule.

Keywords: Anchor chain; catenary mooring; chain-soil interaction; coupled Eulerian–Lagrangian; finite element analysis

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79   **INTRODUCTION**

80   Buoyant offshore structures, which provide a variety of functions across the offshore sector,

81   require a mooring system and anchor to keep station. Offshore anchors vary in configuration

82   and operational mode but all generally have a mooring line attachment point, or padeye, below

83   the mudline, with a section of the chain embedded in the soil (Randolph and Gourvenec 2011).

84   For a catenary mooring line applying a horizontal load at the seabed, the embedded chain will

85   form an 'inverse catenary' in the soil to maintain equilibrium between the chain tension and the

86   soil resistance. Quantifying the normal and frictional soil resistances to the chain is required to

87   calculate the configuration and tension profile of an embedded chain, enabling efficient anchor

88   design according to the magnitude and inclination of the load at the padeye.

89   **Analytical solutions**

90   *System behavior:*

91   Solutions for the behavior of the whole length of an embedded chain in undrained soil

92   conditions were developed by discretizing the chain into a number of arc-shaped/curvilinear

93   elements in mechanical equilibrium with transmitted tension and soil resistance and then

94   incrementally integrating the equilibrium equations from one end of the chain to the other. Early

95   work by Reese (1973) discretized a weightless embedded chain as a succession of arcs of circles

96   with varying radii, without considering the effect of frictional soil resistance on axial chain

97   movement. The solution was improved by adding the effects of chain weight and frictional soil

98   resistance, leading to a significant reduction of the chain tension at the anchor (Gault and Cox

99   1974). Vivatrat et al. (1982) considered an embedded chain as a series of curvilinear segments

100   establishing the pair of differential equations governing respectively the variation of chain

101   tension and geometric configuration as

102
$$\frac{dT}{ds} = F + w_c \sin\theta \tag{1}$$

103
$$T\frac{d\theta}{ds} = -Q + w_c \cos\theta \tag{2}$$

104   where T is the tension in the chain acting over the incremental chain length ds, θ is the angle

105   subtended by the chain to the horizontal, w_c is the submerged weight of chain per unit length,

106   and Q and F are respectively the normal and frictional soil resistances per unit length. The chain

107   configuration and equilibrium of an element is illustrated in Figure 1, with boundary conditions

108   of T_a and θ_a at the padeye at depth z_a, leading to tension T_m and θ_m (taken as zero here) at

109   mudline. In general, since for most moorings T_m and θ_m are specified, iteration is needed in

110   order to establish compatible values of T_a and θ_a. The analysis basis for embedded chains

111   proposed by Vivatrat et al. (1982) has since been adopted in many other studies (e.g.

112   Degenkamp and Dutta 1989; Dutta and Degenkamp 1989; Neubecker and Randolph 1995a;

113   Neubecker and Randolph 1995b; Neubecker and Randolph 1996).

114   The differential equations (Eqs. (1) and (2)) were solved in closed-form by neglecting the

115   weight of the chain, resulting in the tension profile (Neubecker and Randolph 1995b)

116
$$\frac{T}{T_a} = e^{\mu(\theta_a - \theta)} \tag{3}$$

117   with the chain friction coefficient μ defined as the ratio F/Q.

118   The Neubecker & Randolph solution also established the variation of the chain angle as

119
$$\frac{T_a}{1+\mu^2}\left[e^{\mu(\theta_a - \theta)}(\cos\theta + \mu\sin\theta) \right]_\theta^{\theta_a} = \int_z^{z_a} Q dz \tag{4}$$

120   which may be simplified for small-angles as

121
$$\frac{T_a}{2}(\theta_a^2 - \theta^2) = \int_z^{z_a} Q dz \qquad (5)$$

122   In all the above solutions, the soil resistances Q and F, normal and parallel to the chain axis,

123   have been assumed uncoupled and fully mobilized simultaneously to the corresponding

124   ultimate capacities Q_u and F_u.

125   ***Chain-soil forces:***

126   Chains comprise multiple separate links with alternating orientations to the chain axis (Figure

127   2a) and chains of bar diameter d_b having individual link geometries that are typically 5-6d_b long

128   and ~3.5d_b wide. Consequently, in order to calculate soil resistances for this complex geometry,

129   approximations about the geometry have been made by introducing an effective width of $E_n d_b$

130   for the normal resistance and effective perimeter $E_t d_b$ for frictional resistance (Reese 1973).

131   Under undrained conditions in soil with shear strength s_u, the ultimate normal resistance Q_u and

132   frictional resistance F_u may therefore be estimated (Degenkamp and Dutta 1989) as

133
$$Q_u = E_n d_b N_c s_u \qquad (6)$$

134
$$F_u = E_t d_b s_u \qquad (7)$$

135   where N_c is the bearing capacity factor. The apparent coefficient of friction μ may be deduced

136   from the effective width and the bearing capacity factors as $F_u/Q_u = E_t/(E_n N_c)$.

137   Proposed values of the effective width parameters, E_n and E_t, and the bearing capacity factor

138   N_c varies among researchers. The effective width of chain in bearing was first introduced by

139   Reese (1973), with an upper bound value of $E_n = 3$ proposed by assuming that no flow of soil

140   through the opening in the chain was permitted. Vivatrat et al. (1982) adopted the bearing area

141   per unit length as the average of the maximum and minimum values, considering different

142   orientations of the chain links with respect to the soil, and the tangential area as the surface area

143   of the chain link, proposing effective width parameters $E_n = 2.6$ and $E_t = 10$ for calculating the

144   normal and frictional resistances per unit length, respectively. For the bearing capacity factor

145   N_c, a value equal to the ultimate lateral bearing capacity factor at depth for a pile, ranging from

146   9 to 11, was proposed initially (Reese 1973; Vivatrat et al. 1982). Yen and Tofani (1984)

147   suggested that $E_n = 2.37$ and $E_t = 5.7 \sim 8.9$ for stud-link chains based on a series of model tests,

148   back-calculating values of N_c between 7.1 and 12.1. Degenkamp and Dutta (1989) quantified

149   average values of E_n and E_t as 2.5 and 8.0 by investigating $6d_b$ stud-link and $5d_b$ studless chains

150   with both orthogonal and diagonal configurations. The bearing factor N_c was assumed

151   equivalent to that of a strip foundation, increasing with depth from 5.14 at mudline to a

152   maximum of 7.6 (Degenkamp and Dutta 1989; Dutta and Degenkamp 1989; Neubecker and

153   Randolph 1995a; Neubecker and Randolph 1995b; Neubecker and Randolph 1996). These

154   result in values of μ between 0.4 and 0.6.

155   **Experimental observations and numerical simulations of embedded anchor chains**

156   Frankenmolen et al. (2016) reported centrifuge tests of chain-soil interaction for surface and

157   embedded chains in carbonate sand. The chain configurations for the embedded chain under

158   monotonic tensioning showed agreement with the analytical solutions set out in Eq. (2), except

159   for cases where the chain formed a padeye angle greater than 70°. However, the back-calculated

160   'operative' coefficient of friction (i.e. derived from tensions and angles at the anchor and

161   mudline using Eq. (3)) were significantly lower than obtained from the chain surface dragging

162   tests. Partial mobilization of the frictional resistance along the embedded chain was considered

163   one of the primary reasons.

164   Rocha et al. (2016) presented results of a series of chain tensioning laboratory tests in very soft

165   clay (water contents of 100 to 120%), including cycling of the mudline load between zero and

166   a fixed maximum value at various mudline chain angles. At each maximum load level the

167   deduced coefficient of friction consistent with Eq. (3), lay in the approximate range 0.2 to 0.3.

168   Han and Liu (2017) proposed a global yield locus for an embedded chain on the basis of data

169   from model tests on tensioning of an Omni-Max anchor in soft clay ($s_u \sim 1.2$ to 1.4 kPa).

170   Mooring lines of both chain and rope were investigated and measured tensions at mudline and

171   anchor padeye were used to evaluate equivalent friction coefficients on the basis of Eq. (3).

172   Again, they showed that the deduced values of μ varied with progress of the test, decreasing

173   from a high value initially to steady values in the range 0.2 to 0.3.

174   The experimental data discussed above challenge the conventional assumption in analytical

175   methods that bearing and friction are mobilized simultaneously along the entire length of the

176   chain. Explicit consideration of the bearing and shear components indicates that combined-

177   loading effects may lead to reduced friction of the chain, resulting in operative friction

178   coefficients that are significantly lower than might be estimated otherwise, for example with

179   values from model tests in clay that are about 50% of the range suggested by Degenkamp and

180   Dutta (1989).

181   Numerical investigations of the behavior of an embedded chain have been reported using the

182   Coupled Eulerian-Lagrangian (CEL) method. Pre-tensioning of an anchor chain for the

183   installation of a catenary mooring line has been examined (e.g. Kim et al. 2015; Zhao and Liu

184   2013; Zhao and Liu 2014; Zhao and Liu 2016), and the chain configuration and tension profiles

185   of the embedded chain compared with existing analytical predictions. Although discrepancies

186   between the results and the analytical tension profile of an embedded chain were noted, these

187   were not interpreted systematically, inspiring the current investigation of the chain-soil

188   interaction using CEL.

189   **Scope of present study**

190   This paper presents results of Coupled Eulerian-Lagrangian finite element analysis of the

191   undrained response of an embedded chain in the orthogonal space of normal and frictional

192   resistances. The tensioning process of the chain is modelled with a fixed anchor point ('padeye')

193   and a horizontally loaded chain at the seabed. The chain is tensioned from the vertical 'installed'

194   position to an equilibrium state in an inverse catenary, with gradually decreasing padeye angle

195   as the tension is increased. Geometric configurations and tension profiles from the numerical

196   analyses are compared with analytical predictions using the (Neubecker and Randolph 1995b)

197   approach. Investigations of the soil resistances mobilized along the chain are presented in a

198   yield-locus context. A method for calculating the local equivalent coefficient of friction of the

199   embedded chain is proposed with the chain configuration established using the Neubecker &

200   Randolph solution and the yield locus derived from separate analyses for a deeply embedded

201   chain. This is then extended to derive a formulation for estimating the global operative friction

202   factor for the embedded chain system, taking account of the variation in mobilized friction

203   along the chain length.

204   **FINITE-ELEMENT MODEL**

205   *Modelling software and approach*

206   All finite element analyses were carried out with the coupled Eulerian-Lagrangian (CEL)

207   approach provided in the software Abaqus (Dassault Systèmes 2014). The basis of the CEL

208   approach is that the element nodes move temporarily with the material during a Lagrangian

209   calculation phase, which is followed by mapping to a spatially 'fixed' Eulerian mesh. This

210   enables efficient simulation of large deformation problems, in this case the cutting of a mooring

211   chain through a soil mass from an initially vertically position to form an inverse catenary.

212   *Chain geometries*

213   The chain was modelled as a line of equally spaced, rigid cylindrical segments, connected using

214   'LINK' connector elements provided as standard in Abaqus (Figure 2a). The LINK connectors

215   transmit axial force but no bending moment, and maintain a fixed distance between the two

216   connected nodes.

217   The length l_s and the diameter d_s of each segment, and the space between adjacent segments, s_s

218   are the three geometric variables governing the dimensions of the model chain. In the current

219   analyses, l_s was taken as 1 m, with s_s and d_s being 0.1 m and 0.25 m respectively.

220   The geometries of chain in the CEL model were selected to model a standard $5d_b$ studless chain

221   as shown in Figure 2a. Each link pair of the $5d_b$ studless chain was represented by one bar

222   segment together with one spacing, i.e. the sum of l_s and s_s was equivalent to the length of one

223   link pair ($6d_b$). For this model, $l_s + s_s = 6d_b = 1.1$ m, giving $d_b \sim 0.18$ m. This provides a way to

224   relate d_s with d_b (i.e. $d_s/d_b = 0.25/0.18$ for this model) for the interpretation of the resistances

225   obtained from the CEL modelling. A detailed calibration exercise is presented later in order to

226   evaluate the numerical limiting normal and frictional resistances of the simulated chain,

227   together with a yield locus.

228   The embedded chain model comprised 49 segments (giving a total length of 53.9 m) as shown

229   in Figure 2b. The padeye was set (arbitrarily) to 9 m below the soil surface, providing sufficient

230   embedment to investigate the chain tensioning in a realistic manner without introducing

231   unnecessary length (and therefore computational expense). Two chain weights of $w_c = 0$

232   (weightless chain) and 1.5 kN/m were considered and the chain was loaded horizontally at the

233   seabed (i.e. as a catenary mooring).

234   ***Soil domain***

235   The soil was assumed to be fully undrained during the chain tensioning process and

236   consequently total stress analyses were performed. The undrained soil strength was assumed to

237   increase linearly with depth z, according to $s_u = 2 + 1.2z$ kPa, with the effective soil weight

238   being 2.4 kN/m^3. These values are typical of deepwater seabed sediments offshore West Africa

239   (Ehlers et al. 2005), which have been the focus of significant attention recently following

240   observations of anchor line trenching (Bhattacharjee et al. 2014; Colliat et al. 2018). A linear

241   elastic-perfectly plastic constitutive model, obeying the Tresca failure criterion, was adopted

242   for the soil. The Young's modulus E was taken as $500s_u$ and Poisson's ratio as 0.49 to

243   approximate incompressible soil response for undrained conditions. The regular mesh

244   comprised 8-noded linear brick elements with reduced integration (EC3D8R in the Abaqus

245   element library). A central fine mesh zone (minimum mesh size $L_e = 0.35d_s$) was used to

246   discretize the soil-flow area where significant soil distortion was expected due to the changing

247   chain configuration during tensioning.

248   ***CEL model setup***

249   Figure 2b presents the global view of the CEL model (top) with the detailed chain model

250   illustrated at the bottom of the figure. Only half of the anchor chain (with a semi-circular cross-

251   section) and soil domain was analyzed because of symmetry with respect to the vertical plane

252   through the chain axis. As shown in Figure 2b, the left boundary and base of the soil domain

253   extended $7.5l_s$ (7.5 m) from the padeye. The length of the whole soil domain was taken as 67.6

254   m, and the out-of-plane thickness as $30d_s$ (7.5 m) to avoid boundary effects. Above the soil

255   surface, a layer of 'void' with height $1l_s$ (1 m) was prescribed, sufficient to accommodate soil

256   heave in front of the chain during tensioning.

257   The anchor chain was initially 'wished-in-place' in the post-installation configuration with a

258   vertical section fully embedded and attached to the padeye, and a horizontal section laid on the

259   seabed with the centerline of the chain level with the seabed surface. Subsequently, the motion

260   of the dragging point was activated by applying a constant horizontal velocity v_d, forcing the

261   chain to cut gradually through the soil. The dragging point was displaced horizontally until the

262   final padeye inclination angle θ_a reached 35° to the horizontal with the dragging point located

263   sufficiently far away from the padeye to ensure that the chain remained horizontal at the

264   dragging point. The response of two segments (#13 and #25, highlighted and numbered on

265   Figure 2b) was examined in post-processing to understand the chain behavior in detail.

266   The local coefficient of friction of the chain-soil interface was taken as 0.5 times the normal

267   force using the 'penalty' contact option in the Abaqus library. A quasi-static solution for the

268   tensioning process of the chain was achieved through a parametric sensitivity study of the

269   combination of mesh density, loading velocity and coefficient of friction of the interface.

270   Justification for the selection of a linear computational loading velocity (of $v_d = 4.0e\text{-}5L_e =$

271    $1.4e\text{-}5d_s$ per increment, equivalent to a velocity of 0.05 m/s) is shown below.

272   ***Validation of the CEL model***

273   Prior to analyzing the embedded chain for a catenary mooring, a straight seven-segment chain

274   was embedded horizontally at a depth of $18d_s$ (4.5 m) and then displaced in normal and axial

275   directions. Comparisons of the normal and frictional soil resistances with those estimated using

276   Eqs. (6) and (7) allowed verification of the CEL modelling, including the soil properties, chain

277   size, interface properties, boundaries, mesh refinement and simulation velocities, consistent

278   with those adopted in the global embedded chain model. Resultant loads on the outer two

279   segments were excluded in the interpretation, avoiding end effects from the two ends of the

280   seven-segment section.

281   A series of analyses were conducted using different input velocities (8.0×10^{-6}, 4.0×10^{-5}, 2.0

282    $\times 10^{-4}$ and 1.0×10^{-3} L_e per time increment) were conducted to quantify how the ultimate normal

283   and frictional capacities per unit length changed with velocity due to numerical errors and

284   therefore help select the simulation velocity. The normalized normal and frictional resistances

285    $N_b = F_{N,\,sum}/Ld_bs_u$ and $N_s = F_{S,\,sum}/Ld_bs_u$ are shown in Figure 3, where $F_{N,\,sum}$ and $F_{S,\,sum}$ are

286   respectively the sums of the normal and frictional soil resistance forces on the five central

287   segments of the chain, while L corresponds to the total length of the central five links, i.e. $5(l_s$

288    $+ s_s)$ and s_u is the undrained strength at the depth of the chain element centroid.

289   Pursuing a completely convergent solution would require use of an extremely small velocity,

290   as observed in CEL analysis of cone penetration by Wang et al. (2015), which would preclude

291   its use in the large-domain full chain system analyses. However, the CEL results of normal and

292   frictional capacities were comparable with the estimated values derived from Eqs. (6) and (7)

293   using the values of $N_c = 7.6$, $E_n = 2.5$ and $E_t = 8$ values recommended by Degenkamp and Dutta

294   (1989). Closest agreement was obtained for the loading velocity of $4.0 \times 10^{-5} L_e$ per increment

295   and so that simulation velocity was selected for the full-chain simulations (Figure 3).

296   Overall, the adopted simulation velocity resulted in an ultimate normal resistance value of

297    $N_{b, ult} = Q_{u, deep}/d_b s_u = 19$ (equal to the Degenkamp and Dutta (1989) recommendation), and

298   ultimate frictional resistance $N_{s, ult} = F_{u, deep}/d_b s_u = 6.5$ (compared to 8 from Degenkamp and

299   Dutta (1989)). This results in an acceptable equivalent 'friction coefficient' μ_p $(= F_{u, deep}/Q_{u, deep}$

300    $= 0.34)$, towards the lower end of the range of $0.4 – 0.6$ discussed earlier.

301   The adopted simulation velocity is about the lowest practical, given that analysis of the full

302   chain system still required about 350 hours on a fast server. For internal consistency, all later

303   simulations were performed applying this velocity at the chain end. While there will still be

304   some velocity variation at different locations along the chain, the variation of the net friction

305   coefficient is relatively minor and will not affect the validity of the conclusions reached.

306   **RESULTS**

307   *Yield loci for embedded chain segments*

308   The normal and tangential motion of the embedded chain induces failure of the soil as it cuts

309   through. Consequently, before analyzing the complete chain system, a suite of numerical

310   analyses were performed on the seven-segment section of chain (weightless and $w_c = 1.5$ kN/m)

311   for a deeply embedded (centroid depth = $18d_s$ (4.5 m, $s_u = 7.4$ kPa)) and a surface-laid chain

312   condition (centroid depth = 0 m ($s_u = 2.0$ kPa)). These analyses were designed to examine how

313   the normal and frictional soil resistances interact and to develop yield loci.

314   The resulting interaction diagrams are shown in terms of normalized normal and frictional

315   resistances N_b and N_s in Figure 4 for the two different embedment conditions. To probe the

316   yield locus for the deep condition, displacement-controlled probes (each at a different

317   displacement angle ω to the normal of the chain axis) were conducted (Figure 4a). A yield locus

318   was then selected, based on the final load path sections (when the load path reaches the failure

319   envelope and turns tangential to it) and the final points of the load paths when loads had

320   stabilized with further displacement. The yield locus took the form of a generalized ellipse:

$$\left(\frac{N_b}{N_{b,ult}} \right)^m + \left(\frac{N_s}{N_{s,ult}} \right)^n = 1 \tag{8}$$

321

322   where $N_{b,\,ult}$ and $N_{s,\,ult}$ are taken as 19 and 6.5 for pure penetration and sliding (see Figure 4a)

323   and m and n are fitting parameters. The exponents m and n of the yield locus function were

324   selected as 2.0 and 2.3, which are close to those proposed by Han and Liu (2017). Furthermore,

325   separate analyses revealed that the shape factors m and n remain consistent for different

326   gradients of the soil shear strength. Also from Figure 4a, normality is shown to apply for the

327   yield locus of the deeply embedded section, with the plastic displacement ratio of the chain

328   consistent with the gradient of the yield locus. In this way, the resistance ratio F/Q during failure

329   of the soil around the chain is related to the kinematics of the chain.

330   Although the above envelope does not fit the numerical data perfectly (for example, it cannot

331   capture discontinuous changes in mechanisms around the envelope), it is simple and easily

332   implementable in design. Given that each point along the chain is at a different point on the

333 envelope, this is likely to mean that minor disparities are insignificant, and consequently, we

334 consider that the approach is sufficiently accurate for design purposes.

335 For the surficial chain segments, it was found difficult to probe the yield locus with a

336 displacement-controlled scheme. Instead, the *Q-F* yield locus was deduced using load-

337 controlled analyses. The load paths are shown in Figure 4b with the final point on each line

338 representing 'failure' (i.e. a point where increasing displacement does not change the mobilized

339 resistance). The resulting envelope is a chili-leaf-shaped locus, with no frictional resistance

340 when the normal force is zero because the chain is weightless. The shape of the locus near the

341 original reflects the contact with a frictional coefficient of 0.5 adopted in the CEL model for

342 the chain-soil contact interface.

343 A comparison between the yield loci for $w_c = 0$ and 1.5 kN/m is presented in Figure 5. The

344 difference between the normal capacities of the weightless and weighty chains is simply $w_c/d_b s_u$.

345 Both yield loci show greater interaction effects than those for a suction caisson (Aubeny et al.

346 2003).

Global chain analyses

348 Figure 6 presents the development of padeye angle θ_a and tensions at mudline T_m and padeye

349 T_a as the chain is dragged. The padeye angle remained at 90° until a significant displacement,

350 approximately 3.5 m, was imposed at the dragging point, and then decreased almost linearly as

351 the dragging distance increased to the target 6.4 m. The delay in change of θ_a reflects the

352 gradually developing catenary profile, with parts of the chain at shallow depth cutting through

353 the soil much earlier than the chain near the attachment point. The tensions develop linearly

354 initially, but with a gradually increasing rate as the chain becomes taut. Minimal chain-weight

355 effect is shown when comparing the results of weightless and weighty chains, apart from slight

356 divergence of the tension responses at a large dragging distance.

357   The relationships between padeye angle and chain tension are shown in Figure 7, comparing

358   the results with the analytical solutions of Neubecker & Randolph (1995b). Although good

359   agreement is evident for the response at the anchor padeye (T_a-θ_a), the analytical solutions

360   indicate higher values of T_m for a given padeye angle than obtained from the CEL analyses.

361   This is discussed in more detail later.

362   Figure 8 compares the configurations of the embedded chain obtained from the CEL analyses

363   for selected chain inclination angles at the padeye (θ_a) during the tensioning process. The CEL

364   results of the chain profiles agree well with the analytical predictions derived from Eq. (4) with

365    $Q = Q_{u, deep} = 19d_b s_u$ and $\mu = \mu_p = 0.34$, indicating the robustness of the Neubecker & Randolph

366   relationships. It appears reasonable to use $Q_{u, deep}$ at all depths in Eq. (4) rather than adjust it to

367   allow for the decrease at shallow embedment. This reflects the small contribution of resistance

368   for the shallowly embedded part of the chain compared with the integrated resistance of the

369   whole chain. The additional embedment due to the chain weight is minimal in proximity to the

370   mudline (Figure 8b), reflecting the relatively low self-weight generated bearing pressure of the

371   chain ($w_c/d_b s_u = 4.2$ for the mudline shear strength of 2 kPa) compared with a limiting bearing

372   factor of over 12 (see Figure 4b).

373   The CEL results of the tension ratio T_a/T_m as a function of the decreasing padeye angle are

374   compared with analytical predictions in Figure 9a for the weightless chain. A gradual increase

375   of the tension ratio with increasing θ_a is shown for both CEL results and the Neubecker &

376   Randolph predictions by Eq. (3). However, the tension ratio obtained from the CEL analysis is

377   significantly higher than the predictions by Eq. (3), e.g. CEL gives $T_a/T_m = 0.90$ for $\theta_a = 75°$,

378   compared with 0.65 by Eq. (3) (Figure 9a).

379   The depth profiles of normalized tension T/T_a in the chain are shown in Figure 9b for the final

380   padeye angle of $\theta_a = 35°$ and chain weights of $w_c = 0$ and 1.5 kN/m. The CEL results show

381   much smaller exponential increases with distance from the padeye compared with the analytical

382   predictions consistent with the increased T_a/T_m ratios discussed above.

383   To globally re-evaluate the 'operative' coefficient of friction over the full length of the

384   embedded chain by Eq. (3), the CEL results of the tension ratio T_a/T_m as a function of the

385   dragging distance are compared in Figure 10 against the analytical predictions derived based

386   on the padeye angle-dragging distance relationships (Figure 6a) and Eq. (3) with varying

387   'operative' coefficient of friction μ values. Full mobilization of the (uncoupled) frictional

388   resistance corresponds to the case of $\mu = \mu_p = 0.34$. In contrast, the CEL T_a/T_m results agreed

389   with the analyses using $\mu = 0.07$, indicating that only about 20% on average of the potential

390   frictional capacity is mobilized along the embedded chain during the tensioning process. Note

391   that the operative coefficient of friction back-figured from the T_a/T_m profile does not affect

392   agreement of the CEL and Neubecker-Randolph T_a-θ_a responses (see Figure 7) since Eq. (5)

393   indicates that the chain tension at the padeye, T_a, is insensitive to the mobilized frictional

394   resistance.

395   The operative coefficient of friction provides a straightforward assessment of the partial

396   mobilization of the frictional resistance, even though it cannot rigorously define the mobilized

397   frictional resistance for each given anchor chain segment. This will be examined in the next

398   section.

399   ***Mobilized soil resistance along the anchor chain***

400   The evolution of normalized normal and frictional soil resistances, $Q/Q_{u, deep}$ and $F/F_{u, deep}$ for

401   two representative segments (identified as 'segment #13' and 'segment #25') in the weightless

402   chain during tensioning is shown in Figure 11. The mobilized frictional and bearing forces for

403   these elements are compared with the yield loci derived for the surficial and deeply embedded

404   chain for chain profiles (1), (2), (3) and (4), which correspond to dragging distances of 1.5, 3.0,

405   4.5 and 6.0 m respectively.

406   The evolution of the soil resistances reflects the cutting through of the chain during the

407   tensioning process. Both segments initially slide (axially) across the seabed, with the data points

408   for normal and tangential resistances travelling along the low Q/Q_u segment of the locus for the

409   surficial chain. Subsequently, the Q-F states transition between the yield locus for the surficial

410   chain and that for the deeply embedded section, corresponding to the soil failure at intermediate

411   depths as the chain cuts progressively deeper into the soil. Finally, the Q-F loading condition

412   on each chain segment moves towards a pure bearing condition, i.e. $Q \sim Q_{u,\,deep}$, and $F \sim 0$, as

413   the chain is tensioned.

414   The normalized normal and frictional soil resistances derived from CEL for all chain segments

415   at the mudline (z = 0), at intermediate depths (z = 0 to $3d_s$) and when deep (z = $3d_s$ to z_a) are

416   plotted in Figure 12 against the yield loci for surficial and deeply embedded chains at padeye

417   angles of $\theta_a = 75°$, 60°, 45° and 35°. It is evident that the Q-F data for the segments embedded

418   at depths from $3d_s$ (0.75 m) to z_a (9 m) lie close to the outer (deep embedment) yield locus,

419   while those for the surficial segments lie close to the inner (surficial) locus, with some transition

420   segments in between those two groups.

421   For the segments at the mudline, the operative F/Q ratios (i.e. μ in analytical equations) lie just

422   above the line of μ_p, showing 'sliding' behavior. For the deeply embedded sections, normal

423   resistance is close to fully mobilized ($Q \sim Q_{u,\,deep}$), while frictional forces are small ($F \sim 0$), so

424   that the friction ratio F/Q is considerably smaller than μ_p. This is attributed to the dominance of

425   normal movement, rather than axial movement, which is restricted by the padeye and the axial

426   stiffness of the chain. Due to the relatively small value of Q near the soil surface, the global

427   operative coefficient of friction is dominated by the F/Q ratios for the deeply embedded part of

428   the chain.

429   As the tensioning proceeds, the operative coefficient of friction obtained from the CEL analyses

430   may be expected to reduce as an increasing length of chain cuts through the soil to become fully

431   embedded, mobilizing a greater proportion of the full bearing resistance. This cannot be

432   captured rigorously by a simple constant coefficient of friction, even though a value of $\mu = 0.07$

433   gives a reasonable fit to the overall behavior (Figure 10).Method for calculation of the

434   equivalent frictional coefficient

435   The *Q-F* yield locus for the deeply embedded chain sections may be used as the basis for an

436   analytical approach to determine the *F*/*Q* ratio at each point along the embedded chain as a

437   function of embedment depth and the chain angle. In this section, a method is proposed based

438   on the chain configurations predicted by the Neubecker-Randolph solution.

439   The schematic of the yield locus is illustrated in Figure 13 together with the plastic potential.

440   The normality rule was adopted appropriate for deep undrained conditions (e.g. Bransby and

441   Randolph 1998; O'Neill et al. 2003) and based on the evidence shown in Figure 11. The

442   incremental (plastic) displacement normal and tangential to each chain segment defines the

443   plastic flow vector, i.e. $\tan\omega = \Delta u_t / \Delta u_n$, during the cutting through of the chain. Therefore, the

444   local equivalent coefficient of friction, μ_{le} in Figure 13, can be determined by

$$\mu_{le} = \frac{F_{deep}}{Q_{deep}} = \frac{N_s}{N_b} = \left[\frac{m}{n}\left(\frac{N_{s,ult}}{N_{b,ult}}\right)^n \tan\omega\right]^{1/(n-1)} = \left[\frac{m}{n}\left(\frac{N_{s,ult}}{N_{b,ult}}\right)^n \frac{\Delta u_t}{\Delta u_n}\right]^{1/(n-1)} \tag{9}$$

446   and assuming (for simplicity) that the normal resistance is fully mobilized ($Q_{deep} = Q_{u,\,deep}$).

447   Separate analyses were performed with the value of Q_{deep} derived rigorously according to the

448   yield locus. However, it was found for the cases considered here that this made no significant

449   difference to the results compared with assuming $Q_{deep} = Q_{u,deep}$. The equation can be modified

450   to allow for the chain weight and the chain inclination as

451
$$\mu_{le} = \left[\frac{m}{n} \left(\frac{N_{s,ult} + \dfrac{w_c \sin\theta}{d_b s_u}}{N_{b,ult} - \dfrac{w_c \cos\theta}{d_b s_u}} \right)^n \frac{\Delta u_t}{\Delta u_n} \right]^{1/(n-1)}$$
(10)

452   The relationship between μ_{le} and $\Delta u_t / \Delta u_n$ is therefore determined by Eqs. (9) and (10) with the

453   shape parameters m and n of the yield locus for the deeply embedded chain and the resistance

454   factors $N_{s,\,ult}$ and $N_{b,\,ult}$ already known.

455   The above equations can be used to analyze the response of the chain system in the soil as

456   follows:

457   Step 1: Estimate the configuration of an embedded chain for a given padeye angle θ_a using the

458   Neubecker and Randolph (1995b) solution (Eq. (4)) with $\mu = \mu_p = F_{u,\,deep}/Q_{u,\,deep}$.

459   Step 2: Update the configuration by applying a small incremental change of the padeye angle

460   (e.g. $\Delta\theta_a = 1.0 \times 10^{-8}$ rad) using the parameters from the previous step.

461   Step 3: Calculate the incremental displacement in the normal and tangential direction for each

462   individual chain element along the embedded chain.

463   Step 4: Calculate μ_{le} for each chain element using Eq. (9) or (10).

464   Step 5: Calcualte the updated mudline load and anchor padeye loads using the updated μ values.

465   Step 6: Return to step 2 and loop until the final design mudline load (or a target padeye chain

466   angle) is reached.

467   The values of $\Delta u_t/\Delta u_n$ and μ_{le} calculated from Eq.(9) are shown on Figure 14 as a fuction of the

468   embedment depth for selected configurations of an embedded chain with padeye angles $\theta_a =$

469   75°, 60°, 45° and 35°. The incremental displacement ratio $\Delta u_t/\Delta u_n$ at given depth z/z_a decreases

470   as padeye angle θ_a decreases (Figure 14(a)), reflecting a reduction of the tangential motion of

471 the chain as it is tensioned. For a given padeye angle, there is a rapid increase of $\tan\omega$ from the

472 padeye to the mudline, because of the relaxation of the axial movement constraint when remote

473 from the padeye. Similar to the trend of $\Delta u_t/\Delta u_n$ shown in Figure 14(a), μ_{le} decreases as the

474 padeye angle θ_a decreases, and there is a marked increase of μ_{le} from the padeye to the soil

475 surface (Figure 14(b)).

476 The values of μ_{le} derived from the proposed method are compared with the CEL results of the

477 F/Q in Figure 15 for cases of $w_c = 0$ and 1.5 kN/m. Note that segments in the vicinity of the

478 mudline were not considered in the analysis because the shallow part of the chain only

479 contributes a very small proportion of the overall resistance. Despite the fluctuations of the CEL

480 results, the overall predictions by Eqs. (9) and (10) agree well with the CEL results.

481 The above approach can also be used to calculate the average operative coefficient of the

482 friction μ_{op}^{mod} assuming normal resistance is fully mobilized ($Q = Q_{u,\,deep} = 19d_b s_u$):

$$\mu_{op}^{\text{mod}} = \frac{\displaystyle\int_{z_a}^{0} \mu_{le} Q_u}{\displaystyle\int_{z_a}^{0} Q_u} \tag{11}$$

484 This was further examined by comparing the predictions of T_a/T_m calculated by Eq.(3) adopting

485 μ_{op}^{mod} and constant μ_{op} of 0.07 with the CEL results during the tensioning process as shown in

486 Figure 16. The tension ratio calculated with μ_{op}^{mod} shows an increase from 0.91 when $\theta_a = 75°$

487 to just under 0.98 at when $\theta_a = 35°$. Despite fluctuations of the CEL results, the T_a/T_m values

488 derived from μ_{op}^{mod} show better agreement with the numerical results than the simple and

489 constant $\mu_{op} = 0.07$, particularly for small chain angles. This demonstrates the feasibility of the

490 proposed method for the calculation of the equivalent frictional coefficient of the embedded

491 chain.

492   **CONCLUSIONS**

493   A series of large deformation finite element analyses using the CEL approach were conducted

494   to investigate the interaction between chain elements and the soil during different displacement

495   trajectories and also the global system response of a catenary anchor chain cutting through soil.

496   The results provide new insight into the response of an anchor chain system, in particular the

497   operative axial 'friction' along the chain. A new design method is proposed for predicting

498   tension and angle along the inverse catenary of an embedded anchor chain.

499   The CEL results of the configuration of the embedded chain and the tension at the padeye

500   showed good agreement with analytical predictions, implying the suitability of assuming that

501   normal soil resistance is fully mobilized when performing design calculations (as previously

502   suggested by Neubecker and Randolph (1995b)). However, the CEL results showed

503   significantly less tension attenuation from the mudline to the padeye of the embedded chain

504   compared to previously published design approaches. This was shown to be due to the low

505   proportion of the potential frictional capacity that is mobilized over the embedded chain during

506   chain cutting. Partial mobilization of the frictional resistance was shown to result from

507   interaction between the normal and frictional soil resistances as the embedded chain cuts

508   through the soil, with almost full normal resistance and thus low frictional resistance being

509   mobilized through most of the embedded part of chain governed by the stationary padeye

510   condition. Consequently, the local mobilized coefficient of friction was shown to decrease

511   significantly with increasing depth. The global 'operative' coefficient of friction, taking account

512   of increasing shear strength with depth, is weighted towards the chain response at depth and

513   hence is much lower than the nominal potential friction coefficient.

514   Equations for yield loci for the embedded chain were obtained from the numerical results and

515   normality was demonstrated for the embedded chain sections. This allowed development of a

516   new method for calculating the local equivalent coefficient of friction (i.e. *F/Q*) at every point

517   along the chain, based on the yield locus and the chain configuration, and hence the incremental

518   displacement trajectories at each point. A formulation was also developed to estimate the global

519   operative coefficient of friction acting along the chain, facilitating more accurate calculation of

520   the tension variation along the embedded anchor chain. The resulting analytical approach

521   showed good agreement with the CEL results locally and globally, confirming the potential for

522   the approach for design of mooring systems with embedded anchor padeyes.

533   **REFERENCES**

534   Aubeny, C. P., Han, S. W., and Murff, J. D. (2003). "Inclined load capacity of suction caissons."

535   *International Journal for Numerical and Analytical Methods in Geomechanics*, 27(14), 1235-

536   1254.

537   Bhattacharjee, S., Majhi, S. M., Smith, D., and Garrity, R. (2014). "Serpentina FPSO Mooring

538   Integrity Issues and System Replacement: Unique Fast Track Approach." In *Proc., Offshore*

539   *Technology Conference*, Offshore Technology Conference.

540   Bransby, M. F., and Randolph, M. F. (1998). "Combined loading of skirted foundations."

541   *Géotechnique*, 48(5), 637-655.

542   Colliat, J.-L., Safinus, S., Boylan, N., and Schroeder, K. (2018). "Formation and Development

543   of Seabed Trenching from Subsea Inspection Data of Deepwater Gulf of Guinea Moorings." In

544   *Proc., Offshore Technology Conference*, Offshore Technology Conference.

545   Dassault Systèmes (2014). *ABAQUS 6.14 Analysis User's Manual*, Simulia Corp, Providence,

546   RI, USA.

547   Degenkamp, G., and Dutta, A. (1989). "Soil resistances to embedded anchor chain in soft clay."

548   *Journal of Geotechnical Engineering*, 115(10), 1420-1438.

549   Dutta, A., and Degenkamp, G. (1989). "Behaviour of embedded mooring chains in clay during

550   chain tensioning." In *Proc., Offshore Technology Conference*, Offshore Technology

551   Conference.

552   Ehlers, C. J., Chen, J., Roberts, H. H., and Lee, Y. C. (2005). "The origin of near-seafloor crust

553   zones in deepwater." In *Proc., Proc. Int. Symp. on Frontiers in Offshore Geotechnics: ISFOG*,

554   927-934.

555   Frankenmolen, S. F., White, D. J., and O'Loughlin, C. D. (2016). "Chain-Soil Interaction in

556   Carbonate Sand." In *Proc., Offshore Technology Conference*, Offshore Technology

557   Conference.

558   Gault, J. A., and Cox, W. R. (1974). "Method for predicting geometry and load distribution in

559   an anchor chain from a single point mooring buoy to a buried anchorage." In *Proc., Offshore*

560   *Technology Conference*, Offshore Technology Conference.

561   Han, C., and Liu, J. (2017). "A modified method to estimate chain inverse catenary profile in

562   clay based on chain equation and chain yield envelope." *Applied Ocean Research*, 68, 142-153.

563   Kim, Y., Hossain, M. S., and Wang, D. (2015). "Dynamically Installed Anchors: Performance

564   of embedded mooring chain profile in clay." *Frontiers in Offshore Geotechnics III*, Meyer, ed.,

565   Taylor & Francis Group, London, 881-886.

566 Neubecker, S., and Randolph, M. (1995a). "Performance of embedded anchor chains and

567 consequences for anchor design." In *Proc., Offshore Technology Conference*, Offshore

568 Technology Conference.

569 Neubecker, S., and Randolph, M. (1995b). "Profile and frictional capacity of embedded anchor

570 chains." *Journal of geotechnical engineering*, 121(11), 797-803.

571 Neubecker, S., and Randolph, M. (1996). "The performance of drag anchor and chain systems

572 in cohesive soil." *Marine georesources & geotechnology*, 14(2), 77-96.

573 O'Neill, M. P., Bransby, M. F., and Randolph, M. F. (2003). "Drag anchor fluke soil interaction

574 in clays." *Canadian Geotechnical Journal*, 40(1), 78-94.

575 Randolph, M., and Gourvenec, S. (2011). *Offshore geotechnical engineering*, CRC Press.

576 Reese, L. C. (1973). "A design method for an anchor pile in a mooring system." In *Proc.,*

577 *Offshore Technology Conference*, Offshore Technology Conference.

578 Rocha, M., Schnaid, F., Rocha, C., and Amaral, C. (2016). "Inverse catenary load attenuation

579 along embedded ground chain of mooring lines." *Ocean Engineering*, 122, 215-226.

580 Vivatrat, V., Valent, P. J., and Ponterio, A. A. (1982). "The influence of chain friction on anchor

581 pile design." In *Proc., Offshore Technology Conference*, Offshore Technology Conference.

582 Wang, D., Bienen, B., Nazem, M., Tian, Y., Zheng, J., Pucker, T., and Randolph, M. F. (2015).

583 "Large deformation finite element analyses in geotechnical engineering." *Computers and*

584 *Geotechnics*, 65, 104-114.

585 Yen, B. C., and Tofani, G. D. (1984). "Soil resistance to stud link chain." In *Proc., Offshore*

586 *Technology Conference*, Offshore Technology Conference.

587 Zhao, Y., and Liu, H. (2013). "Large deformation finite element analysis of the anchor line

588 embedded in seabed soils." In *Proc., ASME 2013 32nd International Conference on Ocean,*

589 *Offshore and Arctic Engineering*, American Society of Mechanical Engineers,

590 V006T010A024-V006T010A024.

591 Zhao, Y., and Liu, H. (2014). "Numerical simulation of drag anchor installation by a large

592 deformation finite element technique." In *Proc., ASME 2014 33rd International Conference on*

593 *Ocean, Offshore and Arctic Engineering*, American Society of Mechanical Engineers,

594 V003T010A011-V003T010A011.

595 Zhao, Y., and Liu, H. (2016). "Numerical implementation of the installation/mooring line and

596 application to analyzing comprehensive anchor behaviors." *Applied Ocean Research*, 54, 101-

597 114.

598   **Figure captions**

599   Figure 1. A global embedded chain and the mechanical equilibrium of a chain element

600   Figure 2. CEL modelling of the embedded chain

601   Figure 3. Capacities of the embedded chain model with different penetrating/sliding velocities
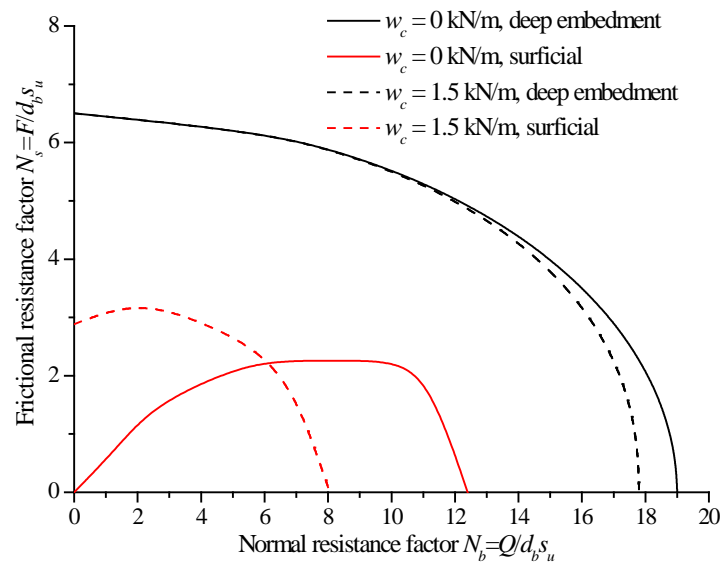
602   Figure 4. Probe testing results of the weightless chain model

603   Figure 5. Yield loci of the embedded chain model

604   Figure 6. Development of padeye angle and forces of the full chain system during tensioning

605   Figure 7. Comparison of tension-padeye angle relationships

606   Figure 8. Configurations of the embedded chain for selected chain inclination angles at the
607   padeye during tensioning

608   Figure 9. Tension in the chain

609   Figure 10. Effect of operative coefficient of friction of the embedded chain on the tension
610   ratio

611   Figure 11. Evolution of the soil resistances to representative chain segments during tensioning

612   Figure 12. Distributions of the soil resistances of the embedded chain with the yield loci

613   Figure 13. Schematic of the yield locus and the plastic potential function

614   Figure 14. Example of the calculation for the local equivalent coefficient of friction based on
615   the configurations solved by the Neubecker & Randolph solution

616   Figure 15. Distribution of the local equivalent coefficient of friction of the chain with
617   different padeye angles

618   Figure 16. Tension ratio calculated with modified operative frictional coefficient

Figure 1. A global embedded chain and the mechanical equilibrium of a chain element

(a) Chain model corresponding to the $5d_b$ studless chain
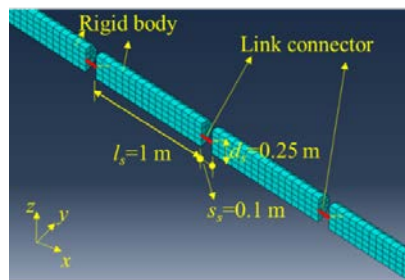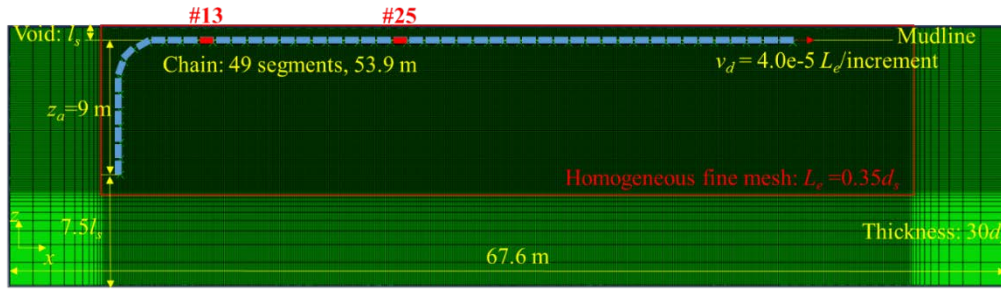
(b) FE model setup

Figure 2. CEL modelling of the embedded chain
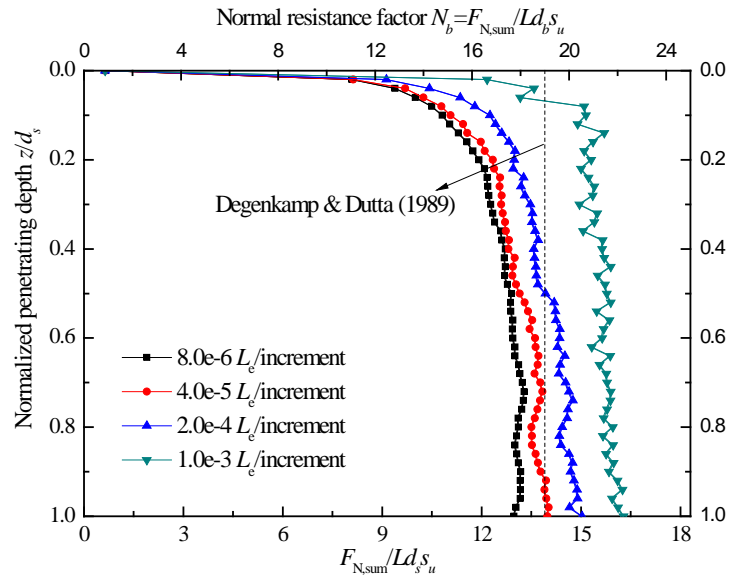
(a) Normal
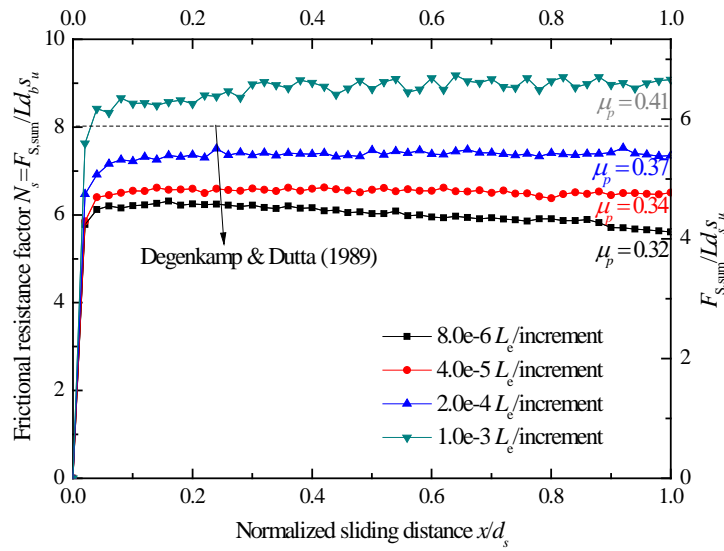
(b) Frictional

Figure 3. Capacities of the embedded chain model with different penetrating/sliding velocities

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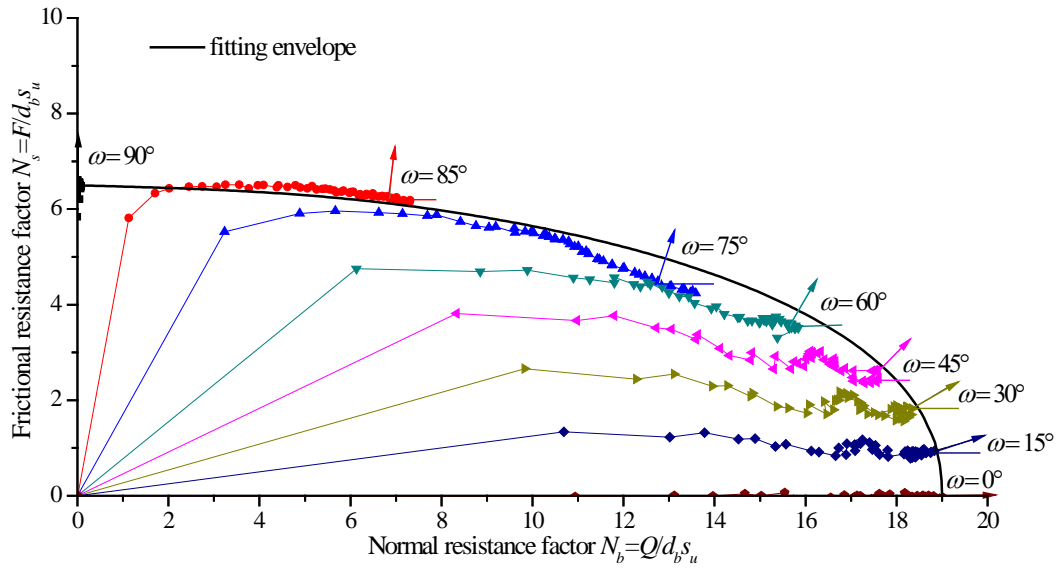
(a) Deep embedment (18d_s)
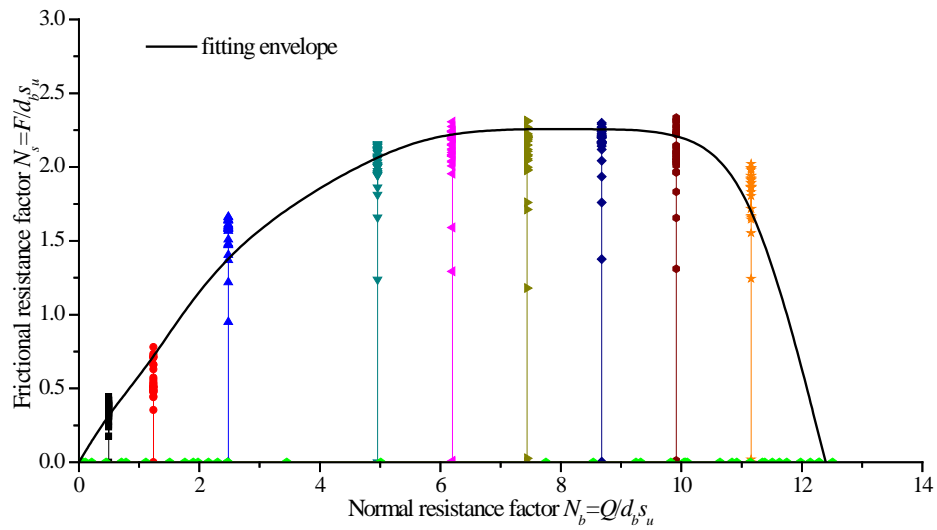
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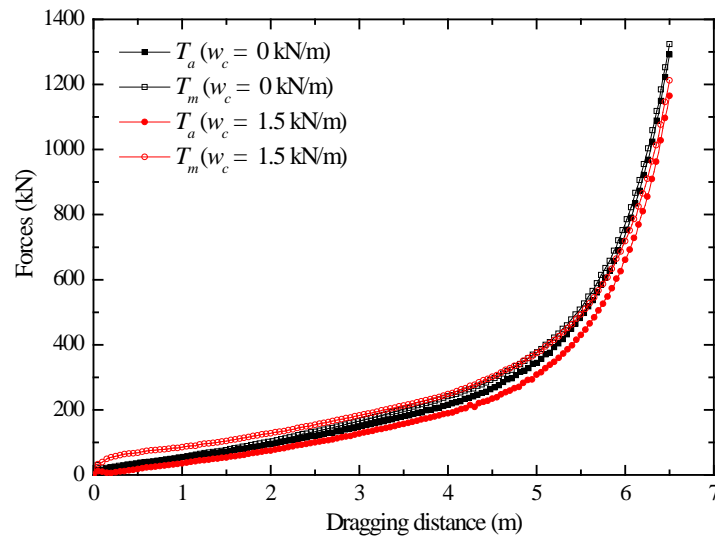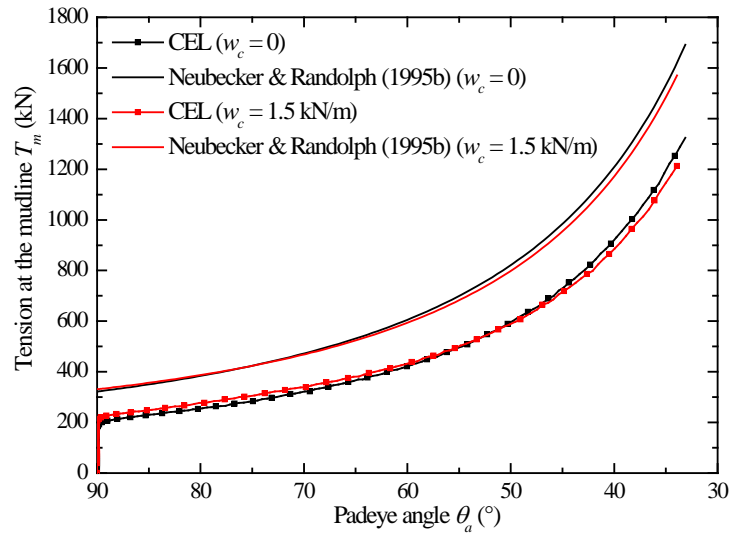(b) Surficial

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Figure 4. Probe testing results of the weightless chain model

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638   Figure 5. Yield loci of the embedded chain model
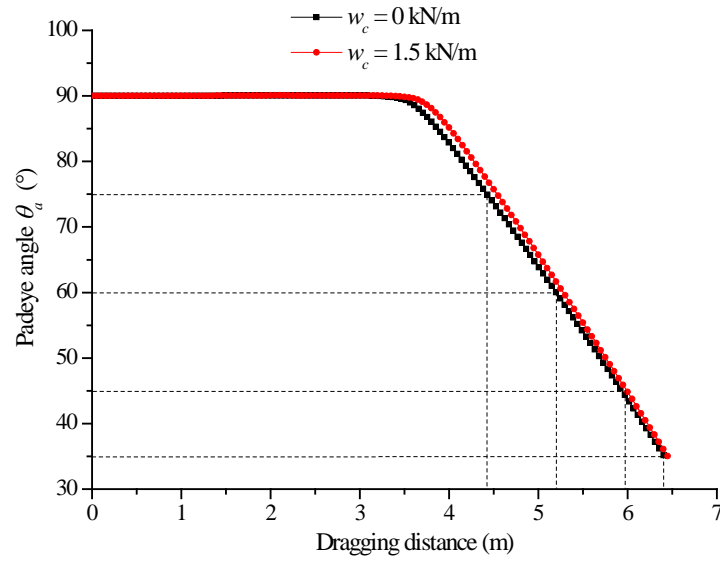
(a) Padeye angle

(b) Mudline and padeye tensions

Figure 6. Development of padeye angle and forces of the full chain system during tensioning

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645

(a) Mudline tension

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647

(b) Padeye tensions

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Figure 7. Comparison of tension-padeye angle relationships

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(a) $w_c = 0$ kN/m

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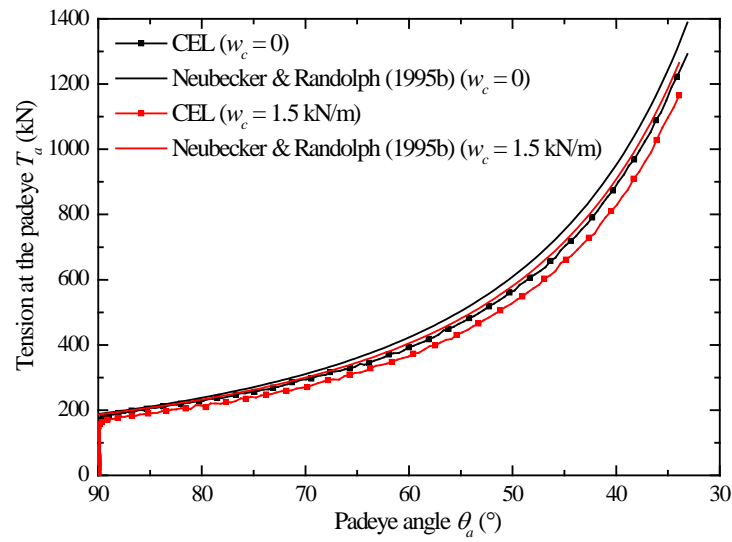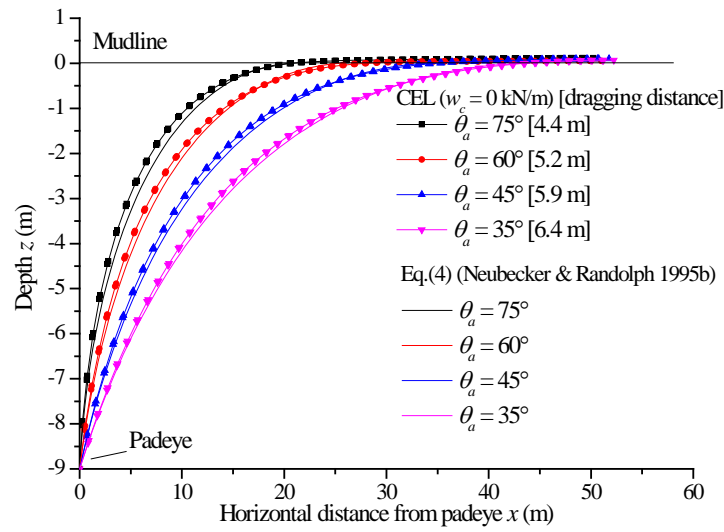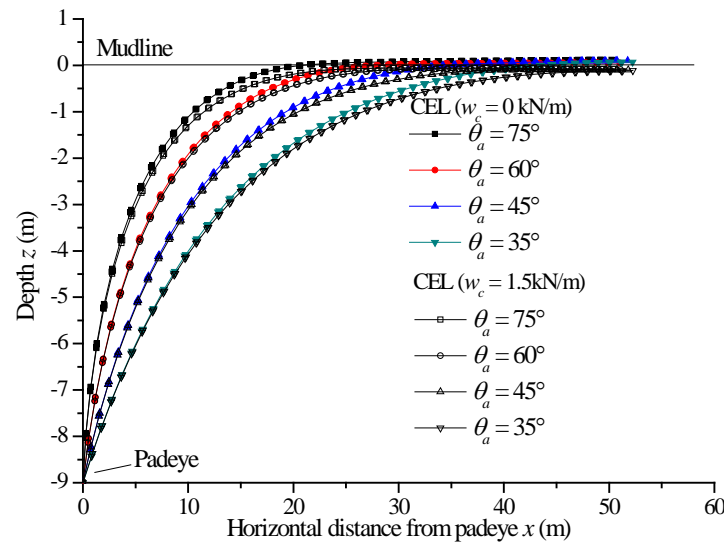
(b) Chain weight effect

654   Figure 8. Configurations of the embedded chain for selected chain inclination angles at the

655   padeye during tensioning

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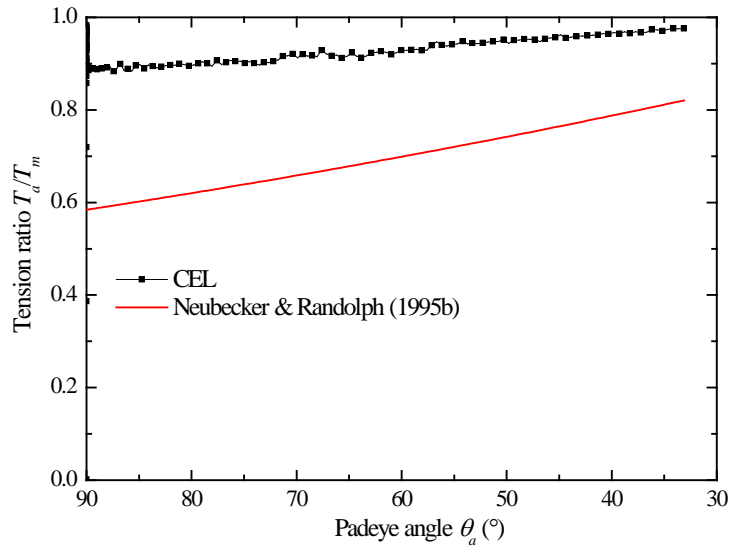
657   (a) Tension ratio during tensioning ($w_c = 0$)

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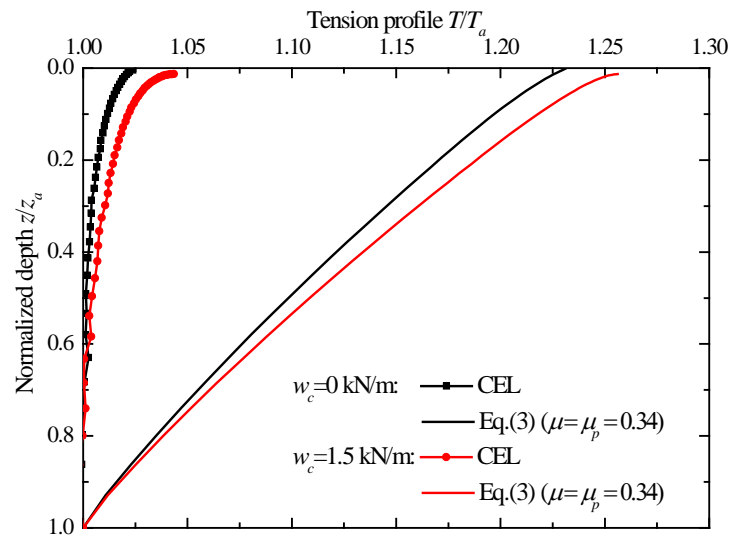
659   (b) Tension conditions in the chain at the padeye angle of 35°
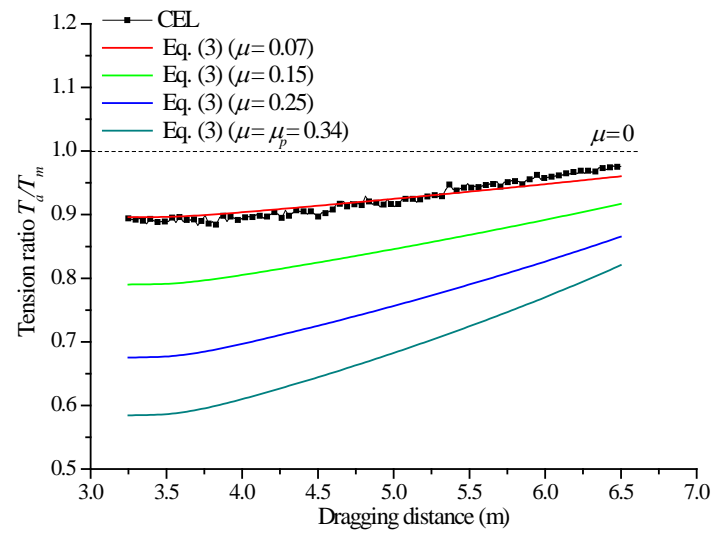
660   Figure 9. Tension in the chain

Figure 10. Effect of operative coefficient of friction of the embedded chain on the tension ratio
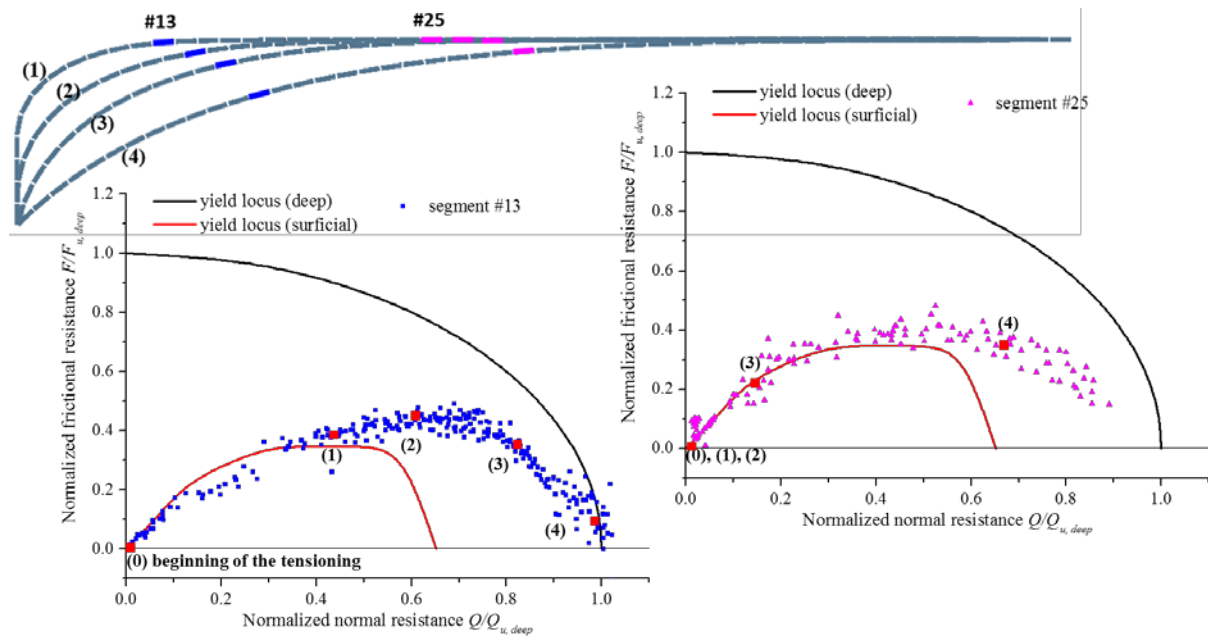
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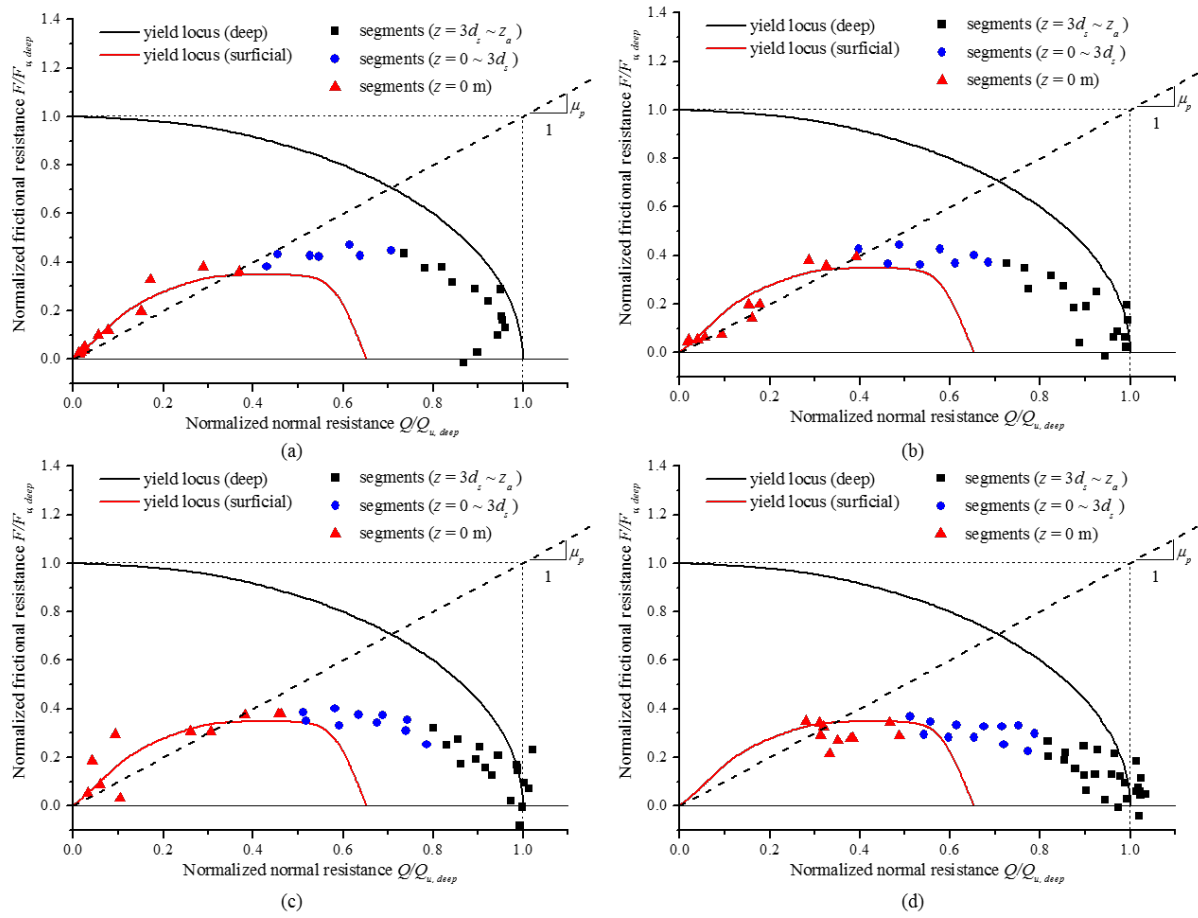665   Figure 11. Evolution of the soil resistances to representative chain segments during tensioning

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668   Figure 12. Distributions of the soil resistances of the embedded chain with the yield loci:

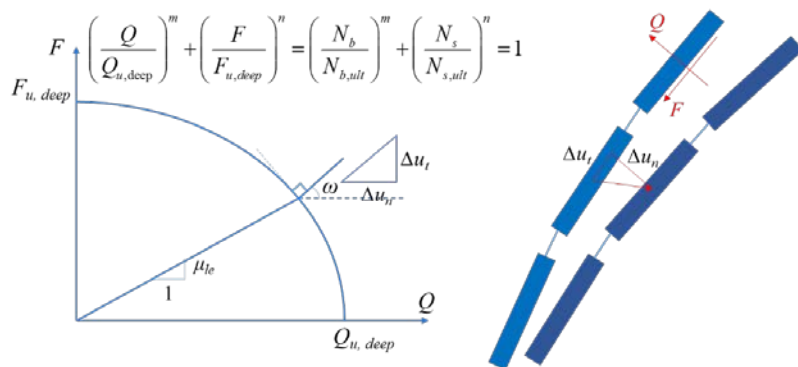669   (a) $\theta_a = 75°$ (b) $\theta_a = 60°$ (c) $\theta_a = 45°$ (d) $\theta_a = 35°$

$$\left(\frac{Q}{Q_{u,deep}}\right)^m + \left(\frac{F}{F_{u,deep}}\right)^n = \left(\frac{N_b}{N_{b,ult}}\right)^m + \left(\frac{N_s}{N_{s,ult}}\right)^n = 1$$

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671

672   Figure 13. Schematic of the yield locus and the plastic potential function

(a)

(b)
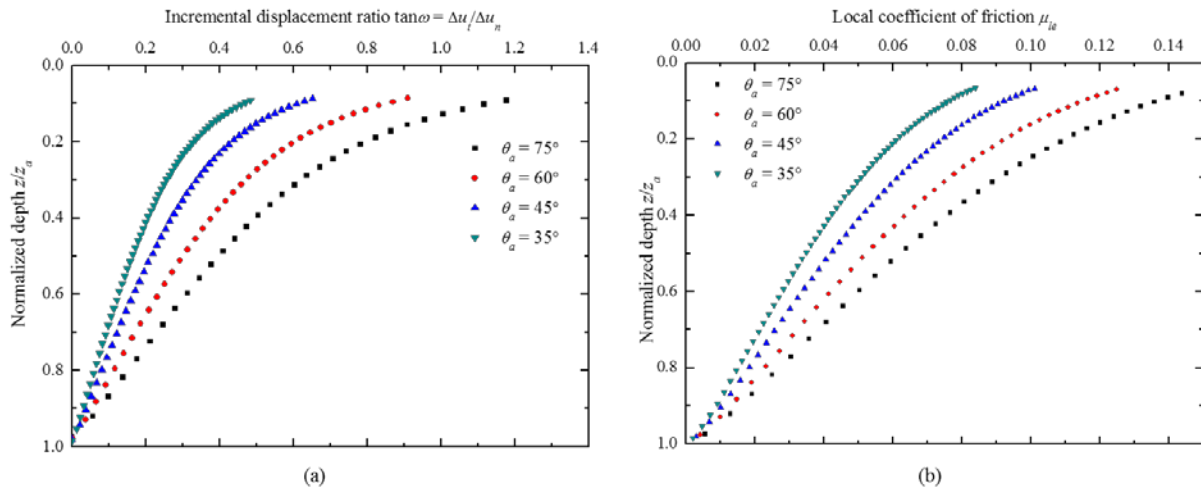
Figure 14. Example of the calculation for the local equivalent coefficient of friction based on

the configurations solved by the Neubecker & Randolph solution: (a) Incremental

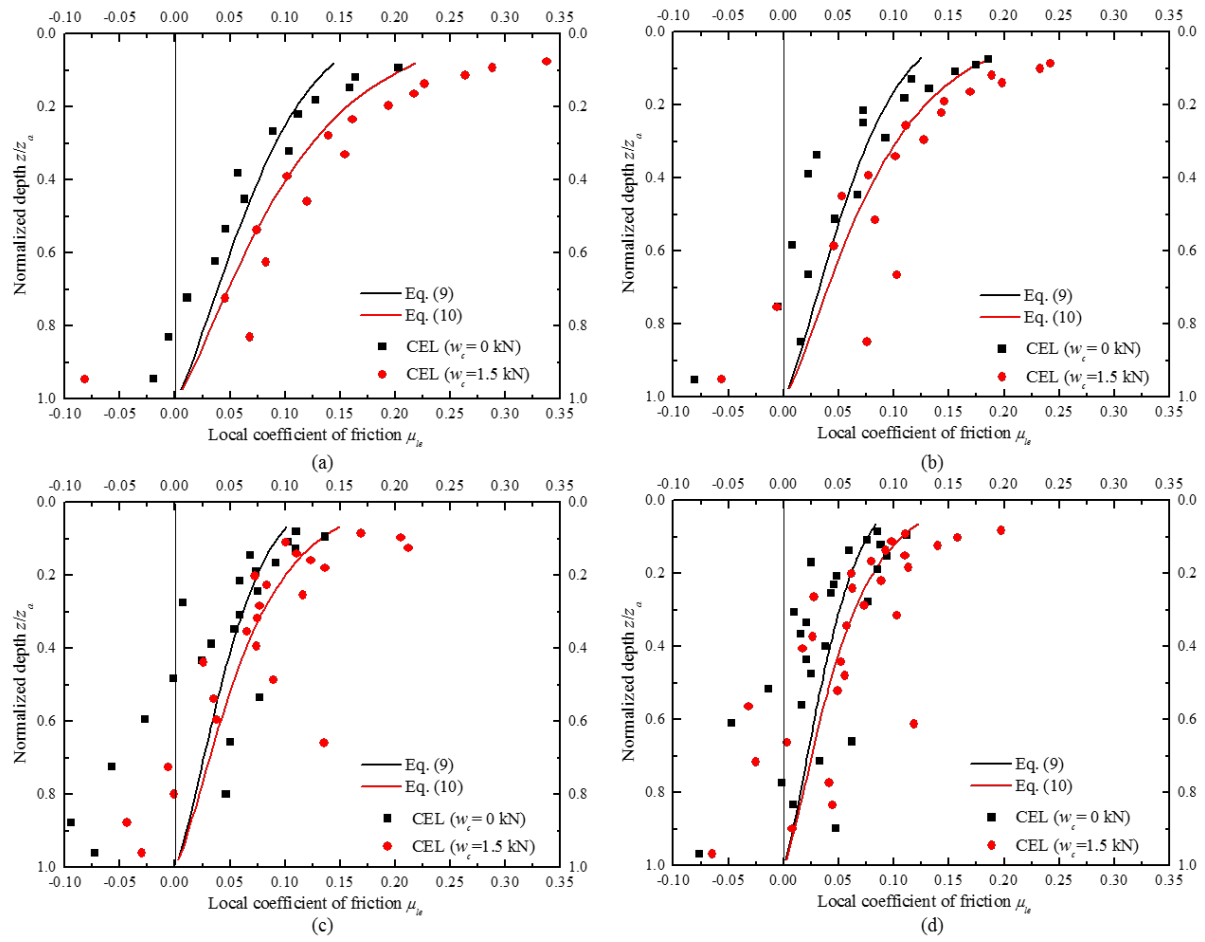displacement ratio; (b) Local equivalent coefficient of friction

Figure 15. Distribution of the local equivalent coefficient of friction of the chain with different padeye angles: (a) $\theta_a = 75°$ (b) $\theta_a = 60°$ (c) $\theta_a = 45°$ (d) $\theta_a = 35°$
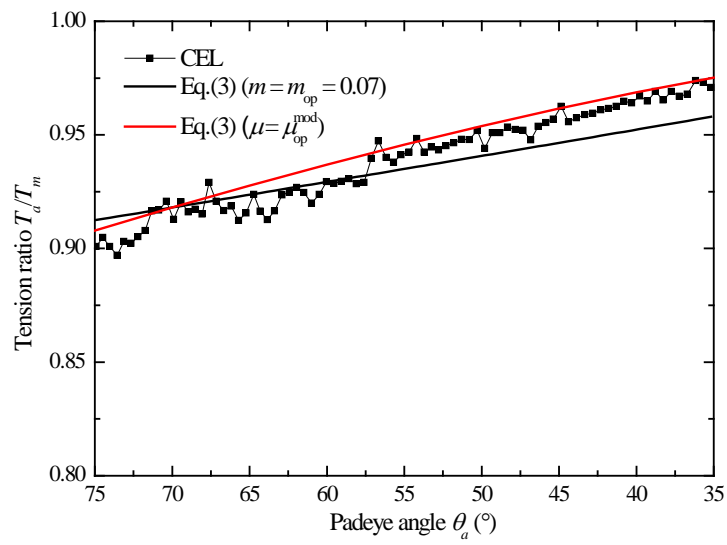
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682   Figure 16. Tension ratio calculated with modified operative frictional coefficient