

Response of the temporal turbulent boundary layer to decaying free-stream turbulence

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The turbulent boundary layer developing under a turbulence-laden free stream is numerically investigated using the temporal boundary layer framework. This study focuses on the interaction between the fully turbulent boundary layer and decaying free-stream turbulence. Previous experiments and simulations of this physical problem have considered a spatially evolving boundary layer beset by free-stream turbulence. The state of the boundary layer at any given downstream position in fact reflects the accumulated history of the co-evolution of boundary layer and free-stream turbulence. The central aim of the present work is to isolate the effect of local free-stream disturbances existing at the same time as the ‘downstream’ boundary layer. The temporal framework used here helps expose when and how disturbances directly above the boundary layer actively impart change upon it. The bulk of our simulations were completed by seeding the free stream above boundary layers that were ‘pre-grown’ to a desired thickness with homogeneous isotropic turbulence from a precursor simulation. Moreover, this strategy allowed us to test various combinations of the turbulence intensity and large-eddy length scale of the free-stream turbulence with respect to the corresponding scales of the boundary layer. The relative large-eddy turnover time scale between the free-stream turbulence and the boundary layer emerges as an important parameter in predicting if the free-stream turbulence and boundary layer interaction will be ‘strong’ or ‘weak’ before the free-stream turbulence eventually fades to a negligible level. If the large-eddy turnover time scale of the free-stream turbulence is much smaller than that of the boundary layer, the interaction will be ‘weak’, as the free-stream disturbances will markedly decay before the boundary layer is able to be altered significantly as a result of the free-stream disturbances. For a ‘strong’ interaction, the injected free-stream turbulence causes increased spreading of the boundary layer away from the wall, permitting large incursions of free-stream fluid deep within it.

Key words: turbulence simulation, turbulent boundary layers

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1. Introduction

Almost every boundary layer created in an engineering or environmental context is in fact exposed to free-stream disturbances. The present numerical study considers the interaction of decaying free-stream turbulence (FST) with the fully turbulent temporal boundary layer to determine the conditions under which these free-stream disturbances are able to actively impart change upon the boundary layer.

A boundary layer developing under a free stream laden with disturbances will tend to exhibit increased skin friction and mass or heat transfer (Blair 1983*a*). Considerable effort (Hancock & Bradshaw 1983; Blair 1983*b*; Castro 1984) has thus been made to correlate observed increases in skin friction coefficient C_f and mass (or heat) transfer coefficient St to parameters of the FST and the boundary layer. Detailed statistics have been reported by previous workers, yet are generally given at a limited number of downstream locations in an experimental facility (Nagata, Sakai & Komori 2011; Sharp, Neuscamman & Warhaft 2009; Dogan, Hanson & Ganapathisubramani 2016). The current methodology is able to observe the entire interaction as it unfolds and seeks to advance our understanding of the boundary layer–FST interaction via detailed direct numerical simulation (DNS).

To date, the problem of the boundary layer developing under FST has been principally investigated experimentally. The DNS of a fully turbulent boundary layer developing under FST is an expensive undertaking that precludes systematic studies. When simulating a turbulent boundary layer with a quiescent free stream, a stretched grid is typically used far away from the wall-bounded turbulent flow. The present physical problem demands adequate resolution of the free stream with its disturbances. Previous numerical investigations have generally made use of either large-eddy simulation (e.g. Li, Schlatter & Henningson 2010; Péneau, Boisson & Djilali 2000) or DNS with modest Reynolds numbers (the study of Xia *et al.* (2014) achieved a final momentum thickness Reynolds number $Re_\theta \approx 250$). Yet there have been many studies considering the transition of an incoming laminar boundary layer under FST (Brandt, Schlatter & Henningson 2004; Hack & Zaki 2014; Kreilos *et al.* 2016). Nominally a transitional study, Wu *et al.* (2017) nevertheless achieved a final $Re_\tau \approx 1000$ for a relatively weak inlet turbulence of 3% of the mean free-stream velocity. Recently, You & Zaki (2019) presented a DNS of a spatially developing boundary layer over the range $Re_\theta = 1200$ –3200 for an incoming turbulence intensity of 10%.

Hancock & Bradshaw (1989) suggested that the relative fluctuating strain rate between FST and boundary layer was an important quantity to characterise their interaction. Formed from the large-eddy length scales and velocity scales of the respective flows, it may be recast as the relative large-eddy turnover time scale between the FST and boundary layer, evolving as the boundary layer grows and the unforced free-stream disturbances decay. A natural opportunity to study the evolving relative large-eddy time scale of the current physical problem is provided by the temporal framework. Kozul, Chung & Monty (2016) demonstrated that the temporal boundary layer is a good model for the incompressible spatially developing turbulent boundary layer both analytically and via comparison of various statistics between the spatial and temporal boundary layers. Additionally, under a quiescent free stream, the mean entrainment of non-turbulent fluid by the turbulent temporal boundary layer $E = d\delta/dt = U_\infty d\delta/dX$ (where δ is the boundary layer thickness, U_∞ is the free-stream velocity and $X = U_\infty t$ for time t) is not unlike the process in a turbulent spatial boundary layer $E = U_\infty d\delta/dx - W_\delta$ (where W_δ is the mean wall-normal velocity at the edge of the boundary layer). The difference in mean

entrained fluid is due only to the small W_δ in the spatial boundary layer that vanishes at large Reynolds number. Thus the temporal boundary layer will capture the finite, non-vanishing part of the entrainment in the asymptotic limit of the spatial boundary layer, i.e. $E \rightarrow 0.22 U_\tau$, where U_τ is the friction velocity (cf. coefficients a_2 and b_2 in figure 18 of Kozul *et al.* (2016)). The Reynolds numbers of the present simulations, although in the fully turbulent regime, clearly fall short of this asymptotic limit. The current temporal model is therefore a potential source of inaccuracy if direct comparison of the entrainment to that of the spatial boundary layer is sought.

The efficiency of the temporal framework, which employs a streamwise-shortened domain, allows us to mitigate some of the cost associated with this demanding physical problem. Whilst a wide-ranging scan of length scales and intensities would be ideal to determine the roles of each in the interaction with the boundary layer, in practice we are limited to cases where the free-stream length scale is a small multiple of the boundary layer thickness. The integral length scale of the FST, growing as its intensity decays in time, must remain much smaller than the domain size such that the associated large-scale energy-carrying eddies evolve freely (Thorner 2016). A simulation where the large-eddy length scale of the FST is much larger than that of the boundary layer thickness is untenable given present computational capabilities: it would require the vast majority of the domain, that is, available computational resources, to be dedicated to simulating the FST, when our primary concern here is its interaction with the boundary layer. In fact, the response of the boundary layer to small-scale turbulence in the free stream remains rather under-explored compared to that of large-scale FST (Nagata *et al.* 2011). Nevertheless, the present efficient temporal framework permits a limited parametric investigation of this costly physical problem. In addition to exposing a boundary layer to FST from its inception, the present work gains access to other regimes by adding or injecting homogeneous isotropic turbulence (HIT) to the free stream of boundary layers already grown to a desired thickness in a quiescent free stream. Such an approach making use of synthesised fields was previously used for wakes developing under free-stream disturbances (Rind & Castro 2012).

Since many engineering problems feature turbulent boundary layers exposed to ambient free-stream conditions that cannot realistically be considered laminar, our work helps to clarify when and how such free-stream disturbances could, via active manipulation, alter the form and development of boundary layers forming over walls. The present parametric study of (wall-bounded) shear flow with FST complements previous systematic numerical campaigns concerning shear flows subject to free-stream disturbances, including wakes (Rind & Castro 2012), stratified wakes (Pal & Sarkar 2015) and shear layers (Kaminski & Smyth 2019). We show how the relative large-eddy turnover time scale indicates whether there will be a ‘strong’ or ‘weak’ interaction between the two flows. If the large-eddy turnover time scale of the boundary layer is less than approximately twice that of the FST, the free-stream disturbances will have time to impart change on the boundary layer before the FST fades away. From the boundary layer’s point of view, it needs time to adjust to the FST via ingestion of the inactive motions from the free stream. Significant changes to the boundary layer eventuate only if the FST is still relatively strong by the time this occurs. Previous equilibrium approaches have attempted parametrisation using physical quantities at a single point in space or time. In contrast, the present temporal simulations expose the inherent developing nature of this physical problem.

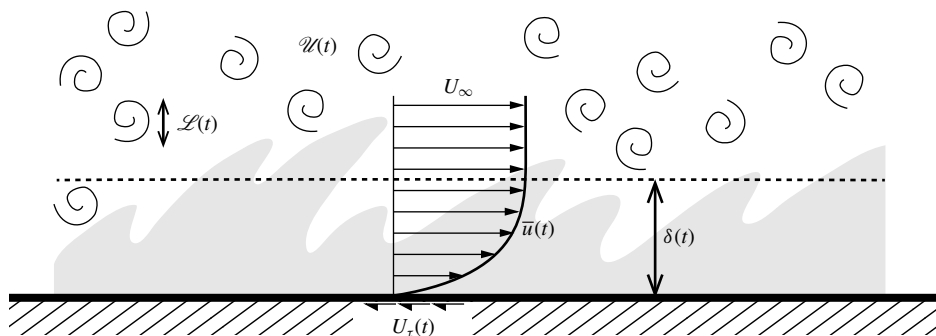


FIGURE 1. Sketch of the physical problem: a turbulent temporal boundary layer developing under decaying FST. The set-up employs a periodic boundary condition in the streamwise direction.

2. Velocity and length scales of the boundary layer–FST problem

The FST to which boundary layers are often exposed will herein be modelled as HIT. The large scales of this HIT will be characterised by a velocity scale (\mathcal{U}) and an integral length scale (\mathcal{L}). Figure 1 sketches the physical problem within the temporal framework. Such an approach is particularly suited to the problem since the evolution of HIT is classically described by temporal decay, and the boundary layer being recast thus (Kozul *et al.* 2016) allows us to directly compare the evolution of the relative large-eddy turnover time scales of the two flows.

To parametrise our physical problem, we estimate how these scales of the HIT evolve with respect to the relevant velocity scale (friction velocity U_τ) and large-eddy length scale (δ ; for 99% boundary layer thickness $\delta \equiv \delta_{99}$, computed from the mean streamwise velocity profile) of the boundary layer. Whether the large scales in decaying HIT are described by the Batchelor or Saffman theories of turbulence is a long-standing debate not entered into by the present work. The following relations are only of interest here as we endeavour to establish how the scales of the boundary layer and FST would evolve with respect to each other assuming no interaction between them. It is generally agreed (e.g. Krogstad & Davidson 2010) that both \mathcal{U} and \mathcal{L} evolve temporally according to power laws; the two classical theories suggest differing exponents. In the Batchelor (1953) theory, integral scales \mathcal{U} and \mathcal{L} satisfy $\mathcal{U}^2 \mathcal{L}^5 = \text{constant}$, and when combined with the empirical relation

$$\frac{d\mathcal{U}^2}{dt} = -A \frac{\mathcal{U}^3}{\mathcal{L}}, \quad (2.1)$$

for some constant A , the decay law $\mathcal{U}^2 \sim t^{-10/7}$ (and associated $\mathcal{L} \sim t^{2/7}$) results. The theory due to Saffman (1967) predicts the group $\mathcal{U}^2 \mathcal{L}^3 = \text{constant}$ which gives $\mathcal{U}^2 \sim t^{-6/5}$ (and $\mathcal{L} \sim t^{2/5}$). The two classical types of turbulence are associated with specific forms of the energy spectrum E : for the Batchelor type $E(\kappa \rightarrow 0) \sim \kappa^4$ for wavenumber κ , whereas Saffman turbulence has the spectrum $E(\kappa \rightarrow 0) \sim \kappa^2$. Which form of turbulence is exhibited, and importantly what value of decay rate arises, depends upon initial conditions (Lavoie, Djenidi & Antonia 2007; Antonia *et al.* 2013; Hearst & Lavoie 2016), but it would appear that the turbulence retains the spectrum (either $\sim \kappa^2$ or $\sim \kappa^4$) with which it was created (Ishida, Davidson & Kaneda 2006). The decay exponent of classic grid turbulence appears to be closer to that suggested by the

Saffman spectrum (Krogstad & Davidson 2010), a conclusion consistent with DNS of temporal grid turbulence (Watanabe & Nagata 2018). Both $E \sim \kappa^2$ (Huang & Leonard 1994; Mansour & Wray 1994) and $E \sim \kappa^4$ (Ishida *et al.* 2006; Thornber 2016) energy spectra have been used to initialise the flow fields of numerical simulations.

The choice of a velocity scale \mathcal{U} for the FST is usually set to be the streamwise root-mean-squared velocity fluctuations u'_e , for comparison to experiments; however, since our HIT is perfectly isotropic any velocity component could have been chosen. The choice of a suitable length scale is rather less obvious. A length scale L_e^u was defined by Hancock & Bradshaw (1983) as

$$U_\infty \frac{d(u'_e)^2}{dX} \equiv \frac{-(u'_e)^3}{L_e^u}, \quad (2.2)$$

for mean streamwise free-stream velocity U_∞ and distance from the turbulence-producing grid X . Several alternative definitions for the energy-carrying integral length scale of HIT have been offered in the literature. A common definition is the value of the integrated normalised autocorrelation to the first zero crossing, r_0 :

$$L_{uu} = \int_0^{r_0} \frac{\overline{u'(x)u'(x+r)}}{\overline{u'^2}} dr, \quad (2.3)$$

as used in Hearst, Dogan & Ganapathisubramani (2018) for example. However, this quantity can be problematic since this zero crossing is somewhat elusive (Dogan *et al.* 2016). The non-dimensional dissipation rate ($C_\varepsilon = \varepsilon L_{uu}/u'_e$) for the current forced HIT is $C_\varepsilon \approx 0.5$, in agreement with the spread of values found for forced HIT in the survey of Kaneda *et al.* (2003). When forcing is turned off within the triply periodic domain, C_ε gradually increases over $t \approx 2T_{e,0}$ to $C_\varepsilon \approx 1.8$ (where $Re_\lambda = u'\lambda/\nu$ is decreasing and is ≈ 30 at this point). However, this value for the dissipation rate is neither well-converged nor particularly reliable, since at this later time the growing integral length scale L_{uu} exceeds 10% of the smallest box dimension. For perfectly isotropic turbulence, the length scale L_e^u from (2.2) can be written as

$$L_e^u = \frac{3}{2} \frac{(u'_e)^3}{\varepsilon}, \quad (2.4)$$

for kinetic energy dissipation rate $\varepsilon \equiv \nu \overline{(\partial u'_i / \partial x_j)^2}$ with kinematic viscosity ν . However, as pointed out in Hearst *et al.* (2018), associating this dissipation-derived quantity with a length scale actually existing in the flow is not always a valid undertaking. Our present use of (2.4) to derive a relevant length scale does not suggest we have an equilibrium state during the decaying phase, as (2.2) assumes. Rather we use it to avoid the ambiguity associated with L_{uu} due to a limited domain size. We use the term ‘large-eddy length scale’ throughout when referring to that of the FST since we are most commonly comparing it to the ‘large-eddy length scale’ of the boundary layer, δ (indeed we will most frequently refer to the ‘large-eddy length scale ratio’, L_e^u/δ). We formally refer to L_{uu} as the ‘integral length scale’. The dissipation-based L_e^u is taken as being representative of large eddies in the FST since it is well defined for restricted numerical domains and dissipation-based length scales are commonly used (e.g. You & Zaki 2019). Later in this work it is shown that using either the dissipation-based L_e^u from (2.4) or a length scale based on a velocity autocorrelation as per (2.3) does not alter our main conclusions.

We seek to estimate the evolution of the relative large-eddy turnover time scales for the boundary layer–FST problem. The behaviour of our HIT lies somewhere between the two classical models (the evolution of the defined velocity and length scales for the HIT is shown later in figure 3). We note the large-eddy turnover time scale of the HIT evolves as $T_e = \mathcal{L}/\mathcal{U} \sim t$ for both the Saffman ($t^{2/5}/t^{-3/5} \sim t$) and Batchelor ($t^{2/7}/t^{-5/7} \sim t$) theories, meaning the following analysis is the same irrespective of the type of HIT exhibited. White (2006) (equation 6-70) offers simple empirical power-law relations for turbulent boundary layers forming over flat plates, such that we can write $\delta \sim t^{6/7}$ by using $X = U_\infty t$, that is, the boundary layer is scaled by an observer travelling with the free stream. Temporal development as $U_\tau \sim t^{-1/7}$ is consistent with a constant boundary layer spreading rate $(1/U_\tau)(d\delta/dt)$ (figure 8d). However, we note the relations of White (2006) suggest $U_\tau \sim t^{-1/14}$. The present problem makes use of boundary layers that have been ‘pre-grown’ to a certain thickness prior to HIT injection into the free stream. Thus their development in time is advanced with respect to that of the HIT by t_0 , the time at HIT injection into the free stream. Armed with indicative power-law relations for the velocity and large-eddy length scales pertaining to the HIT (forming our FST) and that of the boundary layer, we estimate the evolution of the relative large-eddy turnover time scales for our present problem at large t as a simple power law:

$$e \equiv \frac{T_\delta}{T_e} = \frac{\delta/U_\tau}{L_e^u/u_e'} \sim \frac{t^{6/7}/t^{-1/7}}{t} \sim \frac{t}{t} \sim \text{constant}. \quad (2.5)$$

Thus, for the estimated power-law evolution of our individual parameters, at large t , this ratio will tend to remain constant if the boundary layer and FST do not interact. The time evolution of the numerator is perhaps ‘not very accurate’ (White 2006); however, in this context it nonetheless permits an estimate of the relative evolution of the boundary layer with respect to the HIT. The exponent for the quiescent temporal boundary layer of Kozul *et al.* (2016) ranges $\approx [0.71, 0.73]$ (compared to $6/7 \approx 0.86$), and that for U_τ is found to be $\approx [-0.089, -0.083]$ (versus $-1/7 \approx 0.14$ or $-1/14 \approx -0.071$). The significance of the above estimate is that e approaches a constant at large t for non-interacting boundary layer and HIT flows. As we will show later, a ‘strong’ interaction occurs if this parameter is less than around 2 at the moment when the boundary layer is first exposed to the FST. This same quantity was interpreted as a relative fluctuating strain rate by Hancock & Bradshaw (1989) as mentioned in § 1. The aim of the present work is to argue the importance of e from the view of relative lifetimes in explaining potential boundary layer modification by FST. This is in addition to the better understood necessary minimum external turbulence level.

3. Simulation set-up

Hereafter, we refer to fluctuating velocities u , v and w in the x (streamwise), y (spanwise) and z (wall-normal) directions. The appropriate Reynolds decomposition for the temporally developing turbulent boundary layer is given by $u_i = \bar{u}(z, t)\delta_{i1} + u_i'(x, y, z, t)$, where $\overline{(\cdot)}$ indicates averaging in the homogeneous xy planes. Statistics throughout the present work are computed at instantaneous times (i.e. from single velocity and scalar fields) and corresponding instantaneous FST statistics (i.e. u'_{rms} and L_e^u) are quoted. This is in contrast to the time window averaging used for the quiescent boundary layer in Kozul *et al.* (2016). The simulations presented herein are all single realisations meaning only moderate statistical convergence is achieved.

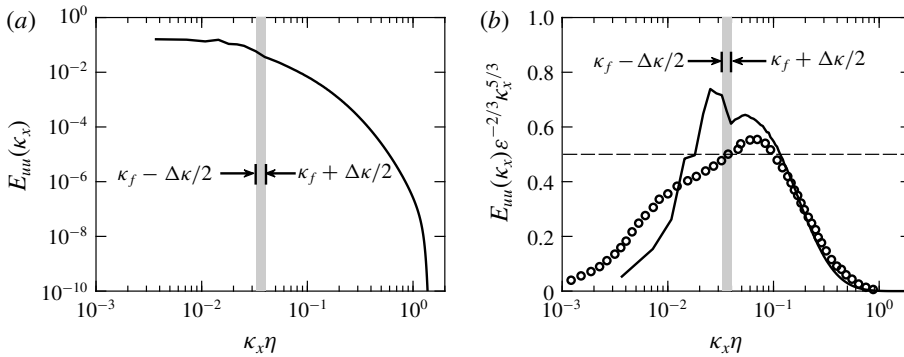


FIGURE 2. One-dimensional spectrum of the current FST cases both (a) uncompensated and (b) compensated: —, current HIT field used to form the FST at $t=0$ for all cases except A1 (table 1) with $Re_{\lambda,0}=82$; \bullet , $Re_{\lambda}=99$ case of Mydlarski & Warhaft (1996); ---, line at 0.5, the expected plateau value for the compensated spectrum within the scaling or inertial subrange region for high-Reynolds-number turbulence. Vertical grey band indicates the forced region in radial wavenumber range, keeping in mind that all $\kappa_x < \kappa_f$ are forced since the one-dimensional spectrum is aliased.

3.1. Generation of free-stream disturbances: HIT

The previously quiescent free stream of the turbulent temporal boundary layer is now seeded with HIT generated in a triply periodic domain in a precursor simulation using the spectral code of Chung & Matheou (2012) (shear turned off). A Fourier pseudospectral method (cf. Rogallo 1981) is used to integrate the Navier–Stokes equations, whose solution is advanced in time using the low-storage third-order Runge–Kutta scheme of Spalart, Moser & Rogers (1991). Quantities external to the boundary layer are identified with subscript e , and values at the beginning of the combined boundary layer–FST simulations with subscript 0. The cases will be characterised by a FST intensity u'_e/U_τ , where u'_e is the isotropic root-mean-squared velocity fluctuations of the HIT. The large-eddy length scale ratio is L_e''/δ .

Figure 2 shows both the uncompensated and the compensated streamwise-velocity one-dimensional spectra for the HIT field used to form the FST for all present simulations (except case A1). The observed peak is due to our forcing at a fixed shell of wavenumbers. Our HIT possesses only a limited region where the turbulence might be approximately inertial. Despite being modest, the present Taylor Reynolds numbers of the HIT still admit power-law decay of the kinetic energy. A time interval $\approx T_{e,0}$ is required before u'_e of the HIT begins this power-law decay.

Table 1 provides the main parameters for the precursor HIT simulations. A desired $L_{e,0}''$ in the FST is achieved via forcing to a selected shell of wavenumbers at constant power (similar to that in Carati, Ghosal & Moin (1995)), centred on forcing wavenumber κ_f . For the present HIT, $\kappa_f L_{e,0}'' \approx 5$ with forcing shell thickness $\Delta\kappa L_{e,0}'' \approx 1$. The ranges of relative length (L_e''/δ) and velocity (u'_e/U_τ) scale ratios are extended by injecting the HIT into the free stream of boundary layers that had been ‘pre-grown’ to different thicknesses δ , or equivalently, Reynolds numbers. The HIT kinetic energy decays according to established power laws as detailed above in §2 and care was taken to ensure the domain size did not constrict this behaviour. In simulations of decaying HIT, estimates of the integral length scale may become unreliable if it approaches a significant fraction the smallest domain dimension, primarily due to a

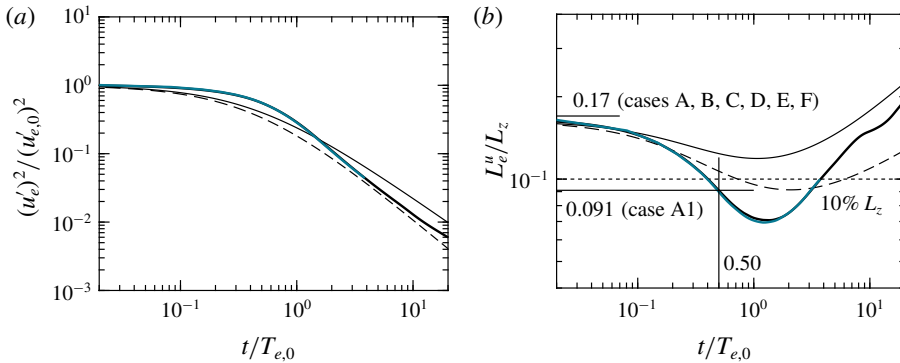


FIGURE 3. (a) Decaying turbulence velocity scale $\mathcal{U} \rightarrow u'_e$ with fits $(u'_e)^2 / (u'_{e,0})^2 = (1 + a_{u'_e} t / T_{e,0})^b$ following Krogstad & Davidson (2010): —, Saffman-type turbulence with $b = -6/5$; ---, Batchelor-type turbulence with $b = -10/7$; constant $a_{u'_e} = 2.3$ for both. (b) Growing length scale $\mathcal{L} \rightarrow L_e^u$ (2.4) formed from fits for u'_e as in (a) and $\varepsilon_e / \varepsilon_{e,0} = (1 + a_{\varepsilon_e} t / T_{e,0})^{b-1}$ for dissipation in the free-stream ε_e (not shown), constant $a_{\varepsilon_e} = 1.3$ for both, curves as for (a): — (teal), in the free stream of case D (table 2); —, decay in the box turbulence code. Dimension $L_z = L_y = L_x/2$ is the smallest box dimension for the simulations. Subscript e denotes quantities external to the boundary layer and subscript 0 values at the beginning of the combined boundary layer–FST simulations. Here $T_{e,0} = L_{e,0}^u / u'_{e,0}$ is the large-eddy turnover time scale of the forced statistically steady HIT.

lack of statistical averaging (Thorner 2016). The present simulations use an L_e^u that is maximally 17% of the smallest domain dimension at the time of insertion into the free stream, when it then decays for $t \approx T_{e,0}$, where $T_{e,0} = L_{e,0}^u / u'_{e,0}$ is the large-eddy turnover time scale of the forced steady-state HIT, before beginning power-law growth. Although figure 3(b) suggests this power-law growth is not seriously impeded up to $L_e^u \approx 0.2 L_z$, the simulations are conservatively halted when $L_e^u \approx 0.1 L_z$, following the observations of Thorner (2016). At the moment of injection into the free stream, the Taylor Reynolds number of the FST is $Re_{\lambda,0} = u'_{e,0} \lambda_{e,0} / \nu \approx 82$, for Taylor microscale λ , for all present cases except A1 (table 1), for which it is $Re_{\lambda,0} \approx 52$. Forcing to the HIT is removed at the moment of injection into the boundary layer's free stream such that the HIT fields begin decaying as the simulations with synthesised initial conditions are launched.

Case A1 is a companion simulation to case A: the HIT injected into the free stream of case A1 at $Re_\theta = 508$ is identical to the HIT in the free stream of case A (where the boundary layer is 'born' under FST) at that same Re_θ . Any difference between cases A and A1 is therefore due to their differing development histories. That is, HIT for case A1 is that for case A (and all others) yet allowed to decay (by removing the forcing) within the precursor HIT simulation for $0.50 T_{e,0}$ before injection, being the same interval of time required by the boundary layer of case A, exposed to the HIT from inception, to reach $Re_\theta \approx 500$. Hence all combined boundary layer–FST simulations (table 2) presented herein made use of only one forced HIT case. Case A1 is then formed by inserting the partially decayed HIT over a boundary layer formed under a quiescent free stream with $Re_\theta \approx 500$. This permitted investigation of the 'recovery' time required following the artificial combination of the fields (§ 3.2), that is, to gauge the difference between our cases formed from artificially synthesised fields and a boundary layer that has begun life under FST.

Case	N_x	N_y	N_z	$L_{e,0}^u/L_z$	$\lambda_{e,0}/L_z$	$Re_{\mathcal{L},0}$	$Re_{\lambda,0}$	$\kappa_{c,max}\eta_{e,0}$	L_x/L_z	L_y/L_z
A to F	1024	512	512	0.17	0.020	670	82	1.84	2	1
A1	1024	512	512	0.091	0.018	268	52	1.99	2	1

TABLE 1. Parameters for the precursor HIT simulations that formed the FST fields once inserted into the free stream of the cases listed in table 2. Physical quantities correspond to values at $t = 0$ (denoted with subscript 0) and external (subscript e) to the boundary layer in the simulations of table 2. Parameter $Re_{\mathcal{L}} = L_e^u u_e' / \nu$ is the turbulent Reynolds number of the HIT formed using the dissipation length scale L_e^u as the large-eddy length scale \mathcal{L} . Length scales are noted as a fraction of $L_z = L_y$ being the smallest and thus limiting domain dimension. Wavenumber $\kappa_{c,max}$ is the cutoff wavenumber for the present HIT simulations. Cases A to F are at steady state and forced until the moment of insertion into the free stream of the boundary layers. The HIT case for case A1 is simply that of the first row but allowed to decay for $0.50 T_{e,0}$ within the triply periodic box turbulence code by removing the forcing.

Case	Line	$Re_{\theta,0}$	$Re_{\tau,0}$	$L_{e,0}^u/\delta_0$	$u_{e,0}'/U_\infty$ (%)	$u_{e,0}'/U_{\tau,0}$	$T_{\delta,0}/T_{e,0}$	$T_{e,0}U_\infty^2/\nu$
A	— (royal blue)	47	—	20	5.0	—	—	275 284
A1	— (light blue)	508	221	1.7	3.6	0.71	0.41	202 477
B	— (crimson)	508	221	3.2	5.0	1.0	0.30	275 284
C	— (dark orange)	1413	508	0.54	10	2.3	4.2	68 821
D	— (teal)	506	220	1.6	10	1.9	1.2	68 821
E	— (magenta)	720	300	1.1	10	2.1	1.9	68 821
F	— (red)	1015	391	1.6	5.0	1.1	0.66	275 284

TABLE 2. Parameters of the present simulations of boundary layers developing under decaying FST. The turbulence intensity relative to the constant free-stream velocity is given by $Tu_0 \equiv u_{e,0}'/U_\infty$. Different values of $L_{e,0}^u/\delta_0$ are achieved by introducing the HIT into the free stream of a temporal boundary layer developing in a quiescent field at various $Re_\theta = U_\infty \theta / \nu$, with momentum thickness θ . A significant difference in intensities $u_{e,0}'/U_\infty$ was achieved by changing U_∞ by a factor of 2 (i.e. cases A, A1, B, F versus cases C, D, E). Here $T_\delta = \delta/U_\tau$ is the boundary layer large-eddy turnover time scale. Case A1 is a companion simulation to case A where we allow the HIT for case A1 to decay for $0.50 T_{e,0}$ before injection, being the same interval of time required by the boundary layer of case A, exposed to the HIT from inception, to reach $Re_\theta = 508$. Note the large difference in $T_{\delta,0}/T_{e,0} = e_0$ between cases C and D: the boundary layer was ‘pre-grown’ to a higher Reynolds number in case C before the FST was added. It therefore has a much larger large-eddy turnover time scale than case D, and also compared to that of the FST. The friction velocity $U_{\tau,0}$ for case A at FST injection (which is when the boundary layer also starts growing) is non-physical due to the numerical trip used. Moreover the relative large-eddy turnover time $e \equiv T_\delta/T_e$ is formed from scales that characterise the fully turbulent (i.e. inertial) boundary layer and the HIT, and thus is not here used to gauge interaction between a transitioning boundary layer ($Re_\theta < 500$ for the present temporal boundary layers) and HIT.

3.2. Combined simulations: the boundary layer is seeded with FST

The finite-difference code used for both the ‘pre-grown’ boundary layers and the synthesised fields for which statistics are presented herein has been validated in Kozul *et al.* (2016). The code employs the fully conservative fourth-order staggered finite-difference scheme of Verstappen & Veldman (2003) to spatially discretise the

Navier–Stokes equations, with the boundary conditions of Sanderse, Verstappen & Koren (2014). As for the precursor HIT simulations, the solution is marched forward in time using the low-storage third-order Runge–Kutta scheme of Spalart *et al.* (1991). The fractional-step method (e.g. Perot 1993) is used after each substep to project the velocity onto a divergence-free space, ensuring satisfaction of the continuity equation. Grid points are clustered near the wall using an error function stretching set by $z(\xi) = \text{erf}[a(\xi - 1)]/\text{erf}(a)$ for $a \approx 2$ and $\xi = [0, 1]$ (Pirozzoli, Bernardini & Orlandi 2016). The HIT and boundary layer velocity fields are combined via thresholding on the passive scalar c with Schmidt number $Sc = 1$, taking a value of C_w at the wall. It is here used as a proxy for the extent of boundary layer growth into the domain since c is initially at the free-stream (top boundary) value C_∞ everywhere. In contrast, the simulations of Rind & Castro (2012) and Pal & Sarkar (2015) embedded wakes in HIT based on criteria relating to the velocity field. The present approach is more akin to the experiments of Hancock & Bradshaw (1989), where the boundary layer developed over a slightly heated plate, allowing the wall-generated turbulence to be distinguished from the FST via an appropriate temperature threshold. The present simulations employ a passive scalar released at the wall for the same purpose, serving to ‘mark’ fluid originating in the boundary layer. Therefore we are able to assess the mixing of one flow (the turbulent boundary layer) with another (the HIT) by adopting a similar thresholding approach, rather than inferring the extent of mixing from the velocity or vorticity fields. We are thus able to attribute the turbulent fluid’s origin with some confidence, as opposed to relying on the velocity field, which is non-local due to the incompressible pressure condition. The present approach also eliminates the possibility of a bias towards any one component of velocity. Several recent studies have shown reliable demarcation of wall-generated turbulence from the free stream based on a passive scalar threshold (compared to one based on vorticity magnitude) both without (Watanabe, Zhang & Nagata 2018) and with (Wu, Wallace & Hickey 2019) FST. Using a threshold based on the kinetic energy was shown to incur the largest error in identifying the turbulent–non-turbulent interface in the study of Watanabe *et al.* (2018). The HIT is first interpolated using cubic splines onto the stretched grid required by the temporal boundary layer simulation. A function effectively masking the HIT by the turbulent boundary layer then gives the combined field $\mathbf{u}_0 = \alpha \mathbf{u}_{\text{HIT}} + (1 - \alpha) \mathbf{u}_{\text{TBL}}$ with

$$\alpha(\mathbf{x}) = \begin{cases} 0, & 0 \leq \frac{C_w - c}{C_w - C_\infty} \leq 0.95 \\ 1, & 0.95 < \frac{C_w - c}{C_w - C_\infty} \leq 1, \end{cases} \quad (3.1)$$

for $\mathbf{x} = (x, y, z)$ and $\mathbf{u} = (u, v, w)$, for scalar contrast $C_w - C_\infty$. Figure 4 shows a schematic of this field combination. All cases except case A are formed thus; for case A the HIT fields form the entire initial velocity fields (with a numerical trip imposed at the wall). Case A is thus analogous to most previous experimental studies of the present physical problem, where the boundary layer is exposed to FST from the beginning of its development. The scalar field is unchanged during the synthesis of the velocity fields (i.e. no fluctuations are added to the scalar field). The artificially synthesised (patched) initial fields are not divergence-free as required by the continuity equation; however, this is corrected after a single time step, when the numerical scheme employed projects the flow onto a divergence-free space. Physical quantities for the present cases are given in table 2. Importantly the final column notes

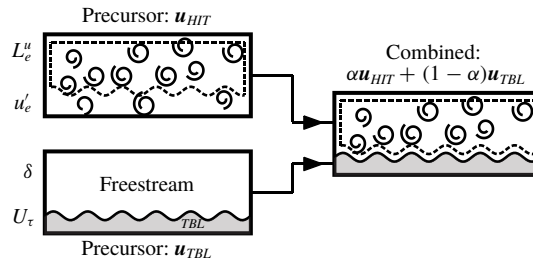


FIGURE 4. Schematic of the combined fields formed from precursor simulations via masking (3.1) using the scalar concentration of the boundary layer, represented by the grey shaded area (TBL, turbulent boundary layer).

$e_0 = T_{\delta,0}/T_{e,0} = (\delta/U_\tau)_0/(L_e^u/u_e')_0$, the initial relative large-eddy turnover time scale between the turbulent boundary layer and the FST. When the fields are combined, a decrease ($\approx 9\%$ for cases A1, B and F; $\approx 11\%$ for cases D and E; and $\approx 6\%$ for case C) in δ results at the first time step post-HIT injection; values of δ_0 (and therefore e_0 at $t = 0$) correspond to that before the HIT injection. No such change occurs in U_τ .

Periodic boundary conditions are imposed in the streamwise direction x as well as the spanwise direction y . A ‘conveyor-belt’ moving-wall set-up is used in the boundary layer simulations. At this bottom wall where $z = 0$, $u = U_w$ and $v = w = 0$ are imposed. The top boundary ($z = L_z$) is a fixed wall with an impermeable boundary condition on the normal velocity ($w = 0$) and slip boundary conditions on velocities tangential to the upper wall ($\partial u/\partial z = \partial v/\partial z = 0$). The familiar configuration, with a stationary no-slip wall and non-zero free-stream velocity $|U_\infty| = |U_w|$, is recovered via Galilean transformation. The resolution of non-spectral discretisation schemes is improved by use of a reference frame with zero mean bulk velocity (Bernardini *et al.* 2013). Therefore the present set-up with zero mean velocity in the free stream is the most advantageous choice for resolution of disturbances away from the wall where grid spacing is larger. An initial trip $Re_D \equiv DU_w/\nu \approx 500$, for trip height D , is used to trigger transition of the precursor boundary layer simulations to a turbulent regime as in Kozul *et al.* (2016). The pressure gradient is set to zero. We use a domain where $L_x = 2L_y = 2L_z$. The simulations can be run until one of the box constraints is met: either $L_e^u \approx L_z/10$ (equivalently $L_e^u \approx L_y/10$) (Thorner 2016) or $\delta \approx L_z/3$ (Schlatter & Örlü 2010). Grid details for the boundary layer–FST simulations are given in table 3.

4. Results

4.1. Visualisations of the FST–boundary layer interaction

As a first view of our simulations, figure 5 shows streamwise velocity fields overlaid with vorticity magnitude contours for case D of table 2, both at the beginning and end of the combined simulation. Figure 5(a) is at the moment when the free stream is seeded with HIT (where $Re_\theta = Re_{\theta,0} = 508$). Vorticity contours are drawn only for the boundary layer (before FST injection) for clarity. This corresponds to the ‘combined’ sketch of figure 4. The strong velocity fluctuations in the free stream have faded significantly in figure 5(b) at a later time (where $Re_\theta = 983$). Vorticity contours are drawn for the whole field at this later time. Figure 5(c) is the same as figure 5(b) but for a reference boundary layer developing under a quiescent free stream permitting a visual comparison.

Case	N_x	N_y	N_z	$L_x U_\infty/\nu$	$L_y U_\infty/\nu$	$L_z U_\infty/\nu$	Δx^+	Δy^+	Δz_1^+	Δz_t^+	$\kappa_{c,min}\eta_0$
A	1024	1024	1408	162 500	81 200	81 200	10.3	5.13	0.20	9.99	1.89
A1	1024	1024	1024	162 500	81 200	81 200	9.16	4.58	0.13	10.9	1.67
B	1024	1024	1024	162 500	81 200	81 200	9.16	4.58	0.13	10.9	1.55
C	1024	512	832	81 200	40 600	40 600	4.60	4.60	0.14	6.27	1.35
D	1024	512	832	81 200	40 600	40 600	4.60	4.60	0.14	6.27	1.35
E	1024	512	832	81 200	40 600	40 600	4.60	4.60	0.14	6.27	1.35
F	1024	1024	1024	162 500	81 200	81 200	9.16	4.58	0.13	10.9	1.55

TABLE 3. Grid details for the simulations of boundary layers developing under decaying FST. The precursor HIT simulations use a constant grid spacing in all three dimensions. For the combined boundary layer simulations, grid points are clustered near the bottom wall using an error function stretching $z(\xi) = \text{erf}[a(\xi - 1)]/\text{erf}(a)$ for $a \approx 2$ and $\xi = [0, 1]$ (Pirozzoli *et al.* 2016). Wavenumber $\kappa_{c,min} = \pi/\Delta z_t$ is the cutoff wavenumber for the largest vertical spacing in the simulation, at the top free-slip boundary, set such that $\kappa_{c,min}\eta_0$ is comparable to, or smaller than, $\kappa_{c,max}\eta_0$ in table 1 for the precursor HIT simulations. Note that $\kappa_{c,max}\eta_0$ in the boundary layer simulations is at the wall. Spacing Δz_1^+ denotes the maximum first grid spacing at the bottom wall, whereas Δz_t^+ is the maximum spacing at the top wall. Cited here are the coarsest grid spacings in wall units observed over the duration of the simulation. Note that cases A1, B and F, and then cases C, D and E use the same initial boundary layer configuration to which either different FST (for the A1 and B pair, case A1 using a partially decayed field) is inserted at the same time (equivalently, Re_θ , see table 2), or the same FST is inserted at different Re_θ (cases B and F have different $Re_{\theta,0}$ but the same FST; the same is true for cases C, D, E). Since the coarsest grid spacings are observed early in the simulation before FST is inserted (i.e. when the boundary layer is developing in a quiescent free stream), these values are identical for these two subsets of simulations.

Figure 6 shows a visualisation of boundary layers developing under both quiescent and turbulent free streams comparing experimental images from Hancock & Bradshaw (1989) to those from our simulations. The numerical images bear some striking similarities to those of the experiment. For all panels, $Re_\theta \approx 700$. At left are the experimental images, where figure 6(a) is of a boundary layer developing in a quiescent free stream and figure 6(c) is of a boundary layer under mild FST. At right are comparable images of the scalar for the numerical cases. Figure 6(b) is for a quiescent free-stream case (Kozul *et al.* 2016) and figure 6(d) is for the present FST case D. The large-eddy length scale ratio is matched between the experimental and numerical FST cases at $L_e^u/\delta = 0.4$, and the intensity differs marginally, being $u_e'/U_\infty = 0.03$ for the experimental case with free-stream velocity U_∞ and $u_e'/U_\infty = 0.04$ for the present case D. It is immediately obvious that the boundary layer with FST is much thicker at the same Re_θ in both the experimental and numerical images. In the quiescent case, we see rounded lobes at the edge of the more compact boundary layer, yet in the bottom images with FST, the edge of the boundary layer is far more jagged, emphasised with the thick white contour at 1% of the scalar contrast. It is clear from these images that one of the main actions of the FST is to, given the same momentum deficit, increase the spread of the boundary layer by transporting fluid mass away from the wall. This conclusion cannot be reached if vorticity or turbulent kinetic energy is used instead of the scalar (§ 3.2) to demarcate wall-generated turbulence from FST. Note the more subtle increase in the boundary layer thickness δ between figures 6(b) and 6(d). The scalar

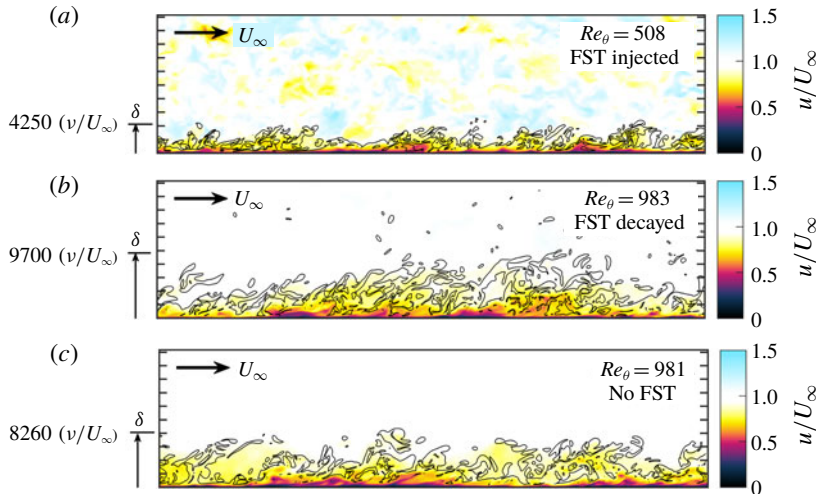


FIGURE 5. Indicative streamwise velocity fields overlaid with contours of vorticity for case D at two different times: (a) $t = 0$, at the moment when the FST is injected into the free stream ($Re_\theta = Re_{\theta,0} = 508$); (b) $t \approx 3.8 T_{e,0}$ after FST injection ($Re_\theta = 983$). (c) Reference case with no FST (Kozul *et al.* 2016) at a comparable Re_θ to (b). Vorticity contours in (a) are those of the boundary layer before FST injection, showing its ‘pre-grown’ extent. Black contour lines are drawn at $|\omega| = 1.4 U_\infty/\delta$ for all panels. For (a,b), actual vertical extent of the domain is twice that shown; full streamwise extent (L_x) shown. For (c) the numerical domain was larger such that the domain shown only represents $\approx (1/2) L_x$ and $\approx (1/4) L_z$ of the actual numerical domain. The streamwise and spanwise extents shown in all panels are equivalent in terms of ν/U_∞ ; tickmarks on the vertical axes show intervals of $2000 \nu/U_\infty$.

here demonstrates the significant redistribution of boundary layer fluid, including large excursions of the order of 2δ . Such enhanced excursions due to FST may be particularly relevant in the context of a reacting flow or pollutant originating in the boundary layer.

4.2. Progress of simulations

Figure 7(a) places our simulations amongst several previous experimental studies by plotting the evolution of the relative large-eddy length scale L_e^u/δ against the relative velocity scale u_e'/U_τ . The current cases begin at the top right of each curve and track downward and to the left in time as do those of Hancock & Bradshaw (1983) as their measurement location moves downstream in the wind tunnel. The curve for case A (initial simulation fields are HIT with no ‘pre-grown’ boundary layer) is only plotted from $Re_\theta \approx 500$ onwards such that the growing boundary layer is behaving canonically following a numerical trip at $Re_D = 500$ (Kozul *et al.* 2016). Here we are able compare companion cases A and A1. There is some discrepancy in their respective curves in figure 7(a) indicating the history effect. Even when we matched the ‘downstream’ condition at $Re_\theta = 508$, the difference in their curves indicates an integrated effect upon the boundary layer of case A since it was exposed to FST from the boundary layer’s inception. As a comparison to the dissipation-based scale L_e^u , the evolution of the longitudinal integral length scale $L_{uu,e}$ (2.3) is shown as a function of the relative

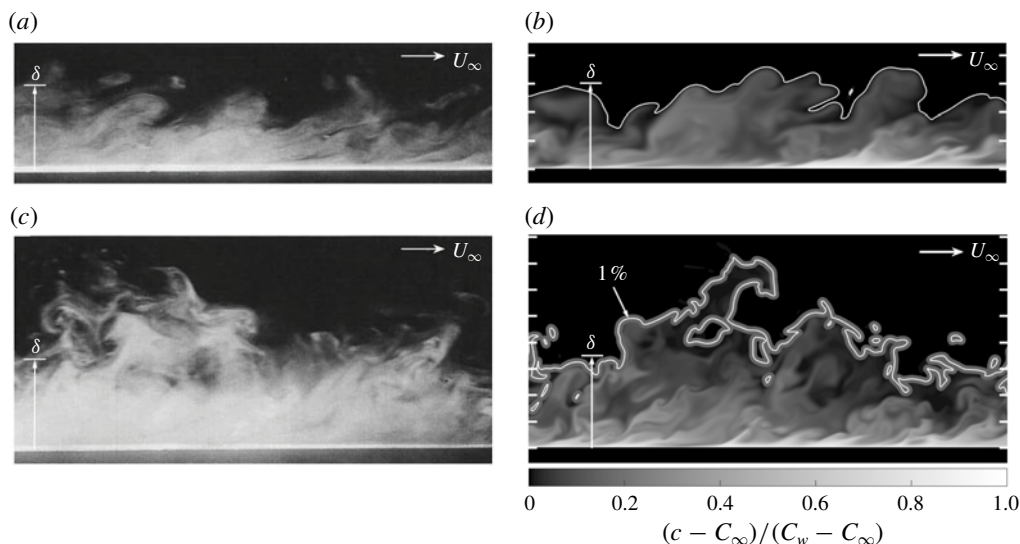


FIGURE 6. Flow visualisations. (a,c) Reproduction of figure 9 from Hancock & Bradshaw (1989) at $Re_\theta \approx 700$ (δ extracted from reproductions of these same images in figure 4.14 of Hancock (1980), these versions preferred for their higher quality): (a) $u'_e/U_\infty \approx 0.0$; (c) $u'_e/U_\infty \approx 0.03$, $L_e^u/\delta_{995} \approx 0.4$. (b,d) Scalar from temporal boundary layer simulations at $Re_\theta \approx 700$, $t \approx 1.5 T_{e,0}$ post-FST injection: (b) with a quiescent free stream from Kozul *et al.* (2016); (d) present case D with FST, $u'_e/U_\infty \approx 0.04$, $L_e^u/\delta \approx 0.4$. White contours in (b,d) show location of 1% scalar concentration; thick grey contour in (d) estimates the distance ℓ_D over which the scalar diffuses from the time of FST injection to the time shown here (cf. § 4.6). Assuming the same aspect ratio in (a,c), the streamwise extent shown for (b,d) is approximately equivalent in units of ν/U_∞ ; white tickmarks on the vertical axes of these panels show intervals of $2000 \nu/U_\infty$.

velocity scale u'_e/U_τ in figure 7(b). The value of this length scale is clearly smaller than that of L_e^u , especially at times just after the FST injection. However, a similar relative behaviour of the length scales is demonstrated for the different cases when this integral length scale is used. The unevenness of $L_{uu,e}/\delta$ towards the end of the simulation (low u'_e/U_τ) for cases D and E is a result of the limited domain size where it is likely less reliable.

Figure 8(a) shows that the boundary layer thickness δ increases in time over that for a boundary layer under a quiescent free stream for cases A1, B, D, E and F. The increase shown is compared to the boundary layer thickness at the time step prior to FST injection δ_0 . Dashed lines show the continued development of the boundary layer thickness, coloured by each case, when the FST is not injected. For cases A1 and B, this is the same precursor boundary layer simulation that was prolonged, these cases only differing in the HIT that was injected, and therefore the $T_{e,0}$ by which the horizontal axis is scaled. Case A, where the FST formed the initial velocity fields, is excluded here (and from figures 8b and 8c) since for that case δ_0 is the non-physical numerical trip making such a comparison physically meaningless. Also in case A the boundary layer is transitioning at early times, where the focus here is on the impact of FST on the fully turbulent boundary layer. Only a limited development extent could be viewed for case C, where the simulation was halted at the point shown, since δ reached our imposed simulation limit of $1/3$ of the box height L_z , at which point

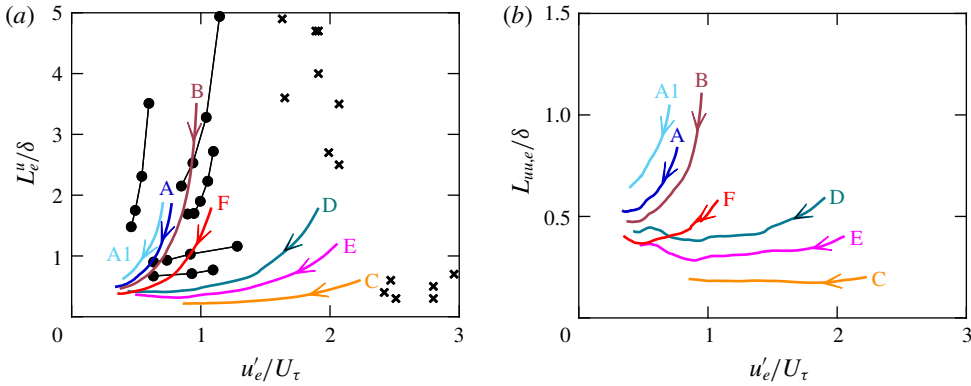


FIGURE 7. Regime diagram showing cases (coloured curves as per table 2) for two different definitions of the large-eddy length scale on the vertical axis. (a) Dissipation-based scale L_e^u from (2.4); (b) longitudinal integral length scale $L_{uu,e}$ (2.3) within the free stream: ●, Hancock & Bradshaw (1983); ×, Dogan *et al.* (2016). Curves track downward towards the left-bottom corner.

$\delta \equiv \delta_{99}$ becomes unreliable. Figure 8(b) shows that the development of momentum thickness θ is more or less unaffected by the FST for all cases.

Figure 8(c) shows that FST increases U_τ over that for the quiescent boundary layer for cases A1, B, D and F. Here the largest change is found for case D compared to the boundary layer allowed to continue growing under a quiescent free stream (dashed line), with smaller divergences from the quiescent curves for cases A1, B and F. Interestingly, case E, which displayed a significant increase in δ over the quiescent development value in figure 8(a), does not show a sustained gain in U_τ . Therefore for this case at least, the FST is mostly an outer-scale interaction working only to increase the boundary layer thickness. Generally, any increases in U_τ appear to be more short-lived: for case D the FST curve appears on its way to merge with the quiescent curve by $\approx 3t/T_{e,0}$ measured post-FST injection. The same cannot be said for the seemingly more permanently increased δ in figure 8(a). The gain in friction Reynolds number due to the FST can be directly calculated at any $t/T_{e,0}$ following FST injection from figure 8(a,c) using $[(Re_\tau)_{FST\ case} - (Re_\tau)_Q]/Re_{\tau,0} = [(U_\tau/U_{\tau,0})(\delta/\delta_0)]_{FST\ case} - [(U_\tau/U_{\tau,0})(\delta/\delta_0)]_Q$, where Q is for quiescent. Any such increase due to FST injection will be proportionally more attributable to an increase in δ rather than an increase in U_τ (the vertical scale of figure 8a showing $\delta/\delta_0 - 1$ is ten times that of figure 8c showing $U_\tau/U_{\tau,0} - 1$).

Figure 8(d) plots the boundary layer spreading rate $U_\infty^+(d\delta/dX)$ similarly as a function of time post-FST injection. For cases D and E we see large, short-lived spikes in the spreading rate above that for the boundary layer allowed to continue growing under a quiescent free stream. Case D with the smaller $e_0 = T_{\delta,0}/T_{e,0} = 1.2$ seems to react faster and exhibit a larger spreading rate than case E with $e_0 = 1.9$. From figure 8(a,c), we know the increased spreading rate for case E is due mostly to the increase in the rate of growth of δ in a more purely outer-flow interaction. Case F displays a more modest but still significant increase in the spreading rate at early times, especially at $\approx 0.5t/T_{e,0}$. The effect is much weaker for the other low-intensity cases A, A1 and B although a small increase is seen for case B over its corresponding quiescent case. The early elevated spreading rates of cases B, D

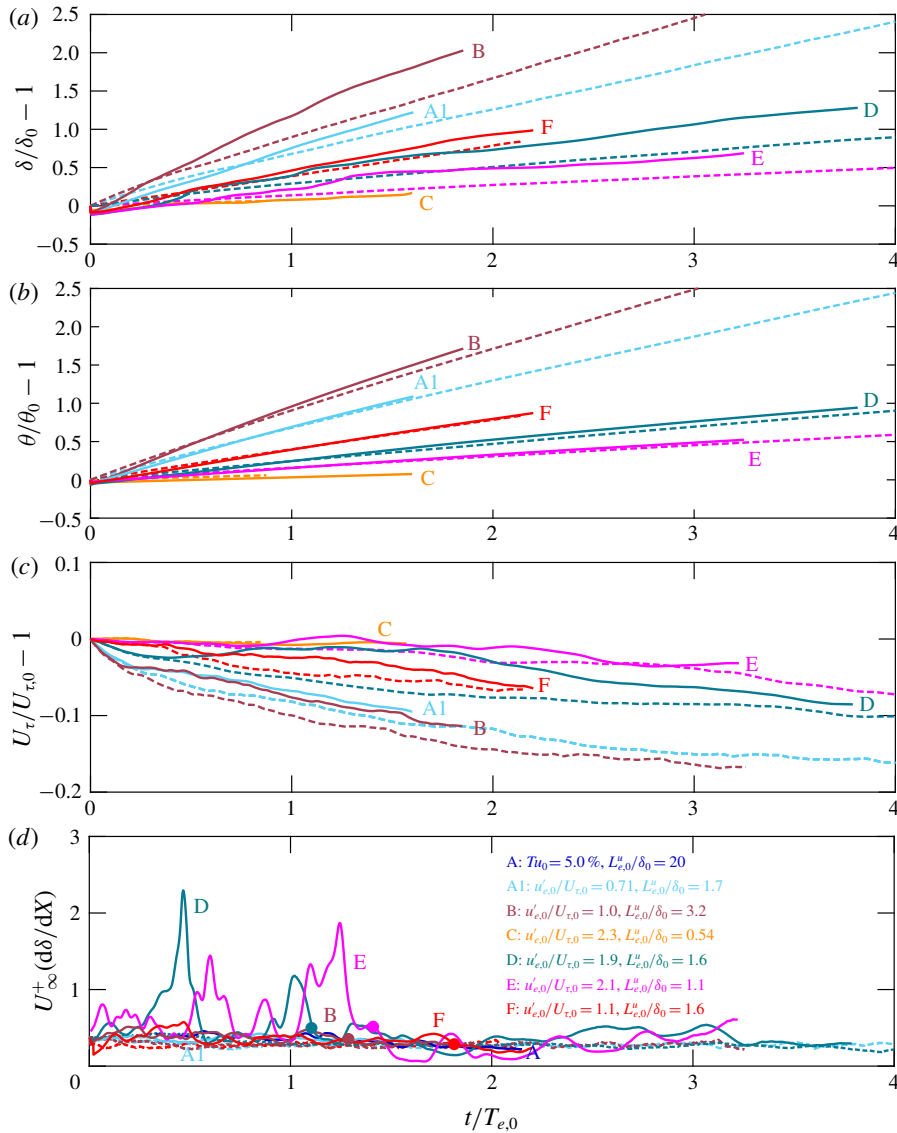


FIGURE 8. (a) Development of δ/δ_0 with time post-FST injection for all present cases except case A, curves coloured as per table 2: ---, similar for precursor ‘pre-grown’ boundary layers allowed to continue development without FST; δ_0 is the boundary layer thickness at the time step prior to FST injection. (b) Development of θ/θ_0 with time post-FST injection; θ_0 is the momentum thickness at the time step prior to FST injection. (c) Development of $U_\tau/U_{\tau,0}$ with time post-FST injection; $U_{\tau,0}$ is the friction velocity at the time step prior to FST injection. Note that the ‘pre-grown’ boundary layers are the same for cases A1 and B, yet the FST was allowed to decay sometime before injection to form case A1, meaning $T_{e,0}$ differs. (d) Boundary layer spreading rate as a function of time post-FST injection for all present cases except case C: ---, spreading rate for precursor ‘pre-grown’ boundary layers allowed to continue development without FST; ●, point at which each interaction has become ‘weak’, i.e. where spreading rate tempers to the quiescent value. Data for (d) are window-averaged over time intervals of $\approx 0.03 T_{e,0}$.

and E relax back towards quiescent values as the FST decays in time and these approximate points are indicated with filled circles. For temporal wakes embedded into HIT, Rind & Castro (2012) similarly found an increase in the growth rate of the wake half-width, with a stronger effect for a higher external turbulence intensity. The spreading rate for case C is not shown in figure 8(d) since it appeared unreliable.

The gain in the skin friction coefficient due to FST injection compared to the canonical values of the quiescent boundary layer $((C_f)_{FST\ case} - (C_f)_Q = \Delta C_f > 0$ for $C_f = 2(U_\tau/U_\infty)^2$) is shown in figure 9, and is considerably larger when computed at matched Re_τ than that at matched Re_θ . Increases in the skin friction coefficient due to FST have been reported previously (Hancock & Bradshaw 1983), as have increases in the mass transfer coefficient or Stanton number St (Blair 1983a). This gain is directly proportional to the gain $U_\tau/U_{\tau,0}$ shown in time post-FST injection in figure 8(c). Figure 8(a) shows the often significant change in δ for the different cases, meaning values along both axes are potentially altered in figure 9(a) in the case of boundary layers developing under FST. Since $(C_f)_Q$ decreases with increasing friction Reynolds number $Re_\tau = U_\tau \delta / \nu$, increasing Re_τ via an increase in δ , but not U_τ , will nonetheless give $\Delta C_f > 0$ with no change in $(C_f)_{FST\ case} = 2(U_\tau/U_\infty)^2_{FST\ case}$. Thus the development of C_f with respect to Re_τ may be altered via an ‘artificial’ increase in Re_τ , rather than a genuine change in the wall-defined value of C_f due to, say, the penetration of free-stream fluid. That is, a gain in C_f over that for the canonical quiescent boundary layer will be due at least in part to the enhanced growth rate of the boundary layer thickness δ at fixed $\Delta Re_{X=U_\infty t} = Re_{\Delta X=U_\infty \Delta t}$ (i.e. at a fixed time) post-HIT injection when compared to the quiescent case. The development of momentum thickness θ post-FST injection is much less affected than that of δ (figure 8a,b) and therefore $(\Delta C_f)_{=Re_\theta}$ (figure 9b, at matched Re_θ) will be almost solely due to a change in U_τ , if indeed there is one. Gain $(\Delta C_f)_{=Re_\tau}$ (figure 9a, at matched Re_τ) is larger due to the permanent shift in δ , irrespective of changes in U_τ . Case E, for example, shows a considerable and persistent increase in δ post-FST injection compared to the quiescent case, although no durable increase in U_τ . As a result, the peak ΔC_f for this case appears more than twice as large when plotting at matched Re_τ to the quiescent boundary layer (figure 9a) than at matched Re_θ (figure 9b). Case E is therefore particularly illustrative: it is important to correctly attribute the origin of an apparent gain ΔC_f for boundary layers developing under FST. The present temporal technique permits appraisal of these separate modifications to the boundary layer by the FST.

Furthermore, we point out the almost horizontal (albeit noisy) gain in the skin friction coefficient in figure 9 for case A, which had the HIT as the initial fields when the boundary layer simulation began, in contrast to all other cases where the FST was injected over a ‘pre-grown’ boundary layer. Where ΔC_f for the other cases tends to rise as the effect of the FST is felt by the boundary layer (i.e. via an increased growth in δ and decrease in U_τ), before decreasing at a later time (higher Reynolds number), the skin friction coefficient gain for case A appears approximately constant. Corresponding with this more ‘passive’ behaviour, case A shows no increased spreading rate in figure 8(d) for the same simulation period as shown in figure 9. The change in FST intensity and length scales for case A is very similar to its companion case A1 for the development extent shown in figure 9, yet ΔC_f shows a sharper increase for case A1 (although a lower peak ΔC_f) as the boundary layer responds to the FST. Case A’s behaviour here points to an upstream perturbation to the burgeoning boundary layer, from which full recovery back to quiescent boundary layer values (i.e. $\Delta C_f = 0$) remains elusive.

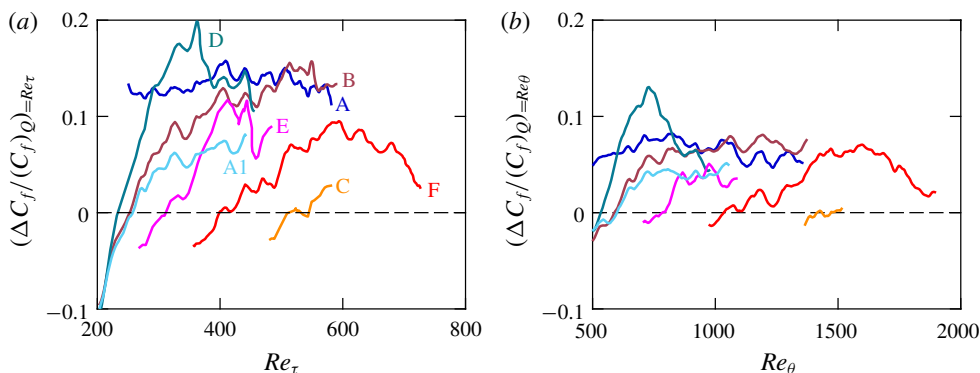


FIGURE 9. Gain in the skin friction coefficient due to FST injection with respect to the quiescent boundary layer of Kozul *et al.* (2016): (a) at matched Re_τ ; (b) at matched Re_θ . The curve for case A (initial simulation fields are HIT with no ‘pre-grown’ boundary layer) is plotted from $Re_\theta \approx 500$.

4.3. Comparisons of velocity statistics to experimental data

Notwithstanding a non-physical adjustment period after the artificial combination of fields, we are able to find some agreement with an experimental study of the same problem (Nagata *et al.* 2011) for case D in figure 10, showing subdued mean streamwise velocity and w_{rms}^+ profiles. The small-scale turbulence of Nagata *et al.* (2011) allows for quantitative comparison to the present cases, although the upstream history is different compared to our synthesised fields. Here Re_τ and u_e'/U_τ are roughly matched for the two cases, although the length scale ratio L_e^u/δ is larger for the case of Nagata *et al.* (2011) being 0.403, compared to 0.235 for the present case D at this point in the simulation. An important FST effect is the reduction of the boundary layer wake (referring to the region external to the logarithmic region) in the mean streamwise profile in figure 10(a): Blair (1983b) reported a totally subdued wake for a value of around $u_e'/U_\infty \approx 0.05$. For this level of turbulence at $u_e'/U_\tau = 0.50$, $u_e'/U_\infty \approx 0.02$, we find the wake weakened yet still present. The diminishing wake is a manifestation of diminishing intermittency in the outer region of the boundary layer due to the free-stream disturbances. The profile of u_{rms}^+ (not shown) is identical to the quiescent case at matched Re_τ from the wall up to $z^+ \approx 400$. Note the flat w_{rms}^+ profile in figure 10(b) away from the wall: being well outside the boundary layer, this is where the flow resembles HIT. Figure 10(c,d) shows a subdued Reynolds shear stress profile plotted against both inner and outer coordinates. The present DNS demonstrates the negligible effect on the Reynolds shear stress due to FST very close to the wall for $z^+ \lesssim 10$ in figure 10(c). The substantial depression of the Reynolds shear stress profile for $z^+ \gtrsim 10$ with respect to the quiescent case seems wholly attributable to the diminished w_{rms}^+ profile since a decrease with respect to the quiescent case is evident across the same region in figure 10(b). Figure 10(d) with a linear horizontal axis gives a more tangible view of the effect of the FST with respect to the large-eddy scale of the boundary layer δ .

Although we would not necessarily expect to find agreement with this experimental boundary layer, which has been exposed to FST from inception in contrast to ours, for our case D the FST has indeed effected change deep into the boundary layer. Our injected FST apparently affects our boundary layer in a similar way to that in

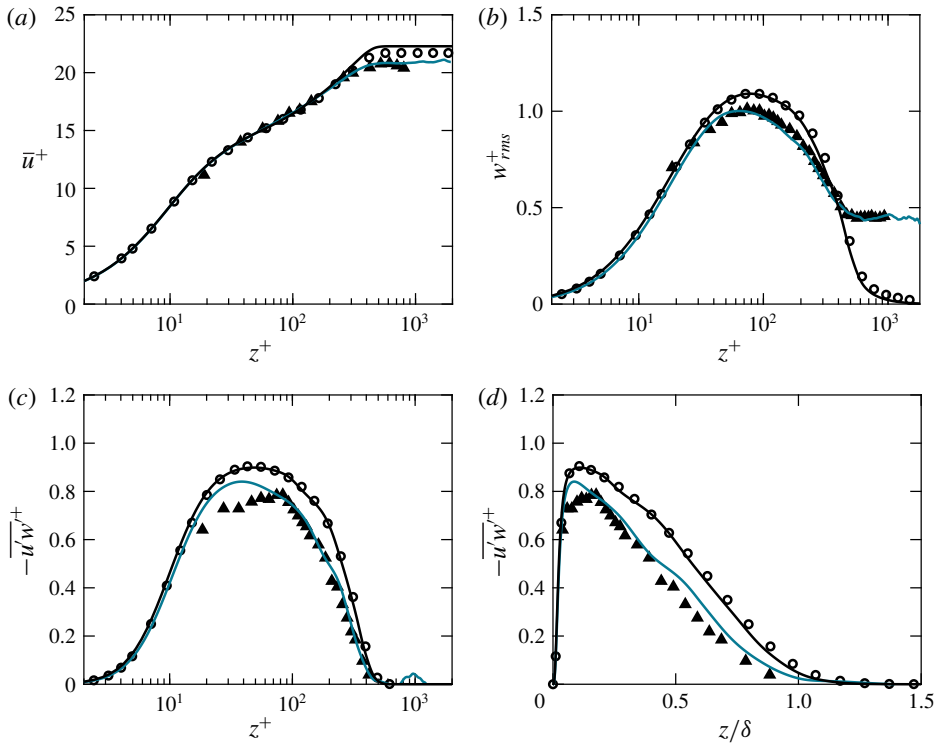


FIGURE 10. FST-altered profiles for case D at $Re_\tau \approx 450$ ($Re_\theta \approx 940$): (a) mean streamwise velocity; (b) root-mean-squared wall-normal velocity fluctuations. Reynolds shear stress profiles plotted against (c) inner-scaled wall-normal distance and (d) outer-scaled distance: — (teal), present case D, at $t/T_{e,0} \approx 3.4$ post-FST injection, with FST parameters $L_e^u/\delta = 0.403$, $u_e'/U_\tau = 0.43$; —, quiescent temporal boundary layer from Kozul *et al.* (2016) at matched $Re_\tau \approx 450$ ($Re_\theta \approx 1200$); \circ , quiescent spatial boundary layer of Simens *et al.* (2009) at $Re_\tau = 445$ ($Re_\theta = 1100$); \blacktriangle , experimental case LG-2 of Nagata *et al.* (2011) at $Re_\tau = 475$ ($Re_\theta = 1100$) with FST parameters $L_e^u/\delta = 0.235$, $u_e'/U_\tau = 0.45$.

this particular experimental case. We consider this an example of a ‘strong’ boundary layer–FST interaction. In contrast to these modified profiles for case D, similar profiles for case C showed no discernible change from the quiescent profiles due to the presence of the FST. We highlight the large difference in $T_{\delta,0}/T_{e,0}$ between cases C and D, which may explain why the boundary layer in case C appears to ignore the FST in an apparent ‘weak’ interaction with it.

4.4. Intermittency factor

The intermittency factor γ is shown in figure 11. We find that the effect of the FST is to spread the boundary layer edge significantly compared with the boundary layer under a quiescent free stream for all cases. The intermittency profile is here calculated as the fraction of points in a homogeneous xy plane above 1% of the scalar contrast $C_w - C_\infty$. Figure 11(a) corresponds to the time of FST injection, figure 11(b) after $t \approx 0.9 T_{e,0}$ and figure 11(c) after $t \approx 1.7 T_{e,0}$. That is, curves are plotted at roughly the same time intervals in terms of the initial FST large-eddy time

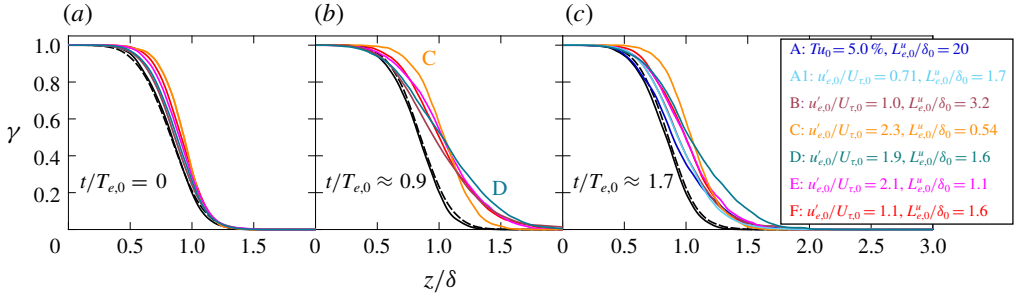


FIGURE 11. Intermittency factor calculated as the fraction of points at each homogeneous xy plane where the scalar exceeds 1 % of the scalar contrast $C_w - C_\infty$: (a) at the time of FST injection (not relevant for case A); (b) at $t \approx 0.9 T_{e,0}$ post-FST injection; (c) at $t \approx 1.7 T_{e,0}$ post-FST injection. Quiescent temporal boundary layer from Kozul *et al.* (2016) at matched Re_θ to —, case C in each panel; and ---, case D in each panel, together spanning the range of Re_θ covered by the current simulations. Coloured curves as per table 2.

scale for all cases. This definition of the intermittency is directly inspired by the approach of Hancock & Bradshaw (1989), where γ was defined as the fraction of total time at a measurement location for which the flow in their wind tunnel was ‘hot’ (based primarily on the detection of an increase in the measured temperature to a specified level above the background ‘cold’ level) in their heated boundary layer. This is a more quantitative representation of the smeared-out boundary layers shown in figure 6. Drawing attention to two cases in particular, we note the initial FST intensity $u'_{e,0}/U_{\tau,0}$ is similar for cases C and D. Scaling with δ along the horizontal axis eases the comparison between these and other cases: the value of δ at the time of FST injection differs by a factor of almost 3 between cases C and D for example. For all cases except case C, there is a flattening of the γ profile at the later times at $z/\delta \approx 1.5$, this effect being most marked for case D. The effect is weakest for case C, which has a high initial relative large-eddy turnover time scale $e_0 = T_{\delta,0}/T_{e,0}$; curves for case C in figure 11(b,c) appear offset from that of the quiescent boundary layer rather than smeared out. The curve for case D in figure 11(b) sits noticeably higher than that for case C over the region $1 \lesssim z/\delta \lesssim 2$. Thus the smearing effect appears to be strongest for case D, which has a lower e_0 than case C, and persists for the longest time (i.e. most flattened in figure 11c).

4.5. Wall-normal velocity variance profiles in time

In figure 10, the profiles for case D are plotted at a single Reynolds number to best match that of the experimental case to which it is being compared. In our temporally developing flow, this corresponds to one point in time following (‘downstream of’) the FST injection. A major strength of the present technique is the possibility of viewing the entire boundary layer–FST interaction as it unfolds.

Figure 12 plots the profiles at multiple points in time for all the present cases showing different responses to the injected FST. Figure 12(a–c) shows the three cases with lower FST intensity at injection (cases A, A1 and B) and figure 12(d–f) shows those at higher FST intensity (cases D, E and C). Curves in grey show profiles for the quiescent boundary layer of Kozul *et al.* (2016) matched to the Re_θ and Re_τ of the final curves in each panel. Insets in each panel show the evolution of the peak

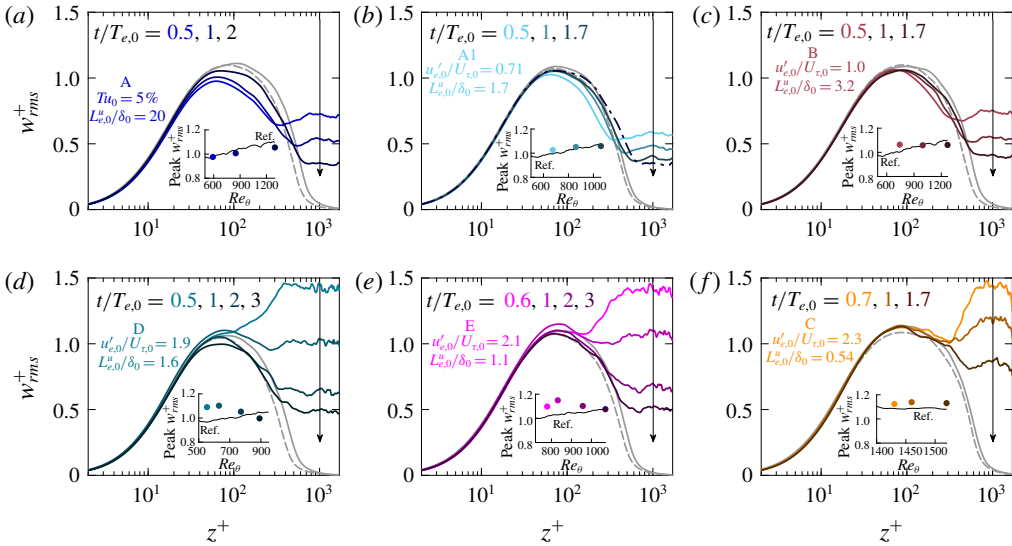


FIGURE 12. Wall-normal velocity variance evolution in time for all present cases from table 2; curves darkened as time (measured post-FST injection) progresses as indicated by the arrows. Top row shows cases with lower-intensity FST at injection with (a) case A, $u'_{e,0}/U_\infty = 5.0$; (b) case A1, $u'_{e,0}/U_\infty = 3.6$; and (c) case B, $u'_{e,0}/U_\infty = 5.0$. Bottom row for high-intensity cases, all with $u'_{e,0}/U_\infty = 10$: (d) case D; (e) case E; and (f) case C. Curves in grey for temporal boundary layer developing under a quiescent free stream from Kozul *et al.* (2016), with matched Reynolds number to the final curve shown in each panel: --- (grey), at matched Re_θ ; — (grey), at matched Re_τ . Inset in each panel shows the peak w_{rms}^+ value (for $z^+ < 100$) for each of the times $t/T_{e,0}$ post-FST injection shown, and compares it to that for the reference quiescent boundary layer (indicated by ‘Ref.’) at matched Re_θ . In (b): ----, case A at $t/T_{e,0} = 2$ for comparison.

w_{rms}^+ (for $z^+ < 100$) at the same $t/T_{e,0}$ for which profiles are plotted and compares them to that for the quiescent boundary layer at matched Re_θ (since figure 8 showed that the development of δ , which will alter the development of Re_τ , was much more affected than θ). For example, the second coloured circle marker on all insets in this figure shows the peak w_{rms}^+ at $t/T_{e,0} = 1$ post-FST injection compared with that of the quiescent boundary layer at matched Re_θ .

In figure 12(a–c) (lower FST intensity), small decreases from the quiescent profiles at the final times are evident. Indeed, only very subtle differences in the peak w_{rms}^+ are evident during the length of the simulations (see insets). Ingested FST fluid nonetheless lowers the final peak w_{rms}^+ slightly for cases A and B. Differences with respect to the quiescent case are smallest for case A1, where the injected FST is the partially decayed field used for cases A and B. The curve at $t/T_{e,0}$ for case A was added to figure 12(b) for comparison to its companion case A1 at $t/T_{e,0} = 1.7$. The curves are not exactly the same, indicating the history affect, yet this comparison (along with the agreement of case D with experimental data in figure 10) demonstrates the present technique’s ability to give ‘physical’ downstream fields. In figure 12(d–f), the three cases have the same $Tu \equiv u'_{e,0}/U_\infty$ (and similar $u'_{e,0}/U_\tau$) when the FST is injected, yet their relative length scales differ significantly: at injection $L'_e/\delta \approx 1.6$ for case D, ≈ 1.1 for case E and ≈ 0.54 for case C. We find that despite being exposed to the same levels of FST intensity, the outcome is different for the high-intensity cases,

with the FST in case D able to, after some time, effect change in the w_{rms}^+ profile with respect to a quiescent spatial DNS profile at a matched Re_τ (subdued peak for $z^+ \gtrsim 40$ after $t \approx 3 T_{e,0}$ in figure 12d). Only a mild difference from the quiescent case is found in the curve at the final $t/T_{e,0} = 3$ shown for case E (figure 12e); however, the evolution of the peak w_{rms}^+ is similar to that of case D, showing an early increase with respect to the reference quiescent case followed by a decline as the FST decays. We suggest the observed lowering of the final peak w_{rms}^+ as compared to the reference quiescent cases is due to the intensity of the FST being lower than the quiescent peaks at this point. At early times when the FST is stronger, the peaks for cases B, D and E in particular appear higher than that for the quiescent boundary layer, reflecting an early FST intensity level that exceeds the peak w_{rms}^+ of the reference quiescent boundary layer.

There is a key difference between cases D and E: at $t/T_{e,0} = 3$, we suggest that there have been more substantial FST fluid incursions deep into the boundary layer in case D than in case E, resulting in a lowering of the final peak value. The final circle marker in the inset of figure 12(d) for case D lies below that for the quiescent boundary layer, whereas the final circle marker for case E in figure 12(e) lies directly on the black curve, meaning the peak w_{rms}^+ is unaltered due to the FST at this later time. This will be further explored in § 4.6. A similar conclusion (i.e. final peak w_{rms}^+ lower than that for the reference quiescent case) is demonstrated by the low-intensity cases in figure 12(a–c); however, the effect is more subtle since the starting FST is weaker. A conclusion on case C is precluded by the limited timespan available; however, we note that changes in the profiles of other cases were effected after similar time delays of $t \approx 1.7 T_{e,0}$. In figure 12, time intervals of $t/T_{e,0}$ are equivalent for the evolution of HIT in all cases except A1, whose free stream was seeded with a partially decayed HIT field possessing a different $T_{e,0}$ (table 1). Yet, time passed in terms of the initial boundary layer large-eddy time scale $T_{\delta,0}$ is very different across the different cases, by a factor equivalent to the difference in $e_0 = T_{\delta,0}/T_{e,0}$ shown in table 2. Thus the boundary layer time scale for case C is roughly three times that for case D, and we suggest that this parameter effectively dictates the ‘reaction’ time of the boundary layer to the FST.

Figure 12 suggests that the boundary layer–FST interaction in our cases can be largely understood as a simple mixing of the two flows. The peaks in the w_{rms}^+ profiles (insets) are increased when the FST is higher than the peak of the wall-generated turbulence, and then decreased when the FST is lower than that of the wall-generated turbulence. Section 4.6 offers conditional statistics building on this basic idea. For wakes subject to external turbulence, Pal & Sarkar (2015) introduced an argument based on u'_e/u'_{cl} , where u'_{cl} is the wake turbulence at the centreline, and found this to be the key parameter governing the influence of external fluctuations, and not u'_e/U_0 , where U_0 is the centreline deficit velocity. In the present wall-bounded flow we can take u'_{cl} to be analogous to the peak in velocity variance profiles (i.e. $w'_{rms,peak}$ in the insets of figure 12). We indeed see the influence of the relative ratio of $w'_e/w'_{rms,peak}$ as stated above. Pal & Sarkar (2015) reported that the external turbulence and centreline turbulence tend to eventually decay at similar rates. This is in contrast to the boundary layer where the inner peak is ever energised by U_τ , and thus the ratio $w'_e/w'_{rms,peak}$ always falls as the FST decays. Whether the boundary layer is impacted by the FST while $w'_e/w'_{rms,peak}$ is still above unity (Pal & Sarkar 2015) is governed by the relative large-eddy turnover time scale. This effectively dictates the ‘adjustment time’ required between the boundary layer and FST.

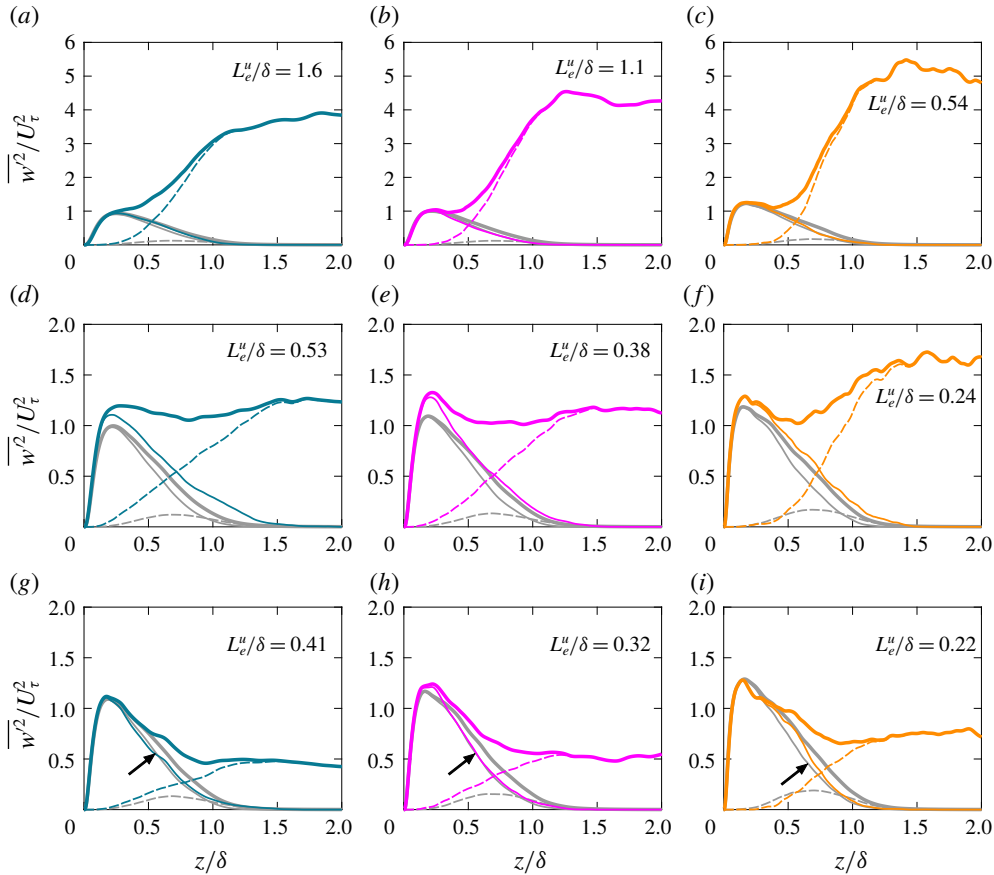


FIGURE 13. Boundary layer and FST contributions to the wall-normal Reynolds stress profiles deduced via conditional averaging on the passive scalar: —, contribution from the boundary layer; ---, contribution of FST; —, sum of two profiles, equivalent to non-conditionally averaged profiles. (a,d,g) Case D; (b,e,h) case E; (c,f,i) case C. (a–c) At the time of FST injection; (d–f) at $t \approx 0.9 T_{e,0}$ after FST injection; (g–i) at $t \approx 1.7 T_{e,0}$ after FST injection. Curves in grey are for the boundary layer developing under a quiescent free stream from Kozul *et al.* (2016) at matched Re_τ for each panel. Note that the vertical scale in (a–c) differs from that of the subsequent panels.

4.6. Free-stream contributions to wall-normal Reynolds stress profiles

We seek to further dissect the profiles shown in figure 12 for the high-intensity cases. This analysis follows that of Hancock & Bradshaw (1989) where the turbulent free-stream fluid was distinguished from that of the wall-generated turbulent flow by heating the boundary layer. In the present case, the passive scalar released at the wall will ‘mark’ that fluid as originating in the boundary layer. Figure 13 shows the ‘free-stream’ versus ‘boundary-layer’ contributions to the wall-normal Reynolds stress profile $\overline{w^2}/U_\tau^2$ for present cases C, D and E. We here use the same thresholding for the conditional averages as used for the injection of the FST as per (3.1): if the scalar concentration is below 5% of the scalar wall contrast $C_w - C_\infty$, it will be considered to be a free-stream contribution, else it is deemed to be wall-generated turbulence for the purposes of the present analysis. We note a 1% threshold was

used in the flow visualisation shown in figure 6 and for calculating the intermittency profiles of figure 11. This was both to emphasise differences between the quiescent and FST cases, and since δ is based on the 99 % thickness throughout this work. In contrast a 5 % threshold is more appropriate for figure 13 which directly considers ‘free-stream’ or ‘boundary-layer’ fluid, following the synthesis of these fields via 5 % thresholding. The quiescent boundary layer of Kozul *et al.* (2016) is subjected to the same analysis and curves are shown at matched Re_τ in each of the panels for reference. However, it should be noted that our passive scalar is diffusive, with $Sc = \nu/\mathcal{D} = 1$, for \mathcal{D} diffusivity of the scalar. At the latest time shown in figure 13, $t \approx 1.7 T_{e,0}$ following FST injection, the scalar will diffuse over a length $\ell_{\mathcal{D}} \approx \sqrt{2\mathcal{D}t} = \sqrt{2(\nu/Sc)1.7 T_{e,0}} \approx 1.8\sqrt{T_{e,0} U_\infty^2/\nu} (\nu/U_\infty) \lesssim 500 \nu/U_\infty$ (Westerweel *et al.* 2009), where $T_{e,0} U_\infty^2/\nu$ is noted in table 2 for all the present cases. This corresponds to a quarter of one of the tickmark intervals shown in figures 5 and 6. As a fraction of the temporal mean of the boundary layer thickness $\bar{\delta}$ over $t = [0-1.7] T_{e,0}$ post-FST injection, $\ell_{\mathcal{D}} \lesssim 7\%$ of the smallest $\bar{\delta}$ (case D). The error incurred in considering the scalar as a ‘marker’ of the wall-generated turbulence does not impede the analysis, since changes potentially due to diffusion in the curves conditioned on the scalar in figure 13 are small compared to the horizontal extent shown (2δ). Moreover, it is small compared to the 50–200 % growth in δ compared to δ_0 by this time post-FST injection (figure 8a), due in large part to the increasing Reynolds number irrespective of FST.

Figure 13(a–c) shows the penetration depth of the FST to be similar for the three cases at the time of FST injection. The picture changes at a later time. In figure 13(d) for case D and figure 13(e) for case E, after about $t \approx 0.9 T_{e,0}$ (again, for roughly the same time interval in terms of the initial FST large-eddy time scale for all cases), there is significant incursion of free-stream fluid through the boundary layer towards the wall until $z/\delta \approx 0.1$ after which it is negligible. However, the contribution to the total profile by free-stream fluid is almost zero below $z/\delta \approx 0.4$ for case C in figure 13(f). The same is true for case C at a later delay of $t \approx 1.7 T_{e,0}$ following FST injection. Note that the time delays post-FST injection considered here are the same as those for the intermittency profiles in figure 11. Not only does FST fluid apparently penetrate closer to the wall in cases D and E, the FST contribution profiles (i.e. dashed lines) straighten more quickly for these cases, especially in figure 13(d) for case D. At the time of injection for case D in figure 13(a), the profile of the FST contribution was concave and thus dropping off at a faster rate as one moves towards the wall. In contrast, the free-stream contribution for case C in figure 13(f) after the same time delay remains rather more curved similar to how it appears at the time of FST injection in figure 13(c), pointing to a smaller quantity of penetrating free-stream fluid. However, the dashed line appears straighter at a later time still in figure 13(i). Such an observation suggests differing adjustment times of the respective boundary layers to the FST.

Our profiles show marked qualitative similarity to those of Hancock & Bradshaw (1989) (their figure 13) given a delay after FST injection, which helps to bolster confidence in the ability of our artificially synthesised fields to replicate not only mean statistics of laboratory fields, but also the relative contributions of the free-stream and wall-generated turbulence. As they stated, we too see a substantially altered structure in the outer layer, while the main effect in the inner layer is increased ‘inactive’ motions (in the Townsend (1961) sense, that is, not significantly contributing to the shear stress $-\overline{u'w'}$) from the free stream. They showed two cases with roughly the same intensity $u'_e/U_\infty \approx 0.04$ yet different length scales, with one case at $L'_e/\delta = 1.90$

and the other with $L_e^u/\delta = 0.71$. They too observed deeper incursions of free-stream fluid at the larger length scale ratio. Since the present analysis gives results very similar to theirs, it would seem that, whilst $e = (u_e'/U_\tau)/(L_e^u/\delta)$ might influence how quickly FST is able to penetrate, the final form of the profiles is ultimately more influenced by the free-stream value of $\overline{w'^2}/U_\tau^2$.

At the later time $t \approx 1.7 T_{e,0}$ following FST injection (figure 13*g-i*), the ‘boundary-layer’ contributions from both the present FST cases and the quiescent boundary layer align rather convincingly (see arrows in figure 13*g-i*). Thus the $\overline{w'^2}/U_\tau^2$ profiles appear to approximately correspond to a superposition of that for the quiescent boundary layer and that for the FST with weights approximately set by the intermittency γ following a sufficient time delay. We suggest this explains the resulting profiles for the present work: incursions of free-stream fluid will tend to bring the outer Reynolds stress profiles closer to the value of the FST in the free stream. However, there is some disparity between the curves for the ‘boundary-layer’ contributions between the current cases and the boundary layer developing under a quiescent free stream at the earlier $t \approx 0.9 T_{e,0}$ following FST injection in figure 13(*d-f*) (i.e. difference between lines denoting boundary layer fluid when comparing the quiescent and FST cases in each panel). This could be due to the large growth in δ at this early time, meaning that matching $Re_\tau = \delta U_\tau/\nu$ of current cases with those of Kozul *et al.* (2016) may be problematic. The persisting discrepancy in the ‘boundary-layer’ fluid curves for case C (especially near $z/\delta \approx 0.5$ in figure 13*i*) is partly attributable to a small domain size yielding unconverged statistics. At this point in case C we reach the physical limit of the simulation where the boundary layer has grown to 1/3 of the domain height L_z .

4.7. Total turbulent kinetic energy budgets

Components of the energy budget for the total turbulent kinetic energy $k = \overline{u_i' u_i'}/2$ are plotted in figure 14 for cases D, E and C at $t \approx 1.7 T_{e,0}$ post-FST injection, the same final time as in figures 11 and 13. Following the example of Hoyas & Jiménez (2008), the individual terms in the k budget are premultiplied by the distance from the wall z such that, with the logarithmic abscissa, areas under the curves are proportional to the integrated energy. For the temporal boundary layer, the budget equation for k reads (Hoyas & Jiménez 2008; You & Zaki 2019)

$$\mathcal{B}_k \equiv - \underbrace{\frac{\overline{Dk}}{Dt}}_{\mathcal{A}} - \underbrace{\overline{u_i' u_j'} \frac{\partial \overline{u_i}}{\partial x_j}}_{\mathcal{P}} - \underbrace{\nu \frac{\partial \overline{u_i' u_j'}}{\partial x_j} \frac{\partial \overline{u_i'}}{\partial x_j}}_{\mathcal{E}} - \underbrace{\frac{1}{2} \frac{\partial \overline{u_j' u_i' u_i'}}{\partial x_j}}_{\mathcal{T}} + \underbrace{\overline{p' \frac{\partial u_i'}{\partial x_i}}}_{\Pi^s} - \underbrace{\frac{\partial \overline{u_i' p'}}{\partial x_i}}_{\Pi^d} + \underbrace{\nu \frac{\partial^2 k}{\partial x_j \partial x_j}}_{\mathcal{V}},$$

where \mathcal{P} is referred to as the production, \mathcal{E} the turbulent dissipation rate, \mathcal{T} the turbulent diffusion, Π^s the pressure strain, which vanishes in the k budget for an incompressible flow, Π^d the pressure diffusion, \mathcal{V} the viscous diffusion and $\mathcal{A} = \overline{Dk}/Dt = \partial k/\partial t + \overline{u_i} \partial k/\partial x_i$ (Pope 2000) is the mean flow material derivative of k , where only the unsteady part is retained for the temporal boundary layer.

Despite a lack of statistical convergence for curves of the budget terms in figure 14, deduced from single fields at a specific time, we are nonetheless able to assess how the FST has broadly changed, or not changed, the various components of the budget for the different cases throughout the extent of the boundary layer. Similar curves for the quiescent boundary layer (Kozul *et al.* 2016) at matched Re_τ are plotted in figure 14 for comparison. The present simulations are shown to be numerically very well converged since the budget residual (red line) is almost zero in all of the cases.

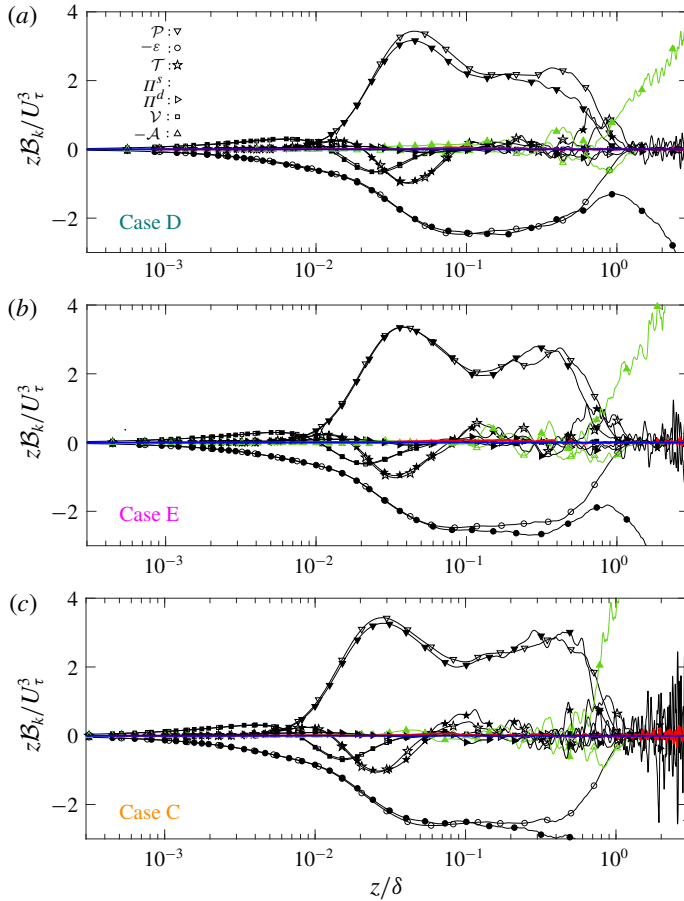


FIGURE 14. Total kinetic energy budget terms for (a) case D, (b) case E and (c) case C, all at $t \approx 1.7 T_{e,0}$ after FST injection. Filled symbols, present FST cases; open symbols, quiescent boundary layer of Kozul *et al.* (2016) at matched Re_τ ; — (red), budget residual; — (blue), line at zero for reference; \mathcal{P} , production; ϵ , dissipation; \mathcal{T} , turbulent diffusion; Π^s , pressure strain; Π^d , pressure diffusion; \mathcal{V} , viscous diffusion; \mathcal{A} , unsteady term (highlighted in green).

It is immediately clear that changes to the boundary layer due to FST injection are restricted to the outer region. Beyond $z \approx \delta$, in the absence of production, the turbulence is simply decaying and therefore balanced only by the unsteady term (highlighted in green). This is similar to the balance for the wake embedded in HIT of Pal & Sarkar (2015) at late times when the production term had decayed. The transport term follows that of the quiescent boundary layer and is negative up until $z/\delta \approx 0.08$, meaning the FST does not change the boundary layer's transport of kinetic energy outward from the wall over this region. This is also similar to that found by Pal & Sarkar (2015), where their FST was similarly not strong enough to overcome the transport of turbulence from the wake core outward. Case D shows a consistent decrease of its production term over $z/\delta \approx 0.03$ –1, whereas the effect is less conclusive for cases E and C.

An increase of the unsteady term $-\mathcal{A}$ within the boundary layer may indicate the penetration of decaying FST into it. This term is close to zero throughout the boundary layer in the quiescent case shown for reference, tends to be slightly negative at the edge of it and then zero in the quiescent free stream. For the present FST cases, this term shows a positive shift at the edge of the boundary layer for all the cases, consistent with the large positive value of $-\mathcal{A}$ in the free stream (since the curves are shown at the same time post-FST injection, the unsteady term \mathcal{A} has the same value relative to U_∞^3 for all cases shown in figure 14; however, the scale U_τ differs considerably between cases D, E and C due to their being at different Re_τ). Despite some subtle changes in the outer part of the boundary layer, figure 14 suggests the inner region is largely immune to the effects of the injected FST; the kinetic energy budget close to the wall remains similar to that of the quiescent boundary layer at matched Re_τ . This aligns with previous results herein indicating the FST–boundary layer interaction to be largely restricted to the outer region of the boundary layer.

5. When can the FST change the boundary layer?

The present work attempts to elucidate if and how the fully turbulent boundary layer is changed by free-stream disturbances by observing its development under decaying HIT injected into its free stream. In conducting simulations of boundary layers exposed to FST, we found examples of both ‘strong’ (e.g. case D) and ‘weak’ (e.g. case C) interactions. The ‘weak’ interaction of case C did not show the characteristic suppression of the wake in the mean streamwise velocity profile (Blair 1983*b*; Thole & Bogard 1996), evident in all other cases, although firm conclusions are precluded by the limited timespan available for that simulation. Case C also displayed less flattening of its intermittency profile, defined on the passive scalar released at the wall, than the other cases. This is despite high FST intensity u'_e/U_τ at injection. That is, the strength, or result, of the interaction cannot be solely predicted from knowledge of the FST intensity (or relative large-eddy length scale ratio L_e^u/δ) in isolation. Doing so may lead to inconsistencies in reported changes to Reynolds stress profiles as suggested from the summary of experimental results of this problem presented in Nagata *et al.* (2011).

Flattened intermittency profiles were observed for all other cases thus increasing δ , the 99% boundary layer thickness. The present technique of ‘marking’ boundary layer fluid with a passive scalar reveals large excursions of boundary layer fluid of the order of 2δ for strongly interacting case D (figure 6*d*). This significant redistribution of the boundary layer would be otherwise underestimated given the more subtle increase in δ . A more spread-out boundary layer with a similar momentum deficit will presumably allow more incursions of FST from the free stream, as was shown in intermittency weighted averaged profiles in §4.6, where we were able to discern free-stream versus boundary layer contributions to the wall-normal Reynolds stress profiles by conditioning Reynolds stress statistics on the scalar released at the wall. We found that the boundary layer in case C (‘weak’ interaction) was indeed receiving contributions from the FST, but that they did not penetrate deeply as quickly in time as those for cases D and E (‘strong’ interactions). Such incursions then serve to alter the velocity variance profiles, such that they are closer in value to that of the FST. We suggest that in case C, the FST decays more rapidly than the rate at which the boundary layer is able to absorb changes in its velocity profiles. Furthermore, increases in the outer region of velocity variance profiles are only observed when the isotropic u'_e/U_τ of the FST differs significantly from the variance profiles of the

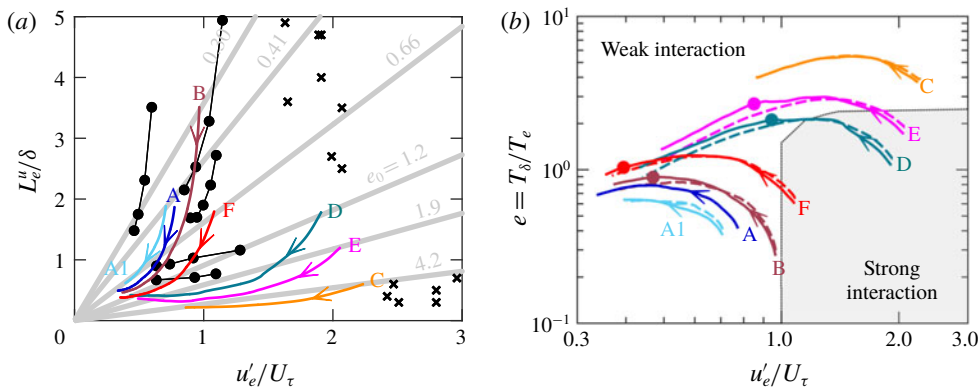


FIGURE 15. Changing relative time scale e on the regime diagram. Coloured curves as per table 2. (a) Regime diagram of figure 7(a) with grey lines of constant $e = (u'_e/L_e^u)/(U_\tau/\delta) = T_\delta/T_e$ added, value noted at top right of each line: ●, Hancock & Bradshaw (1983); ×, Dogan *et al.* (2016). (b) Regime diagram showing evolution of e directly as a function of FST intensity u'_e/U_τ : ●, points at which the interaction becomes ‘weak’, corresponding to the same marked points on figure 8(d); ---, hypothetical evolution of e formed from T_δ of precursor simulations continued without FST injection (as in figure 8) and T_e from the corresponding FST cases.

outer region of the wall-generated turbulence profiles. Looking at, say, $z/\delta \approx 0.5$ in figure 13, the gain over the reference quiescent cases (grey curves) for the total or conventionally averaged $\overline{w^2}/U_\tau^2$ profiles is larger for all cases at an earlier time (figure 13d–f) when the FST intensity is stronger. Profiles deviate less from the quiescent boundary layer profiles at a later time when the FST has decayed significantly (figure 13g–i). In the literature, increases in the peak of the velocity variance profiles have only been shown when u'_e/U_τ of the FST is greater than the peak variance that would be otherwise observed in a boundary layer forming under a quiescent free stream (Thole & Bogard 1996; Dogan *et al.* 2016).

Figure 15(a) adds lines of constant $e = (u'_e/L_e^u)/(U_\tau/\delta) = T_\delta/T_e$, a measure of the relative large-eddy time scales in the present temporal interpretation, to the regime diagram of figure 7(a). As stated above, the main action of the FST on the turbulent boundary layer is to increase the rate of its spread away from the wall, which will then serve to increase the boundary layer thickness δ faster than the $\sim t^{[0.71, 0.73]}$ observed for the quiescent temporal boundary layer of Kozul *et al.* (2016), which we indeed see in figure 8(a) for most of the present cases. Short temporal ranges mean the exponents are somewhat unreliable yet a range of increased exponents $\sim t^{[0.77, 0.80]}$ is found for the present FST cases. This is related to the increased spreading rate for the boundary layer due to FST, an effect found to be stronger for both smaller e_0 and higher FST intensity (figure 8d). An analysis of relevant scales in this problem led to the estimate $e \sim t^0$ (2.5), suggesting e would tend to a constant at large t for non-interacting flows. An increased development rate of δ for a ‘strong’ FST–boundary layer interaction will make the exponent on t more positive. Figure 15(b) plots the evolution of e for the present cases against FST intensity u'_e/U_τ . All cases demonstrate an increase of e in time, before decreasing again. Note that in the present simulations, the increase in e as shown in figure 15(b) is partly attributable to the decrease in L_e^u for early times in excess of that suggested

by power-law fits (figure 3*b*). In figure 8(*d*) we marked the points in time at which we deemed the boundary layer–FST interaction had become ‘weak’ (decrease of boundary layer spreading rate back to that for the boundary layer developing under a quiescent free stream) for cases B, D, E and F. Those same points are marked with filled circles in figure 15(*b*), indicating from which point ‘weak’ interactions presumably begin.

We point to the difference in $e_0 = T_{\delta,0}/T_{e,0}$ when the flows are combined: this quantity differed by a factor of 3 for the two cases C and D that had similar FST intensities at injection. All other things being equal, a different value in e_0 can arise due to a large difference in the relative length scale L_e''/δ , although this does not alone appear to be the critical factor in predicting a ‘strong’ or ‘weak’ interaction. In figure 13(*d*), a strong interaction continues for case D despite $L_e''/\delta = 0.53$ ($e \approx 2$ at this point in time for case D). This length scale ratio is roughly equal to the $L_{e,0}''/\delta_0 = 0.54$ for the weakly interacting case C at FST injection. Since we find a systematic weakening of the effect between cases D ($e_0 = 1.2$) and E ($e_0 = 1.9$), we are led to believe that $e_0 \lesssim 2$ is required for the FST to have sufficient time to strongly interact with the boundary layer. We suggest that if a boundary layer–HIT system satisfies this condition, a ‘strong’ interaction will ensue. We have tentatively shaded a region of the regime diagram in figure 15(*b*) as giving rise to ‘strong’ interactions. The curve of case B is excluded since, although there is apparently scope for the boundary layer and FST to interact temporally, the FST is not sufficiently strong to cause significant changes to velocity variance profiles (figure 12*c*). That is, in addition to the condition on e_0 ensuring that the boundary layer has sufficient time to adjust to the presence of the FST, the FST intensity u_e''/U_τ must also be significant compared to the (peak of the) quiescent velocity variance profiles. The Reynolds numbers of both the turbulent boundary layer and the HIT must presumably be sufficiently high for this inertial criterion to be meaningful. Although we could foresee some Reynolds number effects on the borders of the regime, we expect the essence of the argument on relative time scales to hold.

6. Summary

Bradshaw (1996) noted the importance of parameter e for correlating results for this physical problem over a limited range of FST intensities and length scales, yet at the time it was unclear how general this dependence might be. The present simulations reaffirm the importance of this parameter in predicting the strength of the boundary layer–FST interaction by presenting a limited parametric study of the temporally developing parameter e . Furthermore, for the range of parameters and Reynolds numbers we have here studied, it would be possible to gauge in advance whether or not given HIT will impart change onto a turbulent boundary layer developing beneath it. If the large-eddy turnover time scale of the boundary layer is much larger than that of the FST, then the injected FST will decay to negligible levels before the boundary layer is able to ‘notice’ it. The ‘inactive’ FST fluid does not penetrate closer to the wall than around $z^+ \approx 100$ in such a case. Thus a ‘weak’ interaction could be predicted where a boundary layer’s large-eddy turnover time scale is much larger than that of the FST. Since resulting velocity variance profiles appear approximately as the sum of the FST intensity weighted by the intermittency γ and that for a boundary layer developing under a quiescent free stream, FST intensities must in addition be at least comparable to the peak values of the Reynolds stress profiles for significant change to eventuate. Hancock & Bradshaw (1989) found conditionally sampled

statistics showed a dependence on $(u'_e/L_e^u)/(U_\tau/\delta) = e$, identifying it as a relative fluctuating strain. However, they found that at $e \gtrsim 1$, statistics became markedly less dependent on this parameter. Whilst this implies that the FST has a large relative fluctuating strain compared to the boundary layer, making this, according to such an interpretation, ‘unexpected’, considering it here as a relative large-eddy turnover time scale, we offer an alternative physical explanation: that a large e means that the FST will decay rather more rapidly than the boundary layer is apparently able to adjust to or absorb its effects. Furthermore, Hancock & Bradshaw (1989) did not have the capability to observe co-evolution of the boundary layer and the FST, and were unable to untangle history versus contemporaneous effects. Our temporal approach exposes the dynamism of the boundary layer–FST interaction whose inherent nature will be neglected by an equilibrium model. Despite the advantages of the present temporal model in understanding this interaction, it must be borne in mind that the temporal boundary layer is equivalent to the spatial boundary layer only in the asymptotic limit of large Reynolds number. The (vanishing) difference in entrainment due to the small wall-normal velocity at the edge of the spatial boundary layer, absent in the temporal case, means a potential discrepancy exists between the present simulations and their spatial equivalents with regards to entrained fluid from the free stream.

For the present simulations, the relative time scale interpretation, combined with a condition on the FST intensity u'_e/U_τ , yields a consistent explanation of our results for the Reynolds number range we have been able to tackle. The ability of the FST to change the boundary layer requires the boundary layer to have time – measured in its own time – to be altered by it. Our present temporal investigation sheds light on the conditions under which the boundary layer is able to ‘see’ the injected free-stream disturbances, before they fade away.

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Declaration of interests

The authors report no conflict of interest.

REFERENCES

- ANTONIA, R. A., LEE, S. K., DJENIDI, L., LAVOIE, P. & DANAILA, L. 2013 Invariants for slightly heated decaying grid turbulence. *J. Fluid Mech.* **727**, 379–406.
- BATCHELOR, G. K. 1953 *The Theory of Homogeneous Turbulence*. Cambridge University Press.
- BERNARDINI, M., PIROZZOLI, S., QUADRIO, M. & ORLANDI, P. 2013 Turbulent channel flow simulations in convecting reference frames. *J. Comput. Phys.* **232**, 1–6.

- BLAIR, M. F. 1983*a* Influence of free-stream turbulence on turbulent boundary layer heat transfer and mean profile development, Part I – Experimental data. *Trans. ASME C: J. Heat Transfer* **105**, 33–40.
- BLAIR, M. F. 1983*b* Influence of free-stream turbulence on turbulent boundary layer heat transfer and mean profile development, Part II – Analysis of results. *Trans. ASME C: J. Heat Transfer* **105**, 41–47.
- BRADSHAW, P. 1996 Turbulence modeling with application to turbomachinery. *Prog. Aerosp. Sci.* **32**, 575–624.
- BRANDT, L., SCHLATTER, P. & HENNINGSON, D. S. 2004 Transition in boundary layers subject to free-stream turbulence. *J. Fluid Mech.* **517**, 167–198.
- CARATI, D., GHOSAL, S. & MOIN, P. 1995 On the representation of backscatter in dynamic localization models. *Phys. Fluids* **7**, 606–616.
- CASTRO, I. P. 1984 Effects of free stream turbulence on low Reynolds number boundary layers. *Trans. ASME I: J. Fluids Engng* **106**, 298–306.
- CHUNG, D. & MATHEOU, G. 2012 Direct numerical simulation of stationary homogeneous stratified sheared turbulence. *J. Fluid Mech.* **696**, 434–467.
- DOGAN, E., HANSON, R. E. & GANAPATHISUBRAMANI, B. 2016 Interactions of large-scale free-stream turbulence with turbulent boundary layers. *J. Fluid Mech.* **802**, 79–107.
- HACK, M. J. P. & ZAKI, T. A. 2014 Streak instabilities in boundary layers beneath free-stream turbulence. *J. Fluid Mech.* **741**, 280–315.
- HANCOCK, P. E. 1980 The effect of free-stream turbulence on turbulent boundary layers. PhD thesis, Imperial College, University of London.
- HANCOCK, P. E. & BRADSHAW, P. 1983 The effect of free-stream turbulence on turbulent boundary layers. *Trans. ASME I: J. Fluids Engng* **105**, 284–289.
- HANCOCK, P. E. & BRADSHAW, P. 1989 Turbulence structure of a boundary layer beneath a turbulent free stream. *J. Fluid Mech.* **205**, 45–76.
- HEARST, R. J., DOGAN, E. & GANAPATHISUBRAMANI, B. 2018 Robust features of a turbulent boundary layer subjected to high-intensity free-stream turbulence. *J. Fluid Mech.* **851**, 416–435.
- HEARST, R. J. & LAVOIE, P. 2016 Effects of multi-scale and regular grid geometries on decaying turbulence. *J. Fluid Mech.* **803**, 528–555.
- HOYAS, S. & JIMÉNEZ, J. 2008 Reynolds number effects on the Reynolds-stress budgets in turbulent channels. *Phys. Fluids* **20**, 101511.
- HUANG, M.-J. & LEONARD, A. 1994 Power-law decay of homogeneous turbulence at low Reynolds numbers. *Phys. Fluids* **6**, 3765–3775.
- ISHIDA, T., DAVIDSON, P. A. & KANEDA, Y. 2006 On the decay of isotropic turbulence. *J. Fluid Mech.* **564**, 455–475.
- KAMINSKI, A. K. & SMYTH, W. D. 2019 Stratified shear instability in a field of pre-existing turbulence. *J. Fluid Mech.* **862**, 639–658.
- KANEDA, Y., ISHIHARA, T., YOKOKAWA, M., ITAKURA, K. & UNO, A. 2003 Energy dissipation rate and energy spectrum in high resolution direct numerical simulations of turbulence in a periodic box. *Phys. Fluids* **15**, L21–L24.
- KOZUL, M., CHUNG, D. & MONTY, J. P. 2016 Direct numerical simulation of the incompressible temporally developing turbulent boundary layer. *J. Fluid Mech.* **796**, 437–472.
- KREILOS, T., KHAPKO, T., SCHLATTER, P., DUGUET, Y., HENNINGSON, D. S. & ECKHARDT, B. 2016 Bypass transition and spot nucleation in boundary layers. *Phys. Rev. Fluids* **1**, 043602.
- KROGSTAD, P.-Å. & DAVIDSON, P. A. 2010 Is grid turbulence Saffman turbulence? *J. Fluid Mech.* **642**, 373–394.
- LAVOIE, P., DJENIDI, L. & ANTONIA, R. A. 2007 Effects of initial conditions in decaying turbulence generated by passive grids. *J. Fluid Mech.* **585**, 395–420.
- LI, Q., SCHLATTER, P. & HENNINGSON, D. S. 2010 Simulations of heat transfer in a boundary layer subject to free-stream turbulence. *J. Turbul.* **11**, 1–33.
- MANSOUR, N. N. & WRAY, A. A. 1994 Decay of isotropic turbulence at low Reynolds number. *Phys. Fluids* **6**, 808–814.

- MYDLARSKI, L. & WARHAFT, Z. 1996 On the onset of high-Reynolds-number grid-generated wind tunnel turbulence. *J. Fluid Mech.* **320**, 331–368.
- NAGATA, K., SAKAI, Y. & KOMORI, S. 2011 Effects of small-scale freestream turbulence on turbulent boundary layers with and without thermal convection. *Phys. Fluids* **23**, 065111.
- PAL, A. & SARKAR, S. 2015 Effect of external turbulence on the evolution of a wake in stratified and unstratified environments. *J. Fluid Mech.* **772**, 361–385.
- PÉNEAU, F., BOISSON, H. C. & DJILALI, N. 2000 Large eddy simulation of the influence of high free-stream turbulence on a spatially evolving boundary layer. *Intl J. Heat Fluid Flow* **21**, 640–647.
- PEROT, J. B. 1993 An analysis of the fractional step method. *J. Comput. Phys.* **108**, 51–58.
- PIROZZOLI, S., BERNARDINI, M. & ORLANDI, P. 2016 Passive scalars in turbulent channel flow at high Reynolds number. *J. Fluid Mech.* **788**, 614–639.
- POPE, S. B. 2000 *Turbulent Flows*. Cambridge University Press.
- RIND, E. & CASTRO, I. P. 2012 Direct numerical simulation of axisymmetric wakes embedded in turbulence. *J. Fluid Mech.* **710**, 482–504.
- ROGALLO, R. S. 1981 Numerical experiments in homogeneous turbulence. *NASA TM 81315*.
- SAFFMAN, P. G. 1967 The large-scale structure of homogeneous turbulence. *J. Fluid Mech.* **27**, 581–593.
- SANDERSE, B., VERSTAPPEN, R. W. C. P. & KOREN, B. 2014 Boundary treatment for fourth-order staggered mesh discretizations of the incompressible Navier–Stokes equations. *J. Comput. Phys.* **257**, 1472–1505.
- SCHLATTER, P. & ÖRLÜ, R. 2010 Assessment of direct numerical simulation data of turbulent boundary layers. *J. Fluid Mech.* **659**, 116–126.
- SHARP, N. S., NEUSCAMMAN, S. & WARHAFT, Z. 2009 Effects of large-scale free stream turbulence on a turbulent boundary layer. *Phys. Fluids* **21**, 095105.
- SIMENS, M. P., JIMÉNEZ, J., HOYAS, S. & MIZUNO, Y. 2009 A high-resolution code for turbulent boundary layers. *J. Comput. Phys.* **228**, 4218–4231.
- SPALART, P. R., MOSER, R. D. & ROGERS, M. M. 1991 Spectral methods for the Navier–Stokes equations with one infinite and two periodic directions. *J. Comput. Phys.* **96** (2), 297–324.
- THOLE, K. A. & BOGARD, D. G. 1996 High freestream turbulence effects on turbulent boundary layers. *Trans. ASME I: J. Fluids Engng* **118**, 276–284.
- THORNBUR, B. 2016 Impact of domain size and statistical errors in simulations of homogeneous decaying turbulence and the Richtmyer–Meshkov instability. *Phys. Fluids* **28**, 045106.
- TOWNSEND, A. A. 1961 Equilibrium layers and wall turbulence. *J. Fluid Mech.* **11**, 97–120.
- VERSTAPPEN, R. W. C. P. & VELDMAN, A. E. P. 2003 Symmetry-preserving discretization of turbulent flow. *J. Comput. Phys.* **187**, 343–368.
- WATANABE, T. & NAGATA, K. 2018 Integral invariants and decay of temporally developing grid turbulence. *Phys. Fluids* **30**, 105111.
- WATANABE, T., ZHANG, X. & NAGATA, K. 2018 Turbulent/non-turbulent interfaces detected in DNS of incompressible turbulent boundary layers. *Phys. Fluids* **30**, 035102.
- WESTERWEEL, J., FUKUSHIMA, C., PEDERSEN, J. M. & HUNT, J. C. R. 2009 Momentum and scalar transport at the turbulent/non-turbulent interface of a jet. *J. Fluid Mech.* **631**, 199–230.
- WHITE, F. M. 2006 *Viscous Fluid Flow*, 3rd edn. McGraw-Hill.
- WU, X., MOIN, P., WALLACE, J. M., SKARDA, J., LOZANO-DURÁN, A. & HICKEY, J.-P. 2017 Transitional–turbulent spots and turbulent–turbulent spots in boundary layers. *Proc. Natl Acad. Sci. USA* **114**, E5292–E5299.
- WU, X., WALLACE, J. M. & HICKEY, J.-P. 2019 Boundary layer turbulence and freestream turbulence interface, turbulent spot and freestream turbulence interface, laminar boundary layer and freestream turbulence interface. *Phys. Fluids* **31**, 045104.
- XIA, S., ITO, Y., NAGATA, K., SAKAI, Y., SUZUKI, H., TERASHIMA, O. & HAYASE, T. 2014 DNS study on the development of boundary layer with heat transfer under the effects of external and internal disturbances. *J. Fluid Sci. Technol., JSME* **9**, 1–13.
- YOU, J. & ZAKI, T. A. 2019 Conditional statistics and flow structures in turbulent boundary layers buffeted by free-stream disturbances. *J. Fluid Mech.* **866**, 526–566.