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ABSTRACT

This study numerically investigates the flow and aeroacoustic characteristics of rectangular jets with different nozzle geometries but with the same mass flow rate. Turbulence statistics in the near field are compared, and the influence of nozzle shape on jet dynamics and associated noise is assessed. The configurations analyzed include a planar nozzle and orifice-type nozzles with square, beveled, and filleted edges. The Reynolds number based on the nozzle height is 1.1×10^4 and the maximum Mach number is approximately 0.17. Flow fields are computed using Delayed Detached Eddy Simulation with the Spalart–Allmaras turbulence model, while far-field noise is obtained using Farrisat's 1A formulation from permeable integration surfaces. The numerical framework is validated against experimental data. Results demonstrate that square and beveled orifices induce the vena contraction effect by their sharp edges, resulting in higher exit velocities and elevated near-field turbulence. In contrast, planar and filleted nozzles exhibit weaker turbulence, delayed shear layer roll-up, and longer potential cores. The beveled orifice in particular exhibits pronounced tonal noise peaks at 4.3, 8.5, and 12.7 kHz in the turbulent velocity spectrum. Spectral proper orthogonal decomposition identified dominant energetic modes near the potential core boundary where vortex roll-up and shear-layer interaction occur. The sharp-edged orifices produce overall sound pressure levels approximately 10 dB higher when integrating across the full frequency spectrum (including the tones) and 7 dB higher for the broadband component, compared to the planar and filleted nozzles. These findings underscore the trade-off between jet mixing and noise mitigation in nozzle design, offering insights for optimizing rectangular jets for engineering.

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I. INTRODUCTION

Rectangular jets exhibit unique and distinct flow characteristics compared to circular jets. They have applications in aerodynamic flow control, noise control, and blowing applications. Examples include noise reduction in aircraft^{1,2} and high-speed trains,³ high-velocity hand dryers in public restrooms,⁴ and enhanced mixing of air and fuel in combustion systems.^{5,6} For instance, a practical application of rectangular jets is in the flow control of bluff body wakes. Although many studies have been conducted to reduce the noise generated by cylinders,^{7,8} the potential of rectangular jets in this context remains significant.^{9,10} The growing interest in rectangular jets has led to increased focus on aspects such as aerodynamic efficiency and noise reduction. This necessitates a comprehensive understanding of rectangular jet

dynamics and the potential influential factors, such as nozzle profiles, aspect ratio, Reynolds number, and inlet boundary conditions. Such a fundamental understanding is critical for expanding their applications across diverse sectors.

A circular jet spreads uniformly in all directions, whereas rectangular jets exhibit distinct spreading rates along their major and minor axes, leading to a flow that is no longer axisymmetric, particularly in the near-field region of the jet nozzle. As shown in Fig. 1, the major axes refer to the jet width direction (Z direction), while the minor axes refer to the jet height direction (Y direction). For a rectangular jet nozzle, the aspect ratio is defined as the ratio between the jet width and jet height. When the aspect ratio becomes sufficiently large, the flow field exhibits approximately two-dimensional characteristics. In this case,

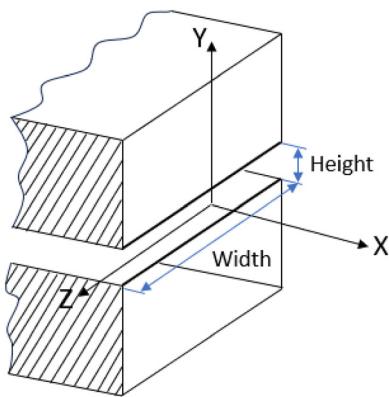


FIG. 1. Schematic of a planar jet.

the rectangular jet is referred to as a “planar jet” as the flow properties are predominantly confined to a plane that is orthogonal to the major axes.

Sfeir^{11,12} experimentally studied the three-dimensional flow properties of a rectangular jet with different aspect ratios (10–60) using hot-wire anemometry. They found that, for a Reynolds number of 1.22×10^4 , based on jet height, there are limited changes in the centerline velocity decay, spreading rate, and turbulence intensity with changes in aspect ratio. However, for a lower Reynolds number of 0.37×10^4 , the effect of aspect ratio becomes more significant. At lower values of aspect ratio, at this Reynolds number, the centerline velocity decay, spreading rate and turbulence intensity behavior is different from two-dimensional jet behavior. As the aspect ratio increases, these differences become smaller. In an experiment with aspect ratios ranging from 2 to 20, Quinn¹³ demonstrated that higher aspect ratios increase mixing rates, reduce the length of the potential core, and increase turbulence kinetic energy in the shear layers. Deo *et al.*¹⁴ systematically studied the effect of aspect ratio on the flow properties of rectangular jets experimentally. Their research aligned with Quinn¹⁵ and also showed that increasing the nozzle aspect ratio enhances jet spreading and mixing efficiency.

Deo *et al.*¹⁶ expanded upon this work by investigating the influence of Reynolds number on rectangular jets. Their findings indicate that a rectangular jet is sensitive to the nozzle exit conditions, such as velocity profile, turbulence intensity, and nozzle geometry, at Reynolds numbers below 10^4 , based on jet height. However, this sensitivity decreases as the Reynolds number increases. This observation is consistent with the findings from Sfeir.¹² Similar to circular jet flow, a Reynolds number of 10^4 is a critical threshold in rectangular jet flows. It often marks a transitional regime where the flow evolves from a laminar-like regime with coherent shear layers to a fully turbulent state characterized by enhanced mixing, breakdown of organized vortices, and increased jet spreading rates due to intensified turbulent entrainment and three-dimensional vortex interactions.^{16,17} This makes it an important condition for understanding jet dynamics and designing systems involving jet flows.

The aspect ratio, Reynolds number, and inlet conditions have all been identified as factors influencing the behavior of rectangular jet flows. Another crucial factor is the nozzle geometry. In the study by Sfeir,¹¹ two types of nozzles were tested. One consisted of rectangular

slots with sharp edges, while the other one featured rectangular channels with a length equal to fifty times the slot height. In the experiment, the nozzle geometry was found to significantly influence both the velocity and temperature fields, such as velocity and temperature decay along the centerline, the velocity profile along the major axis (Z direction in Fig. 1), and the jet width distribution. In a follow-up experiment, Sfeir¹² found that jets emanating from long rectangular channels have shorter development lengths than those from sharp-edged orifices. The flow also remains more uniform in the X-Z plane (as defined in Fig. 1) over extended downstream distances. In contrast, jets from sharp orifices, particularly those with small aspect ratios, are less likely to develop into self-preserving two-dimensional flows. The characteristic “saddle-back” velocity profile along the major axis is more pronounced in these cases than in jets from long rectangular channels.

Further insights from Deo *et al.*¹⁸ revealed that the geometric profile of nozzle exits, such as sharp or filleted edges, directly impacts the shear layer interactions, turbulence intensity, and downstream flow stability, emphasizing how shape-specific features of the nozzle govern jet behavior. Abdel-Rahman¹⁹ reviewed the relevant research and highlighted that nozzle shape dictates the initial shear layer formation and turbulence at the jet exit, leading to variations in mixing and spreading rates. Together, these findings demonstrate that nozzle shape significantly alters the flow field by changing turbulent structures and mixing efficiency, making it a critical design parameter in engineering applications.

In addition to the experimental research, numerical studies have also been conducted to investigate the flow properties of rectangular jets. To simplify the analysis, computational models were often reduced to limited span representations with periodic boundary conditions applied in the simulations. Early work by Le Ribault *et al.*^{20,21} employed large eddy simulations (LES) to study planar jets, revealing insights into the development of the shear layers and demonstrating the accuracy of LES in capturing jet flow dynamics. A recent study by Matsuyama²² also employed LES but with an implicit sub-grid viscosity model to study the near-field turbulence properties of a rectangular jet at a Reynolds number of 10^4 . This study shows the accuracy of implicit LES in resolving these multi-scale interactions without requiring subgrid-scale modeling, providing robust predictions of jet spreading rates and turbulence intensities.

In addition to LES, direct numerical simulation offers resolution of turbulence without relying on sub-grid scale models, as demonstrated in studies of transitional rectangular jets.^{22,23} These investigations provided benchmark data for validating turbulence models and highlighted the importance of the initial conditions in determining jet evolution. Due to the computational cost of LES and Direct Numerical Simulation (DNS), the jet nozzle geometry is often replaced by a simplified inlet boundary condition, and the flow is simulated under the assumption of infinite spanwise extent using periodic boundary conditions. Few studies have considered the three-dimensional effect from the jet nozzle aspect ratio and moved beyond the infinite spanwise assumption.^{24,25} These investigations demonstrate that near-field turbulence statistics are significantly influenced by nozzle geometry and boundary conditions, whereas in the far-field the flow transitions toward self-similarity. The study of Jiang *et al.*²⁵ also shows the frequency characteristics of coherent structures in rectangular jets and identifies dominant frequencies that govern vortex roll-up in the shear layer, entrainment, and transition from organized instabilities to fully

developed turbulent flow downstream. These findings provide insights into jet flow behaviors and their associated phenomena.

While previous experimental and numerical studies have explored the influence of nozzle geometry on the development of rectangular jets, their focus has primarily been on statistical flow properties. Although some recent studies have used high-fidelity numerical methods to investigate the noise generation mechanisms of jet flows, they primarily focus on circular jets.^{26–28} While several experimental studies have examined the effects of nozzle geometry on jet flows,^{11,12,18} most of these investigations focused on the aerodynamic characteristics, with limited attention to the aerodynamic noise and detailed flow dynamics. The aeroacoustic noise generated by rectangular jets has attracted increased attention, also motivating this current work. Furthermore, the near-field flow features of rectangular nozzles, e.g., vena contraction, vortex development, and entrainment, have not been systematically reported for different nozzle geometries. In this work, the different nozzle geometries have the same mass flow rate, allowing them to be directly compared and the effect of the nozzle shape isolated. In addition, existing numerical simulations often simplify the nozzle inlet conditions by prescribing predefined velocity profiles instead of modeling the actual nozzle geometry. This simplification limits the ability to accurately capture the influence of nozzle shape on the jet behavior and its resultant acoustic characteristics.

To address this gap, this study conducts numerical simulations aimed at investigating the impact of nozzle geometries on the near-field turbulence statistics, jet dynamics, and the aerodynamic noise of rectangular jets, which are commonly used in engineering applications. The novelty of this work lies in the first-time investigation of both jet dynamics and aerodynamic noise in different rectangular jet configurations. The nozzle configurations analyzed include a planar nozzle and orifice-type nozzles with square, beveled, and filleted edges. These geometries have wide engineering applications and have been studied by researchers.^{11–13,15,18,29} By comparing the flow field, jet dynamics, and aeroacoustic characteristics of the four configurations, the influence of nozzle geometry will be quantitatively assessed. The near-field turbulence statistics, vortex roll-up frequencies, and noise levels will be evaluated. The findings will facilitate the selection and design of rectangular nozzles, supporting tradeoffs between jet mixing and acoustic noise in engineering applications.

To balance the simulation accuracy and computational cost, fully compressible Delayed Detached Eddy Simulation with the Spalart–Allmaras turbulence model is employed to simulate the flow field. This approach enables the numerical investigation of various nozzle geometries while maintaining both accuracy and affordable computational cost. To calculate the far-field noise, the free-space time-domain solution of the Ffowcs Williams–Hawkings equations (Farassat 1A formulation) will be used. To isolate the effect of nozzle shape and exclude other influential factors, the simulations will be performed under the infinite spanwise length assumption similar to the boundary conditions used in the numerical studies discussed previously. The bulk flow velocity of all the simulations will be kept constant, and the Reynolds number based on the bulk flow velocity and the height of the jet is approximately 1.1×10^4 . Spectral Proper Orthogonal Decomposition (SPOD) is employed to analyze the flow field, offering detailed insights into the spatial distribution and influence of dominant flow modes and their correlation with the radiated acoustic noise. The paper begins with an overview of the numerical methods, followed by details of the

computational domain, grid discretization, and numerical validations. Finally, it presents an analysis of the results for various rectangular jet geometries and their effects on near-field turbulence, flow dynamics, and aerodynamic noise.

II. NUMERICAL BACKGROUND

A. Delayed detached Eddy simulation

In this work, the compressible Navier–Stokes (N–S) equations are adopted, even though the highest Mach number at the nozzle exit in some cases is only approximately 0.17. Detached Eddy Simulation (DES), which combines Large Eddy Simulation (LES) and Unsteady Reynolds-Averaged Navier–Stokes (URANS), is chosen to balance simulation accuracy and computational cost. DES was first proposed by Spalart *et al.*³⁰ In this method, regions near solid boundaries are simulated using URANS, while LES is applied outside these regions. As a result, the computational cost of DES is considerably reduced compared to LES. DES is not a zonal approach because the switch between URANS and LES is determined by the local grid near the solid wall, rather than a predetermined parameter setting for specific zones. The switch is applied by replacing the length scale with the minimum of the wall distance and a grid-based length. However, in separation or transition flows, which are sensitive to the grid and turbulence models, grid-induced separation may occur at the edge of the boundary layer depending on the computational mesh.^{31,32} Therefore, Spalart *et al.*³³ proposed a new treatment to address this issue, which is known as Delayed Detached-Eddy Simulation (DDES). Compared with the DES method, the length scale in DDES is not only dependent on the grid but also on the flow field variables. This improves predictions, particularly for cases involving flow separation, and has been validated in simulations across various scales.^{34–36} The detailed equations can be found in Ref. 33. To ensure consistency with prior studies and to ensure a reasonable computational cost, the DDES methodology is adopted in the present work.

B. Ffowcs Williams and Hawkings equation

The Ffowcs Williams–Hawkings (FW–H) method is an acoustic analogy that extends Lighthill's equation to predict sound generated by moving bodies and turbulent flows by integrating flow quantities over a control surface surrounding the noise sources. Farassat's Formulation 1A is a solution to the Ffowcs Williams and Hawkings (FW–H) equation in free space, where the solid surfaces move at subsonic speeds. The detailed derivation of this formula can be found in Farassat.³⁷ It is expressed as

$$4\pi p'_T(\mathbf{x}, t) = \int_{f=0} \frac{\rho_0(\dot{U}_n + U_{\dot{n}})}{r(1 - M_r)^2} dS + \int_{f=0} \frac{\rho_0 U_n \{r\dot{M}_r + c_0(M_r - M^2)\}}{r^2(1 - M_r)^3} dS, \quad (1)$$

$$4\pi p'_L(\mathbf{x}, t) = \frac{1}{c_0} \int_{f=0} \frac{\dot{L}_r}{r(1 - M_r)^2} dS + \int_{f=0} \frac{L_r - L_M}{r^2(1 - M_r)^2} dS + \frac{1}{c_0} \int_{f=0} \frac{L_r \{r\dot{M}_r + c_0(M_r - M^2)\}}{r^2(1 - M_r)^3} dS, \quad (2)$$

where c_0 and ρ_0 denote the sound speed and air density in the farfield, r is the distance to the observer, S is the integral surface, and M is the

Mach number. Eq. (1) is the thickness noise (p'_T) and Eq. (2) is the loading noise (p'_L). The dot above a variable (\cdot) denotes the time derivative. The condition $f = 0$ indicates that the variables are evaluated on the integration surface. The subscripted quantities are the dot products between a vector and the unit vector implied by the subscript. For example, $M_r = M \cdot \mathbf{r}$, where \mathbf{n} and \mathbf{r} represent the surface normal unit vector (\hat{n}_i) and the radiation direction unit vector (\hat{r}_i) respectively. $U_i = v_i + \rho/\rho_0(u_i - v_i)$, where u_i denotes the fluid velocity and v_i denotes the surface velocity. The loading source term is defined as $L_i = P_{ij}\hat{n}_j + \rho u_i(u_n - v_n)$, where P_{ij} is the compressible stress tensor. An in-house FW-H solver based on permeable FW-H integral surfaces was used in this study.

III. SETUP AND VALIDATION

A. Geometry and numerical settings

Four nozzle configurations are considered in this study, as illustrated in Fig. 2. These are a planar nozzle, a square-edge orifice, a beveled-edge orifice, and a filleted-edge orifice. The planar nozzle, as shown in Fig. 2(a), allows the inlet flow to develop before exit and serves as the baseline case due to its simplicity and practical relevance. The orifice-type nozzles [Figs. 2(b)–2(d)] feature a finite wall thickness of 2 mm and a nozzle height of 4 mm. The beveled orifice has a 45° edge angle, while the filleted orifice includes a 2 mm internal radius. These nozzle configurations are relevant, and similar configurations have been investigated experimentally in rectangular jets in different studies by Sfeir,¹² Quinn,¹³ and Deo *et al.*¹⁸ This present work examines the near-field turbulence, jet dynamics, and aerodynamic noise associated with these nozzle geometries systematically at the same mass flow rate.

Figure 3 shows the computational domain for the jet flow using the planar nozzle as an example. The domain was designed to ensure full jet development without boundary effects while maintaining computational efficiency. The downstream outlet is located 300 h from the front outlet and 5 h from the front wall, with a 200 h spacing between the top and bottom boundaries. The front wall (planar case) and the orifice chamber both have a width of 10 h , providing a balance between accuracy and cost. Free-outflow and non-reflective pressure boundary conditions were applied at the outlet, front outlet, and upper and lower planes to prevent wave reflection,³⁸ while no-slip conditions were enforced on all solid walls. Periodic boundaries in the spanwise direction simulate an infinite span, following a common practice for

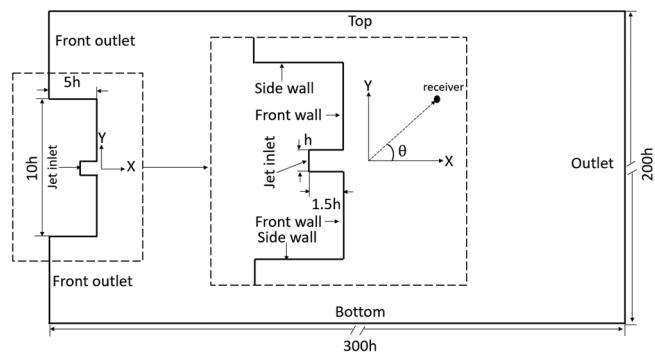


FIG. 3. Computational domain.

reducing computational expense.²² The rationale for the chosen spanwise length is discussed in Sec. III B.

As demonstrated in the studies of Deo *et al.*,¹⁶ the flow characteristics of a rectangular jet exhibit a pronounced dependence on the Reynolds number ($Re = U_b h/\nu$) below 10^4 . However, this dependency becomes less significant as the Reynolds number exceeds 10^4 , marking a transition toward behavior more characteristic of fully turbulent jets. This transition is of critical importance in the study of rectangular jet flows. Therefore, in the present study, a Re of 1.1×10^4 is selected, achieved by setting the jet height (h) to 4 mm and the mean bulk flow velocity (U_b) to 40 m/s. U_b is defined as,

$$U_b = \frac{\int_A \rho (\mathbf{U} \cdot \mathbf{n}) dA}{\int_A \rho dA}, \quad (3)$$

where ρ is the density, \mathbf{U} is the time-averaged velocity vector, \mathbf{n} is the cross-sectional unit normal vector, and A is the cross-sectional area. $\int_A \rho (\mathbf{U} \cdot \mathbf{n}) dA$ is the mass flow rate, which is approximately 8.0×10^{-3} kg/s for the four cases. As shown by the dashed lines in Fig. 2, the mean bulk flow velocity is measured at the nozzle exit plane for the planar nozzle and the filleted orifice, while it is measured at the inlet plane for the square and beveled orifices. These correspond to the region of minimum cross-sectional area for each nozzle type. Defining the Reynolds number based on the mean bulk velocity is justified by the fact that this quantity is typically known, whereas the local velocity

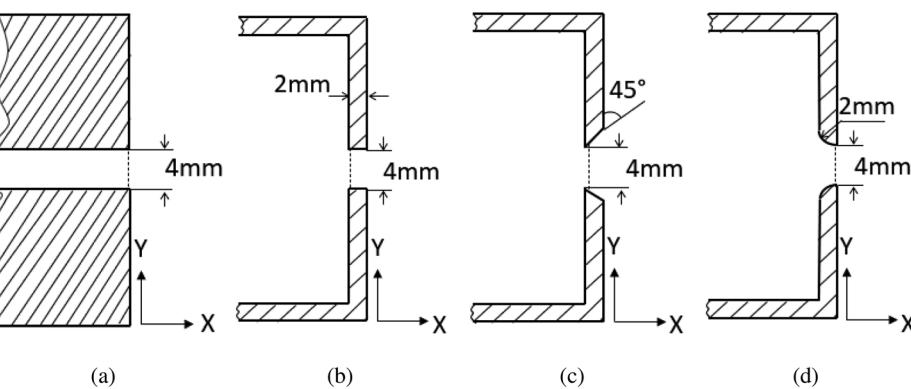


FIG. 2. Different shapes of jet exits, (a) planar nozzle; (b) square-edge orifice; (c) beveled-edge orifice; and (d) filleted-edge orifice. The dashed lines denote the minimum cross-sectional area of the nozzle.

distribution is often unknown in practical engineering scenarios. This definition also facilitates meaningful comparisons between jets issuing from different nozzles operating under the same mass flow rate.

Prior to simulating the jet flow, a precursor simulation of a channel flow was conducted to generate time-varying inflow velocity data for the inlet boundary conditions of planar nozzle case, as shown in Fig. 2(a). Using a fully developed channel flow as the inlet boundary condition for the planar jet, as adopted in the experimental study of Sfeir,¹² allows the exclusion of potential influences from the inlet velocity profile and turbulent intensity. The precursor simulation also enables the use of a shorter inlet section for the planar nozzle, thereby reducing the complexity and computational cost of the simulation. Prior to data collection, the precursor simulation was conducted for approximately 400 dimensionless time units,

$$t^* = \frac{tU_b}{h}, \quad (4)$$

where t is physical time, U_b is the bulk flow velocity of the jet, and h is the jet height. The turbulence statistics are shown in Appendix A. The collected time-varying velocity data were then assigned to the inlet boundaries of the planar nozzle case using the OpenFOAM function `timeVaryingFixedValue`. Further details regarding this procedure can be found in Ref. 3. For the orifice cases, a normal velocity boundary condition with very low turbulence intensity (<1%) was prescribed at the inlet, which is consistent with the experimental studies of Deo *et al.*^{16,18}

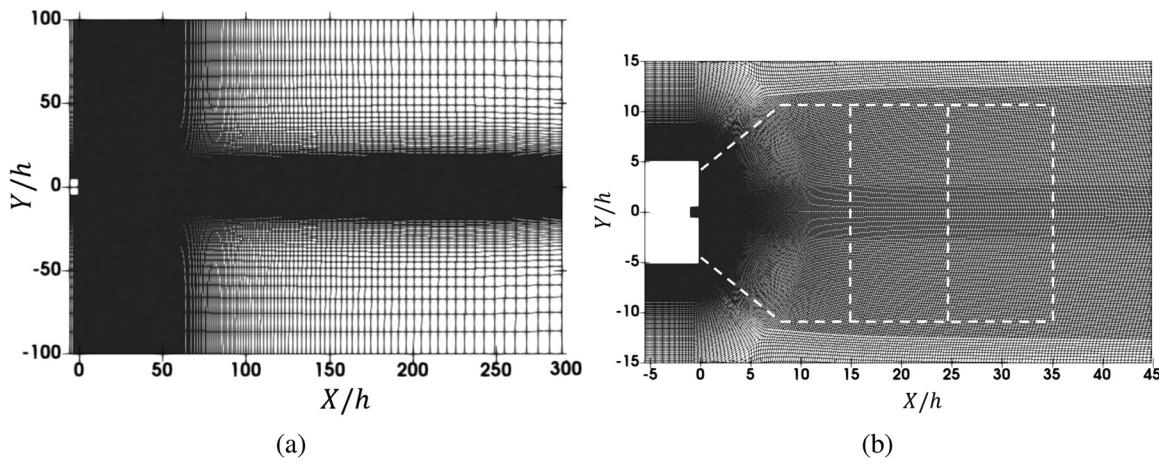


FIG. 4. The mesh of type I exit in the X-Y plane. (a) Overview of the mesh and (b) mesh in the vicinity of the jet nozzle.

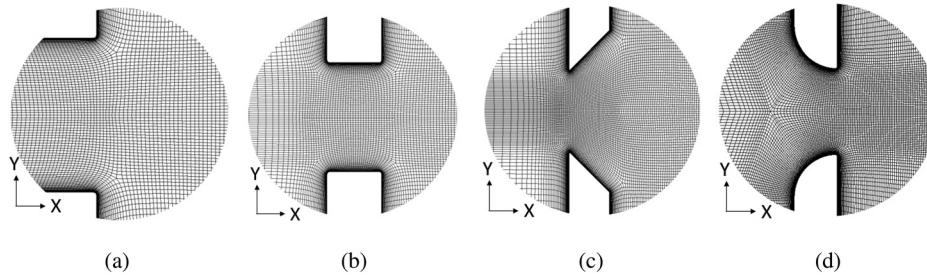


FIG. 5. Details of the mesh for different types of exits. (a) Planar; (b) square; (c) beveled; and (d) filleted.

TABLE I. Grid parameters of the grid independence study.

Case	$\Delta x/\Delta y$	$\max(\Delta x, \Delta y)/h$	$\Delta z/h$	N_{xy}
Baseline	16	0.2	0.2	1.31×10^5
Coarse	32	0.4	0.2	0.86×10^5
Fine	8	0.1	0.2	2.56×10^5
Spanwise-coarse	16	0.2	0.33	1.31×10^5
Spanwise-fine	16	0.2	0.1	1.31×10^5

Method for Pressure-Linked Equations), was utilized to solve the discretized equations. Prior to initiating the time-resolved simulation, the initial flow field was obtained through Reynolds-Averaged Navier-Stokes (RANS) to provide an initial flow field. For the unsteady simulation, a second-order implicit temporal scheme, specifically the “backward” time marching method, was adopted. The physical time step was set to 5×10^{-7} s. This corresponds to a Courant–Friedrichs–Lewy (CFL) number between 0.5 and 1. For spatial discretization, a bounded second-order Total Variation Diminishing (TVD) scheme is employed. This scheme is commonly adopted in high-fidelity simulations to achieve a balance between numerical accuracy and solution stability and has been validated in numerical studies across a range of model scales.^{34,35}

B. Grid resolution study and validation

The grid dependency study was conducted using the planar nozzle case with a spanwise length of $2h$ to reduce computational cost. Since the inlet velocity data are derived from a channel flow simulation, a long inlet section is not required. Therefore, the inlet length is set to only $1.5h$, as shown in Fig. 3. As shown in Figs. 4 and 5, the grid is finest near the nozzle exit. The size gradually increases in the transverse and streamwise directions. However, the grid size inside the integral surface [see Fig. 4(b)] maintains $\max(\Delta x, \Delta y) \leq 0.2h$. The details of the grid independence study cases are summarized in Table I. To maintain $y^+ < 1$, the first boundary layer grid height is set to $1.23 \times 10^{-3}h$. The mesh size is adjusted simultaneously along the solid surface and in the perpendicular direction. The aspect ratio of the baseline case was maintained at a constant value of 16, which is sufficiently small for both DDES and LES simulations.⁴⁰ The value of

$\max(\Delta x, \Delta y)$ is taken within the region enclosed by the permeable FW–H surfaces, as highlighted in Fig. 4(b). N_{xy} denotes the number of cells in the X–Y plane and increases by approximately three times from the coarse mesh case (coarse) to the fine mesh case (fine). The primary changes in cell count occur downstream of the jet exit, while the number of cells in the spanwise direction remains constant across the baseline, coarse, and fine cases. To investigate the effect of spanwise resolution, additional cases (spanwise-coarse and spanwise-fine) were included, where the grid resolution in the spanwise direction was varied. This approach ensures a comprehensive evaluation of grid sensitivity while maintaining reasonable computational cost.

Figure 6 presents the time-averaged velocity profiles along the centerline and in the transverse direction. U_{cl} is the mean streamwise velocity at the centerline. It is observed that the centerline velocity profiles for all cases [Fig. 6(a)], except for the Spanwise-coarse case, exhibit good agreement with each other. The deviations at each monitor between the numerical results and the experimental data are all less than 3%, except for the spanwise-coarse case, which shows a difference of approximately 8% from $30h$ to $60h$. A similar trend is also noted for the mean velocity profiles in the transverse direction [Fig. 6(b)]. η is defined as the varying dimensionless transverse position (Y/X_{cl} , where X_{cl} is the streamwise position of the transverse line). The discrepancy observed in the spanwise-coarse case, up to 40%, highlights the critical importance of the spanwise grid resolution for accurately simulating jet flow. Apart from this discrepancy with inadequate spanwise resolution, the simulated centerline and transverse velocity profiles agree well with the experimental data from Deo *et al.*¹⁶ and Sfeir.¹² Therefore, based on the above comparisons, the grid parameters from the baseline case are employed in the subsequent simulations.

As mentioned earlier, a periodic boundary condition is applied in the spanwise direction of the simulation domain. To justify an adequate spanwise length of the simulations, the spanwise correlation coefficients were calculated for four cases with spanwise lengths of $2h$, $5h$, $10h$, and $15h$, as shown in Appendix B. To balance accuracy in capturing the spanwise variations and the computational cost, the case with a spanwise length of $10h$ was chosen for the subsequent simulations. Therefore, the subsequent validation is performed based on the simulation case with a spanwise domain width of $10h$. In the simulation, the flow field becomes stable after a dimensionless time t^*

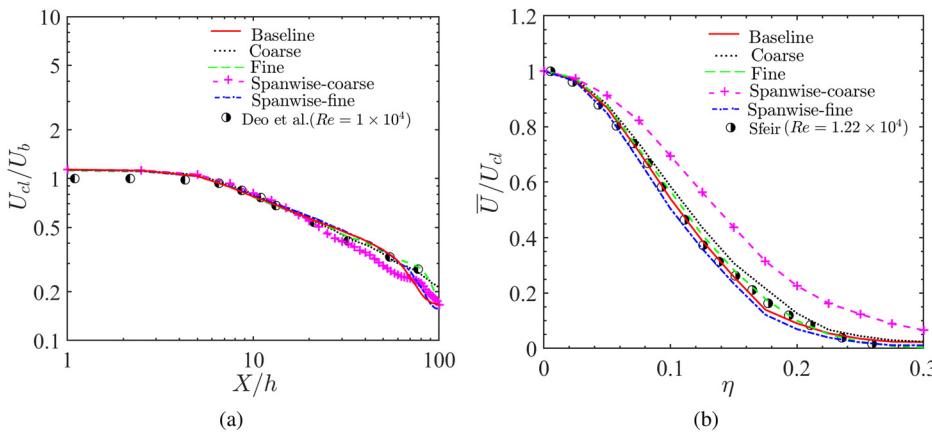


FIG. 6. Mean velocity profiles. (a) Streamwise direction. The experimental data are from Ref. 16. (b) Transverse direction at $x = 30h$. The experimental data are from Sfeir.¹²

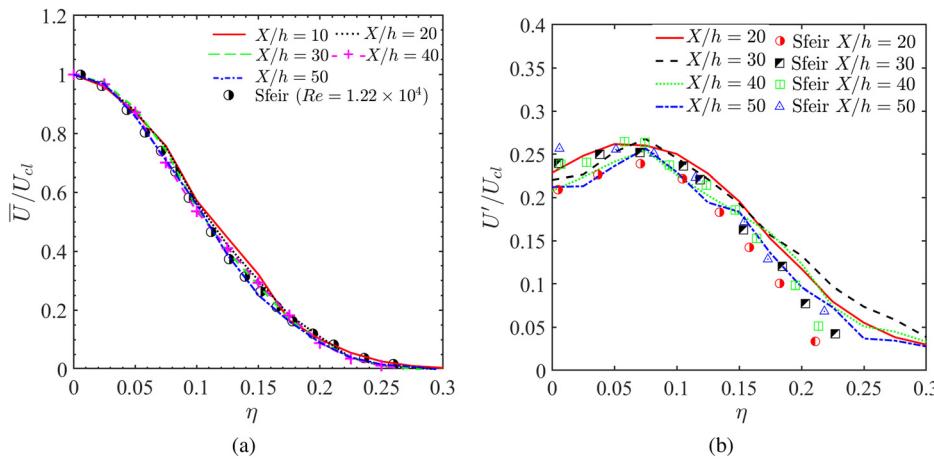


FIG. 7. Dimensionless velocity profile in the transverse direction. (a) Mean velocity profiles and (b) turbulent intensity in the streamwise direction. The experimental data are from Ref. 12.

[defined in Eq. (4)] of approximately 10^3 . The data sampling is then performed over the same time duration again. The total computational wall time for this process was approximately 150 h, utilizing 320 cores on Intel Xeon Gold 6138 processors on the Iridis5 cluster at the University of Southampton.

The examination of self-similarity behavior is illustrated in Fig. 7. In Fig. 7(a), the fitted mean transverse velocity profile from Ref. 12 is used as a reference. The numerical results show excellent agreement with the experimental data. Additionally, the numerical results also demonstrate strong self-similarity between the velocity profiles at different positions ($10 < X/h < 50$), indicating that the flow exhibits consistent scaling behavior in this region. This self-similarity is a characteristic feature of fully developed turbulent jets. In Fig. 7(b), the streamwise turbulence intensity (U'/U_{cl}) along the transverse direction is compared, where U' is the root mean square of the streamwise velocity. The streamwise turbulence intensity also shows a good agreement with the experimental data.

To optimize the placement of the FW-H permeable integral surface, three different locations, as shown in Fig. 4(b), are tested to determine the most effective position for accurately capturing the jet noise. The integral surfaces, shown alongside with the divergence of velocity derivative and the normalized Q criterion (Qh^2/U_b^2), are shown in Fig. 23 in Appendix B. The results, presented in Fig. 24 in Appendix B, show that the noise spectra converge with the placement of the integral surfaces for the two largest integral surfaces. The largest of the integral surfaces was chosen for the acoustic analysis as it encloses a larger portion of the jet.

To validate the acoustic simulation, a jet configuration matching the experimental conditions (listed in Table II) is simulated. The jet height matches the experimental setup, while the spanwise length is

reduced to 10 mm to decrease computational costs. Periodic boundary conditions are used in the spanwise direction. Grid parameters, such as the aspect ratio and mesh distribution, are kept consistent with the baseline case in Table I. However, other parameters, including the height of the first boundary layer mesh and the volume mesh resolution, are refined because the jet velocity increases to 149 m/s. The total grid count for the validation simulation is approximately 27×10^6 cells. All flow conditions match those used in the experiment conducted by Munro and Ahuja.⁴¹

Figure 8 compares the simulated noise spectra of the acoustic validation case with experimental data from Ref. 41. In the comparison, the numerical result is scaled to account for the difference in spanwise width with the experimental setup previously discussed using the following equation:

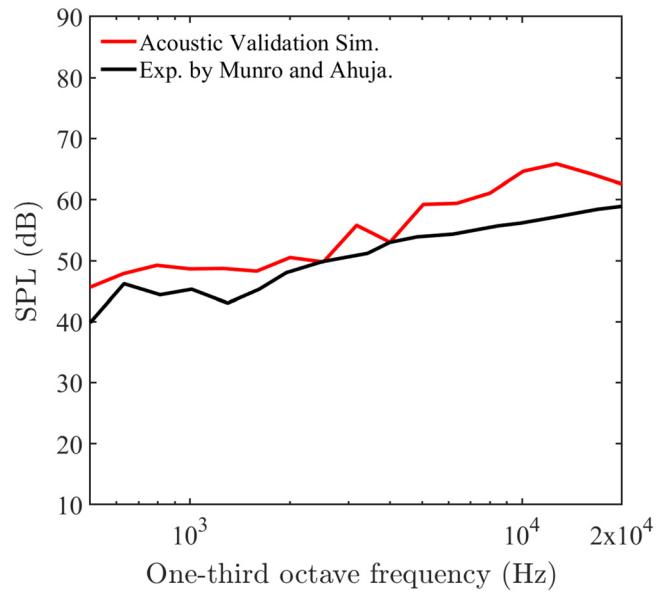


FIG. 8. Validation of the aeroacoustic simulation at an angle of 60° and a radial distance of 2.74 m. The experimental data are from Ref. 41.

TABLE II. Jet parameters for the current acoustic validation simulation and the experiment by Munro and Ahuja.⁴¹

	Acoustic validation sim.	Munro and Ahuja ⁴¹
U_b (m/s)	149	149
h (mm)	1	1
w (mm)	10	375

$$\Delta \text{SPL} = 10 \log_{10} \left(\frac{w_1}{w_2} \right), \quad (5)$$

where w_1 is the nozzle width in the acoustic validation simulation case, and w_2 is the nozzle width in the experiment by Munro and Ahuja.⁴¹ A reasonably good agreement is observed between the simulation and experimental acoustic data, despite, in general, some over-prediction across most frequencies. This discrepancy can be attributed to two main factors. One is the use of periodic boundary conditions in the spanwise direction in the simulation. This neglects end effects, which are present in the experiment. The second is the absence of width consideration in the frequency scaling. Nevertheless, despite these approximations, the overall agreement supports the validity of the numerical simulation setup.

IV. RESULTS AND DISCUSSION

A. Flow field

Based on the streamwise velocity decay, a rectangular jet flow can be divided into three regions. The first is the potential core region, where the streamwise velocity is essentially constant. The second is the two-dimensional region, where turbulence increases and self-similarity begins. The third region is where the flow becomes fully developed and turbulence becomes more homogeneous and isotropic in nature, with large-scale turbulent structures dominating the dynamics.⁴² This region is analogous to the axisymmetric region in a circular jet. Experimental research has demonstrated that the influence of nozzle geometries is most pronounced in the vicinity of the nozzle exit (the potential core region and the two-dimensional region), while the effects tend to diminish and the flow becomes self-similar further downstream.^{11,12,18}

The normalized mean velocity magnitude (\bar{U}_a/U_b) near the potential core region is presented in Fig. 9. \bar{U}_a is the time-averaged magnitude of the velocity defined as $\bar{U}_a = \sqrt{\bar{U}^2 + \bar{V}^2 + \bar{W}^2}$. The potential core length (x_p) is defined as the distance from the jet exit to the point on the centerline, where $\bar{U} = \bar{U}_{\text{exit}}$, and \bar{U} is the mean streamwise velocity. The potential core regions are highlighted by the solid white lines, and the jet half-widths are highlighted by the black dashed lines. The jet half-width ($y_{0.5}$) is defined as the transverse distance from the jet centerline to the point where $\bar{U} = 0.5U_d$. In Fig. 9(a), the flow velocity at the exit of the planar nozzle is relatively uniform within the first nozzle height region and initially remains unexpanded within the potential core region. This behavior is similar to the filleted orifice. For both the square and beveled orifices, both of which have a sharp edge at the exit, there is a vena contraction as highlighted by the white arrow in Figs. 9(b) and 9(c). The vena contraction forms just downstream of the sharp edge due to the transverse velocity component developed near the orifice wall. As the flow is forced through the orifice, it abruptly redirects streamwise, but the transverse momentum persists, leading to flow contraction and acceleration. Consequently, the jet velocity is higher in these cases compared to the planar nozzle. This phenomenon was also reported in the experimental studies from Deo *et al.*¹⁸ and Quinn.¹³

The filleted nozzle also demonstrates a vena contraction [Fig. 9(d)], although it is much weaker than the vena contraction seen in the square and beveled orifices [Figs. 9(b) and 9(c)]. This can be attributed to the presence of the fillet, which gradually directs the flow in the streamwise direction. Unlike the sharp-edged nozzles, where the

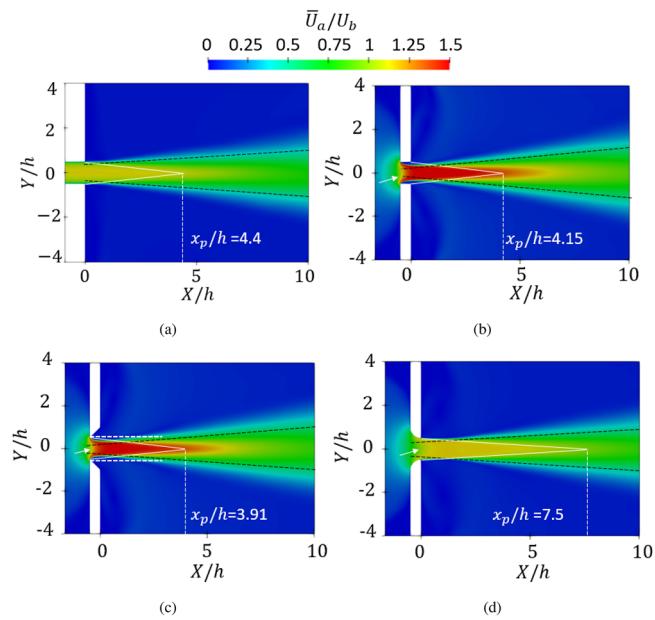


FIG. 9. The normalized time-averaged velocity magnitude \bar{U}_a/U_b contour in the vicinity of different nozzle exits, (a) planar nozzle; (b) square nozzle; (c) beveled-edge nozzle; and (d) filleted-edge nozzle. The black dashed lines denote the jet half-width and the solid white lines denote the potential core. The white arrow indicates the vena contraction.

transverse momentum is abruptly redirected, the contraction in the filleted nozzle facilitates a smoother, more gradual re-orientation of transverse momentum into streamwise momentum. Therefore, the flow velocity along the centerline at ($x/h = 0$) is lower than that observed in the square and beveled orifices, resulting in a longer potential core length ($x_p/h = 7.5$) compared to those two cases.

Figure 10 illustrates the normalized mean streamwise velocity U_d/U_b and streamwise turbulence intensity U'/U_b along the centerline in the X - Y plane. The mean velocity profiles [Fig. 10(a)] of the planar nozzle and the filleted orifice are quite similar, while the square and beveled orifices are similar to each other. The mean velocity profiles of the square and beveled orifices are significantly higher than the other two cases, especially near the nozzle exit. As illustrated in Fig. 9, this increase results from the vena contraction effect induced by the sharp-edged nozzle, which accelerates the flow due to a reduction in effective flow area. The velocity ratio reaches its maximum at one nozzle height downstream and then decreases. In contrast, the profile of the filleted orifice reduces this acceleration effect, having only a small velocity increase immediately after the exit, followed by a relatively constant velocity until $X/h = 3$, whereafter it decreases in a similar manner to the planar nozzle.

The streamwise turbulent intensity (U'/U_d) is shown in Fig. 10(b). The planar nozzle has a similar trend, albeit slightly higher, compared to the filleted orifice up to $X = 14h$. The slight increase in turbulence intensity at the exit of the planar nozzle ($X = 0$), compared to the filleted orifice, is due to the fully developed inlet boundary condition of the planar nozzle, which results in greater turbulence intensity at the nozzle exit. For both the square and beveled orifices, significantly higher turbulence intensity levels are observed compared

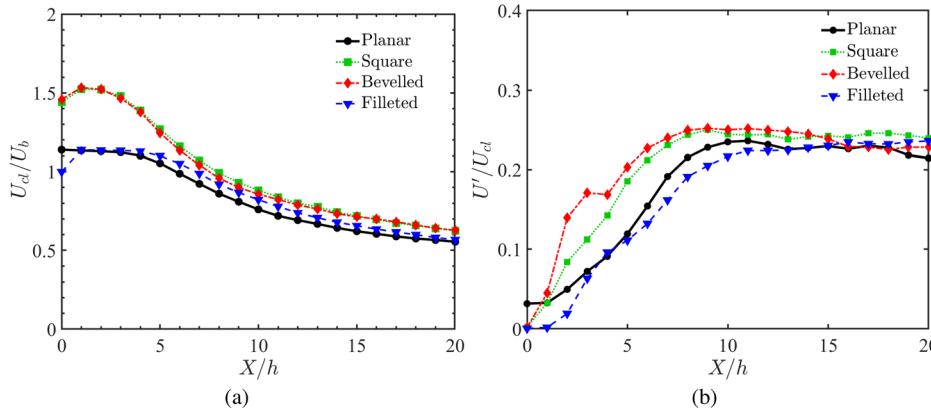


FIG. 10. Near-field evolution of the centerline velocity: (a) normalized streamwise mean velocity and (b) normalized streamwise turbulence intensity.

to the other two cases, extending downstream to $X = 20h$. Their sharp edges accelerate the flow through a vena contraction. This increase in velocity increases the velocity gradients and strain rates, thereby increasing the turbulence intensity. This is consistent with findings from the experimental research by Deo *et al.*¹⁸ Their study also demonstrated a consistent dependence of streamwise turbulence intensity on the fillet radius, with the highest turbulence intensity occurring with no fillet present, i.e., a sharp-edge case. In this present work, by comparing directly the four different nozzle types, it is shown that the square-edge case exhibits similar near-field turbulence statistics to

the beveled-edge case, while the planar nozzle closely resembles the filleted-edge case.

Figure 11 presents the near-field streamwise velocity profiles in the transverse direction at a streamwise positions $X/h = 4$ and 8 . At $X/h = 4$ [Fig. 11(a)], the values of \bar{U}/U_{cl} for the square and beveled orifices in the transverse direction are lower than the other two cases for values of η below 0.2. However, further downstream at $X/h = 8$ [Fig. 11(c)], the mean velocity becomes greater because of the rapid decay of the centerline velocity U_d . This behavior is attributed to the vena contraction effect of the sharp-edge cases, which exhibit

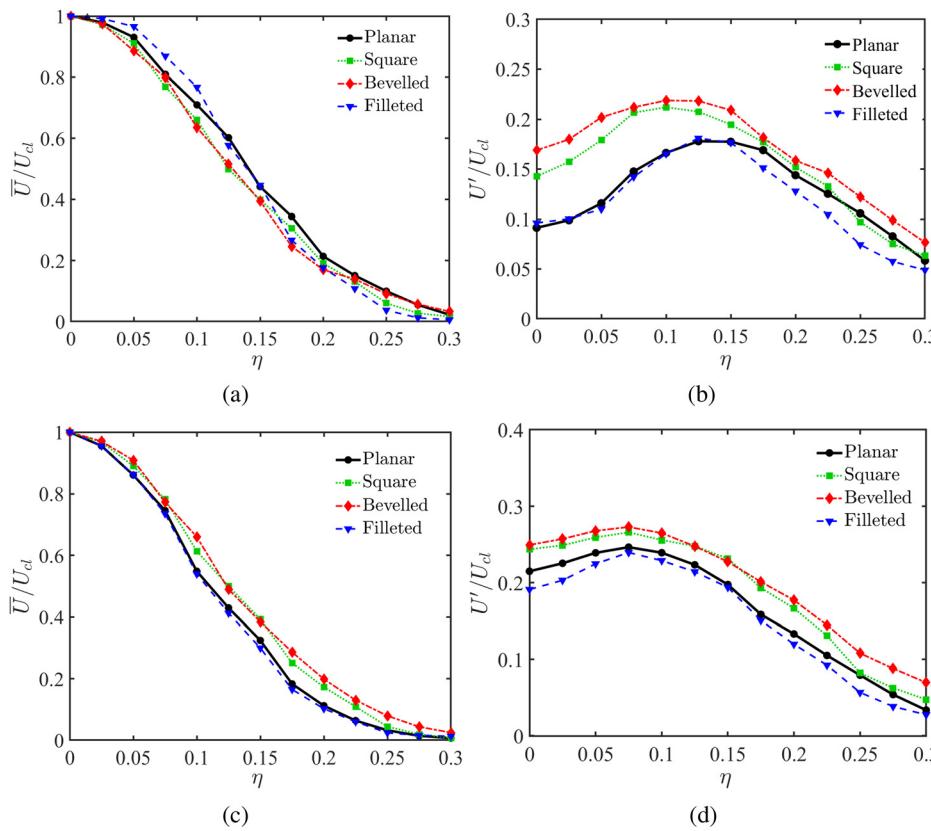


FIG. 11. Near-field streamwise velocity statistics in the transverse direction. (a) normalized streamwise mean velocity at $X/h = 4$; (b) normalized streamwise turbulence intensity at $X/h = 4$; (c) normalized streamwise mean velocity at $X/h = 8$; and (d) normalized streamwise turbulence intensity at $X/h = 8$.

significantly higher centerline velocity (U_{cl}) than the planar or filleted cases at $X/h < 5$. Further downstream, this difference becomes less and the centerline velocities converge as previously shown in Fig. 10(a).

Near the end of the potential core at $X/h = 4$ [Fig. 11(b)], the turbulence intensity peak appears closer to the centerline for the sharp-edge cases ($\eta \approx 0.1$) compared to the planar nozzle and filleted orifice ($\eta \approx 0.15$). This shift in traverse location can also be attributed to the vena contraction, resulting in a higher velocity gradient near the potential core. Further downstream at $X/h = 8$ [Fig. 11(d)], the transverse positions of turbulent intensity peaks of all cases are closer to the centerline ($\eta \approx 0.075$). The reduction in transverse positions of the peaks is due to the downstream spreading and mixing of the shear layer, which makes the turbulence intensity higher near the centerline. In both Figs. 11(b) and 11(d), the square and beveled orifices exhibit consistently higher turbulent intensity values in the transverse profiles. This indicates that the sharp edge produces a stronger shear layer than the other two types of inlet nozzle due to the vena contraction.

The normalized instantaneous vorticity magnitudes ($\omega h/U_b$) of four cases in the mid Z plane are shown in Fig. 12. Only the near field of the nozzle is presented, as the low velocity secondary flows on the outside walls of the nozzle caused by flow entrained by the jet are not of interest and only have a limited influence on the overall jet development. The vorticity magnitude (ω) is calculated as

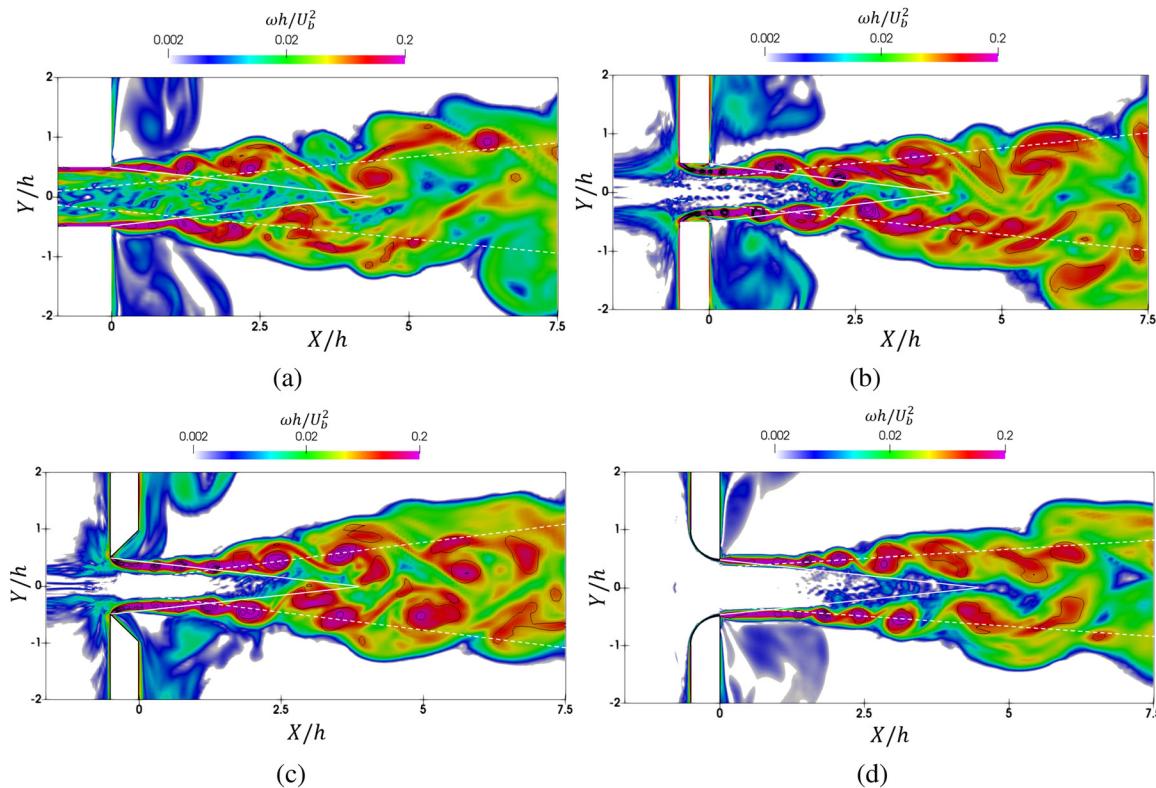


FIG. 12. Visualization of instantaneous normalized vorticity magnitude in the shear layer in the X-Y plane at the mid-span; the contour is vorticity, and the vortex is visualized by normalized Q-criterion (Qh^2/U_b^2) which ranges from 1 to 50; (a) planar nozzle; (b) square nozzle; (c) beveled-edge nozzle; and (d) filleted-edge nozzle. The dashed lines denote the $y_{0.5}$, and the solid lines denote the potential core.

$$\omega = \sqrt{\left(\frac{\partial W}{\partial Y} - \frac{\partial V}{\partial Z}\right)^2 + \left(\frac{\partial U}{\partial Z} - \frac{\partial W}{\partial X}\right)^2 + \left(\frac{\partial V}{\partial X} - \frac{\partial U}{\partial Y}\right)^2}, \quad (6)$$

where (U, V, W) is the velocity vector. When the flow separates from the edges of the jet nozzle, a free shear layer forms due to the velocity difference between the jet and the surrounding ambient fluid, resulting in a strong velocity gradient. Downstream of the nozzle exit, small perturbations arise within this shear layer. These perturbations initiate the development of Kelvin–Helmholtz instabilities, which generate a roll-up of the shear layer and emergence of coherent vortical structures as shown in Fig. 12. The shear layer initially undergoes a consistent growth in thickness characterized by the relatively orderly roll-up of the vortices. The vortex size increases as they convect downstream. As they increase in size, this leads to increased vortex interactions, breakdown of coherent vortices, and the onset of enhanced turbulent mixing near the end of the potential core.⁴³ Therefore, toward the end of the potential core region, the shear layer becomes increasingly unsteady.

This observed process of vortex formation and breakdown is consistent with the findings in the study by Ghasemi⁴⁴ and Nekkanti *et al.*⁴⁵ (although observed for different nozzle types and flow conditions). As shown previously in Fig. 9, the velocity ratio \bar{U}_a/U_b near the exit of sharp-edge cases is close to 1.5, resulting in a larger velocity gradient compared to the other two cases. This larger velocity gradient generates greater strain rates and causes earlier vortex roll-up in the

shear layer [Figs. 12(b) and 12(c)] compared to the planar nozzle and filleted orifice shear layer [Figs. 12(a) and 12(d) respectively]. Near the end of the potential core, the rolled-up vortices in the shear layer penetrate the boundary of the potential core and even extend beyond the centerline of the jet. Therefore, there is a strong interaction between both shear layers as the streamwise distance increases from the jet exit. This phenomenon is more significant for the sharp-edge orifices (square and beveled orifice) than for the other two cases. The vortex emergence position in the shear layer of the filleted orifice is delayed and is extended to approximately $X/h = 2$, as shown in Fig. 12(d). This is the longest of the four cases. This is because the filleted nozzle, with its curvature guiding the flow through the orifice exit, results in smaller velocity gradients and lower turbulence intensity in the shear layer immediately downstream of the nozzle, thereby delaying the onset of Kelvin–Helmholtz instabilities and vortex roll-up.

To quantify the jet spreading and decay, the similarity relationship for a plane jet is investigated by the following equations:

$$\left(\frac{U_b}{U_{cl}}\right)^2 = C_{1u} \left(\frac{X}{h} + C_{2u}\right), \quad (7)$$

$$\frac{y_{0.5}}{h} = K_{1u} \left(\frac{X}{h} + K_{2u}\right), \quad (8)$$

where X is the streamwise location, $y_{0.5}$ is the jet half-width, and C_{2u} and K_{2u} are the virtual origins. C_{1u} is the slope of $(U_b/U_{cl})^2$, which indicates the decay rate of the centerline velocity U_{cl} . K_{1u} is the spreading rate, which describes the decay of streamwise velocity (\bar{U}) in the transverse direction. C_{1u} and K_{1u} are both determined by the streamwise velocity at $x/h = 10$ and $x/h = 20$.

Table III summarizes the centerline velocity decay and spread ratio for different nozzle types. For reference, it also includes values from previous research from Matsuyama²² and Deo *et al.*¹⁶ The decay rate (C_{1u}) of the planar nozzle is similar to the numerical results of a planar jet reported by Matsuyama.²² The difference is approximately 6%. Meanwhile, the C_{1u} value for the filleted-edge case is similar to the experimental result of a filleted nozzle from Deo *et al.*,¹⁶ differing by approximately 4.6%.

Comparing the four different nozzle types in Table III, the decay rate C_{1u} of the planar nozzle and filleted orifice are similar, while the square and beveled orifices have lower values, which indicates a slower decay of center velocity ratio. An explanation for this is that the vena contraction effect, previously identified, increases the exit velocity in both the sharp-edged cases. This causes the velocity on the centerline to decay over a longer distance to the same velocity compared to the other two cases (planar nozzle and filleted orifice) without the strong vena contraction. For the spreading rate K_{1u} , the difference between the current result for the planar case and that reported by Matsuyama²² is 13.6%, while the difference for the filleted nozzle with

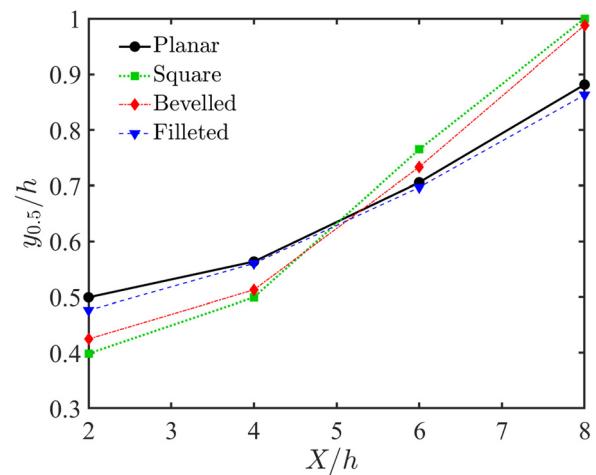


FIG. 13. Streamwise evolution of the jet half-width for different nozzle profiles.

the experimental data from Deo *et al.*¹⁶ is 16%. Again, comparing the four different nozzle types, the planar nozzle has the smallest value of spreading rate, while the square orifice has the largest. The difference between the planar nozzle and the square orifice spreading rate is approximately 18%.

Figure 13 presents the streamwise evolution of dimensionless jet half-width ($y_{0.5}/h$) in the nearfield. For $X/h < 4$, both the square and beveled orifices have smaller $y_{0.5}/h$ values because of the vena contraction effect, as illustrated in Figs. 9(b) and 9(c). However, it increases more rapidly than the planar nozzle for $X/h > 4$. This rapid increase in jet half-width results in higher spreading rates in the near field of the jet nozzle for $X/h < 10$. This enhanced spreading is attributed to greater turbulence intensity and mixing in the shear layer at the edge of the potential core, as shown previously in Figs. 11 and 12, which promotes greater momentum transfer between the jet and the surrounding quiescent fluid.

B. Jet dynamics

To investigate the time-dependent behavior of the shear layer, a monitor was positioned at $2h$ on the lip line within the shear layer to analyze periodic features in the turbulent velocity spectrum. The power spectral density (PSD) of the streamwise velocity at this location is shown in Fig. 14. For the planar nozzle, as shown in Fig. 14(a), the PSD reveals broadband humps centered around approximately 3.3 kHz, whereas the filleted orifice exhibits a more distinctive hump at 5.2 kHz. In Fig. 14(b), the spectrum of the square orifice case exhibits a broadband spectrum, except for small peaks at frequencies of

TABLE III. Summary of velocity decay (C_{1u}) and jet spreading rates (K_{1u}) for the four different nozzle types in the current simulations. The experimental data from Ref. 22 is a planar nozzle. The experimental data from Ref. 16 is a filleted nozzle.

	Sim. Planar	Sim. Square	Sim. Beveled	Sim. Filleted	Matsuyama ²² Planar	Deo <i>et al.</i> ¹⁶ Filleted
C_{1u}	0.163	0.129	0.118	0.164	0.153	0.172
K_{1u}	0.100	0.118	0.113	0.116	0.088	0.100

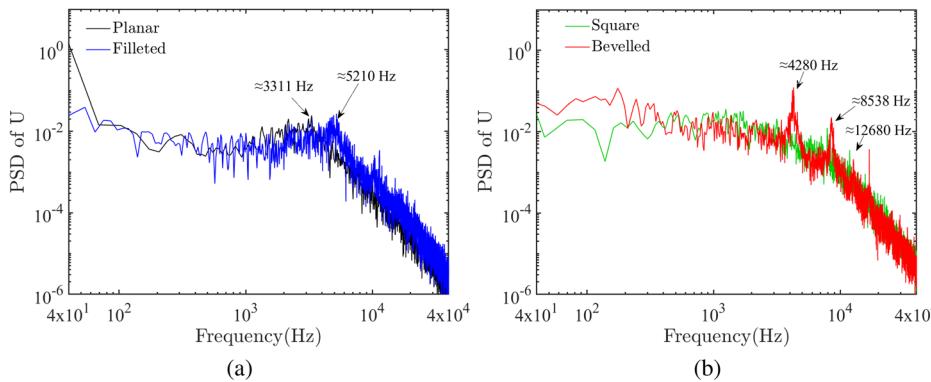


FIG. 14. PSD of the streamwise velocity. (a) Planar nozzle and filleted orifice and (b) square and bevelled orifices.

approximately 8.5 and 12.7 kHz. In contrast, the spectrum of the bevelled orifice exhibits prominent peaks at 4.3, 8.5, 12.7, and 16.9 kHz, with significantly higher amplitudes compared to the other three cases. The first peak corresponds to the fundamental vortex roll-up frequency in the jet shear layer, while the subsequent peaks represent its harmonics. The amplitudes of these harmonics are significantly lower than that of the fundamental frequency. Therefore, only the fundamental, second, and third harmonics are analyzed further. It was previously shown that the bevelled orifice case had the highest velocity gradient, resulting from the increased centerline velocity due to the strong vena contraction effect (shown in Fig. 9). This generates a shear layer for this case with larger values of vorticity (shown in Fig. 12). For this particular nozzle geometry, the amplitudes of the peaks are more pronounced compared to other cases. These tonal features align with the vortex structures depicted in Fig. 12, highlighting the direct relationship between shear layer dynamics and the spectral characteristics of the velocity. The significance of the prominent peaks in the bevelled case arises from the vortex motion and the interactions between the vortices in the shear layers, which will be discussed in the following section.

SPOD is a powerful technique for analyzing coherent flow structures in the frequency domain and their underlying dynamics. Originally proposed by Lumley,⁴⁶ it has since been further developed and applied to study dynamics in flows.^{43,47–49} SPOD allows for the

extraction of dominant coherent structures and dynamic modes in complex turbulent flows, providing insight into both the spectral characteristics and spatially coherent structures. In this study, SPOD is employed to identify the flow structures most responsible for the generation of tonal peaks in the bevelled orifice jet and to determine the spatial regions that contain these modes. To investigate the relationship between the pronounced tonal peaks in the bevelled orifice and the vortices in its shear layer, SPOD is used to visualize the coherent structures in the flow field.

Figure 15 shows the first and second modes of the fundamental frequency (4.3 kHz) and its harmonics (8.6 and 12.9 kHz) for the bevelled orifice. A dimensionless time length [Eq. (4), $t^* = tU_b/h$] of 500 of the pressure field is sampled for the calculation of SPOD. Such a length of sampling is crucial in identifying dominant modes, particularly in turbulent flows where both time and spatial coherence are significant. As shown in Fig. 15, the instability at 4.3 kHz concentrates toward the rear half of the potential core (from $1.6h$ to $4h$), and it is the most energetic in both modes 1 and 2. This region closely corresponds to the area where the vortices in the shear have grown to sufficient size that interactions between the two shear layers occur [see Fig. 12(c)]. Therefore, the peaks observed in the turbulent velocity spectrum of the bevelled orifice, as shown in Fig. 14(b), are strongly associated with vortex behavior in the free shear layers downstream of the jet exit.

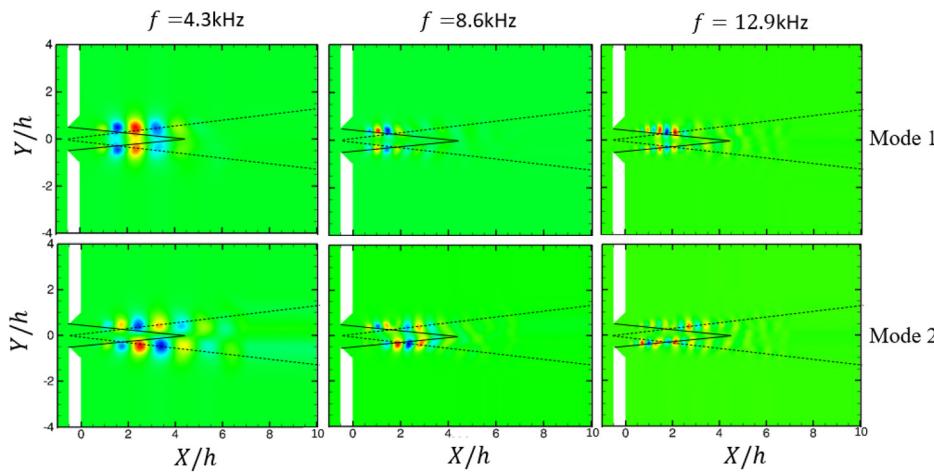


FIG. 15. SPOD modes of the bevelled orifice case. The solid black line is the potential core boundary, while the dashed black line is the jet half-width ($y = y_{0.5}$).

Similar SPOD modes are observed in the other three cases, though not shown here due to space limitations. All these modes reflect vortex dynamics within the shear layer. The same mechanism applies. However, the associated modes of spectral peaks and humps exhibit lower amplitudes compared to the beveled case. The peaks and humps in the spectra are closely associated with periodic phenomena such as vortex roll-up and vortex interactions between both shear layers. The more pronounced peaks observed in the spectra for the beveled orifice are likely attributed to the stronger shear layers generated by the vena contraction effect, which are also closer together due to the strong vena contraction effect (see Fig. 12). This results in stronger interactions compared to the other cases.

Figure 16 illustrates the instantaneous vorticity field of the beveled case at four different instants in time. The dimensionless time interval is defined as $\Delta t^* = \Delta t U_b / h$, where Δt is the physical time difference. The first snapshot corresponds to $t = 0.327$ s. As highlighted by the white arrows, the transverse distance between the vortices in both shear layers of the beveled nozzle at $X = 2h$ periodically reaches a minimum. The time duration of this process is $\Delta t^* = 2.4$. The vortices from both shear layers appear almost simultaneously at $X = 2h$, which explains this periodic reduction in distance due to their synchronized motion. Such in-phase vortex motion is observed only in

the beveled case, even though the square case exhibits a similar configuration and comparable turbulence statistics near the nozzle, as shown in Figs. 10 and 11. This difference is likely attributable to the different flow interactions governing the early stage of vortex formation in the beveled nozzle, compared to the more complex vortex roll-up dynamics present in the square nozzle, as shown in Fig. 12.

The frequency of this periodic vortex interaction can be estimated from the instantaneous vorticity field at different times to be 4.2 kHz, very close to the peak frequency observed in Fig. 14(b). Furthermore, the SPOD results in Fig. 15 show that the most energetic vortical structures occur at 4.3 kHz, indicating that the interaction frequency matches the dominant frequency of vortex interaction in this region. Although a similar vortex frequency is present in the square nozzle case, no consistent in-phase vortex motion is found. This explains why the peaks in Fig. 14(b) for the square case are weaker than those in the beveled case. The pronounced peaks observed in the streamwise turbulent velocity spectrum in the beveled nozzle case can be attributed to several interacting factors. First, the shear layer strength is enhanced due to the vena contraction effect. Second, the spacing between the upper and lower shear layers is reduced, which is also a result of the vena contraction. Finally, the sharp edge of the beveled nozzle promotes simultaneous vortex roll-up and shedding from both shear layers that

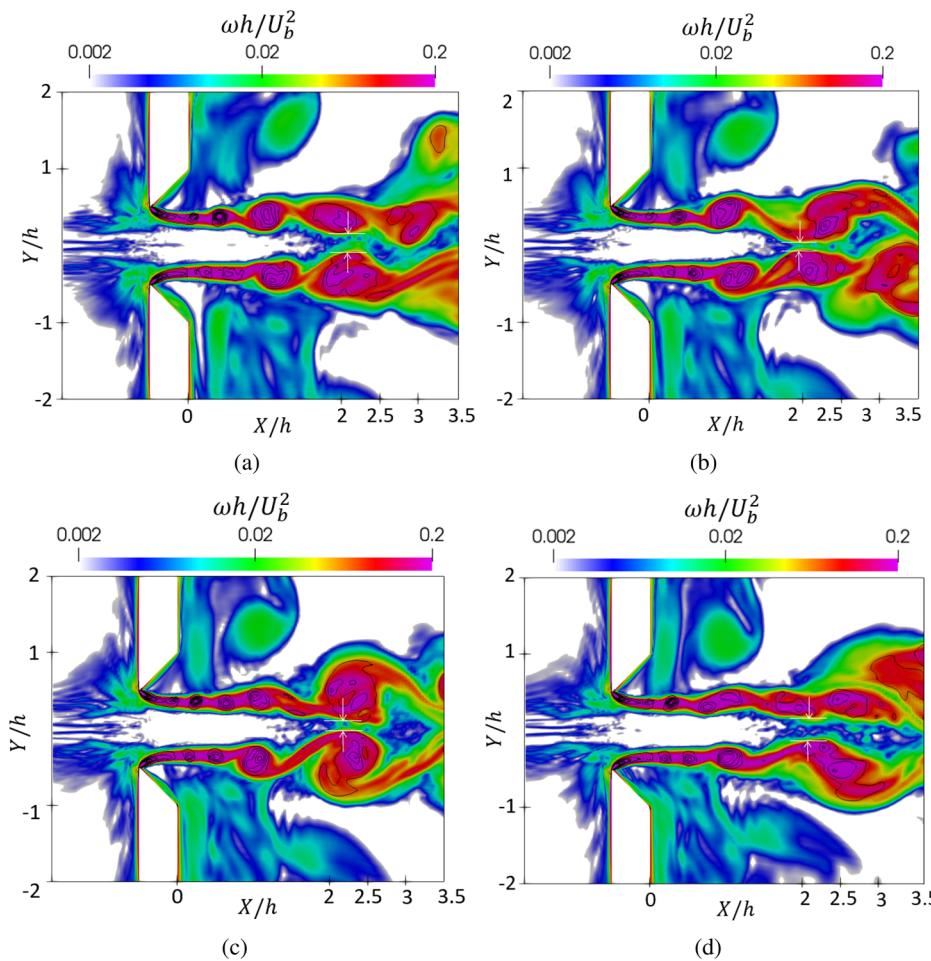


FIG. 16. Just turn to latex: The normalized vorticity field of beveled nozzle at different timesteps. (a) $\Delta t^* = 0$; (b) $\Delta t^* = 2.4$; (c) $\Delta t^* = 4.8$; and (d) $\Delta t^* = 7.2$.

are crucially in phase with each other. These combined effects contribute to the strong tones seen in the turbulent velocity spectrum.

Based on the above analysis, the tonal peaks in the spectra of square and beveled nozzles arise from vortex interactions between the two shear layers at the nozzle lips, which is enhanced by their sharp edges. This explains why the planar and filleted nozzles, despite generating vortex roll-up in the detached shear layers, exhibit weaker peaks in the turbulent velocity spectrum. Sharp-edged nozzles are thus suitable for applications requiring high centerline velocity and turbulent mixing under a fixed mass flow rate. In contrast, planar and filleted nozzles are preferable where lower flow-induced disturbances and controlled momentum flux are required.

C. Aerodynamic noise

The aeroacoustic characteristics of the four cases are analyzed to evaluate the influence of nozzle profile on noise generation. To

visualize the acoustic propagation in the flow field, the contour of the divergence of the time-derivative of the velocity field ($\nabla \cdot \dot{\mathbf{U}}$) for the four cases is presented in Fig. 17. The field contains both hydrodynamic and acoustic perturbations, which propagate at different speeds to the far field. The acoustic waves are identified through animations of $\nabla \cdot \dot{\mathbf{U}}$ field. In all cases, wave fronts can be observed in the region $X/h < 15$, as indicated by the white arrows, propagating away from the jet toward the far field. The center of these wave fronts is located near the jet centerline, specifically in the region of $X/h < 10$. Although this does not pinpoint the exact location of the sound generation, it strongly suggests that the noise source lies near the end of the potential core, where vortex interaction occurs, as shown in Fig. 12. This agrees with the findings of Laufer and Yen,⁵⁰ who argued that, in circular jets at low Mach numbers, noise sources are concentrated near the end of the potential core.

The results of the FW-H permeable integration surface placement study, shown in Fig. 24 in Appendix B, are applied to all the

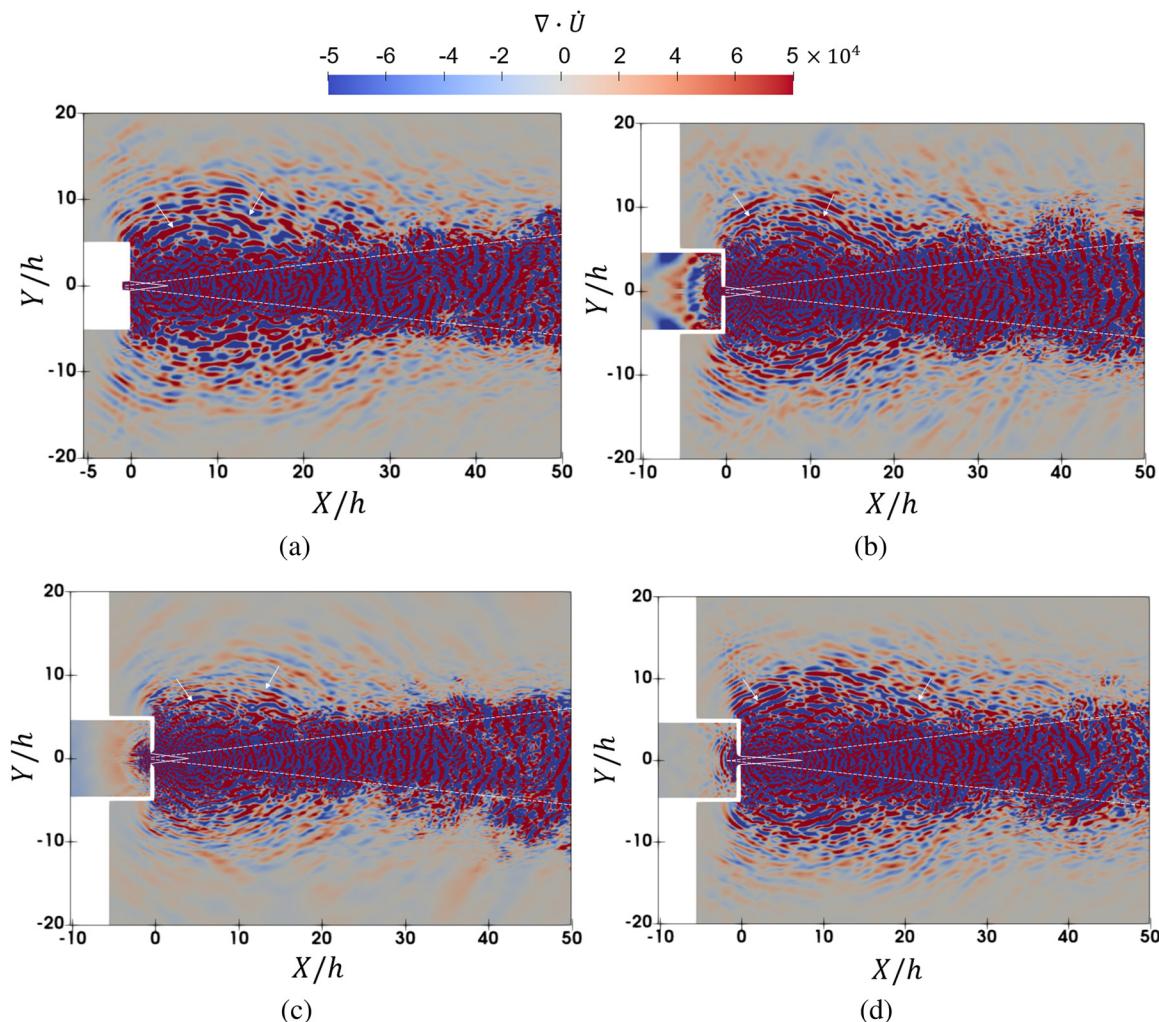


FIG. 17. Visualization of a sound wave by divergence of the time derivative of the velocity field ($\nabla \cdot \dot{\mathbf{U}}$) contour of (a) planar nozzle, (b) square orifice, (c) beveled orifice, and (d) filleted orifice. The white solid lines are the inner mixing boundary, and the dashed lines denote the jet half-width.

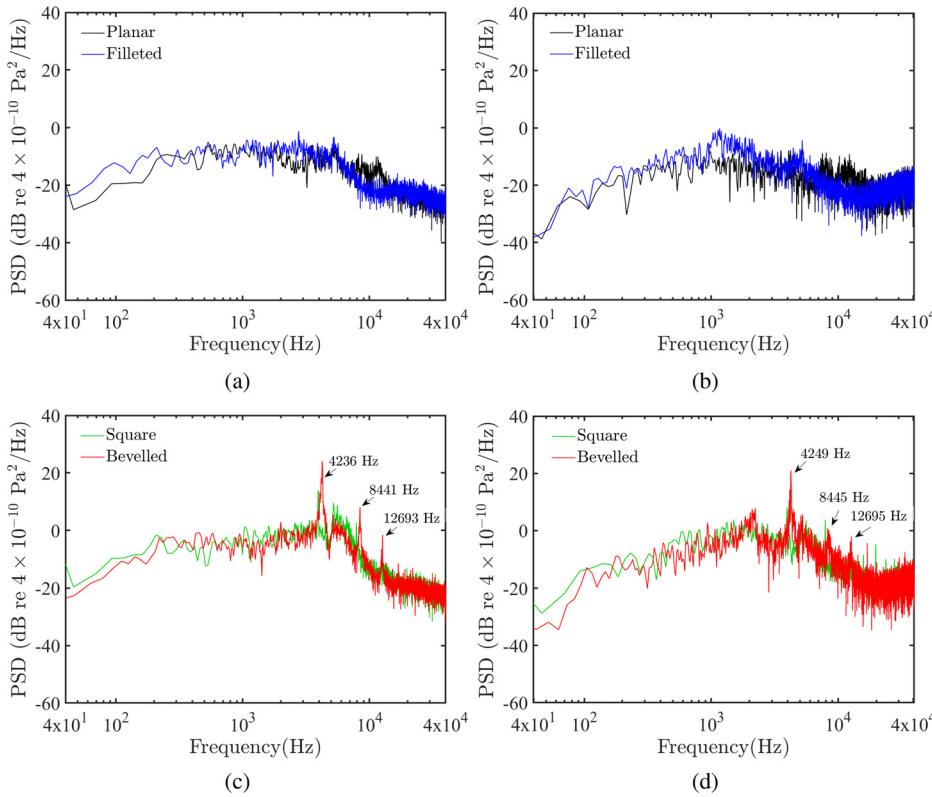


FIG. 18. Narrow-band noise spectra. (a) Planar nozzle and filleted orifice at 30° ; (b) planar nozzle and filleted orifice at 90° ; (c) square and beveled orifices at 30° ; and (d) square and beveled orifices at 90° .

acoustic calculations in this section. Nineteen receivers are uniformly positioned around the jet nozzle at angular positions ranging from $0^\circ < \theta < 180^\circ$ with a radial distance of 5 meters from the nozzle exit. Figure 18 presents the narrowband noise spectra for the four different geometries at $\theta = 30^\circ$ and 90° . Notably, the noise spectra of the planar nozzle and the filleted orifice are broadband in nature. However, distinct peaks in the acoustic spectra are observed in the square and beveled cases at frequencies of approximately 4.24, 8.44, and 12.7 kHz at both angles. These peaks, previously seen in the turbulent velocity spectrum (shown in Fig. 14), are associated with vortex roll-up and interaction in the shear layer and, importantly, propagate into the far field as sound. However, although vortex roll-up and interactions, etc.

have been qualitatively identified in all four cases, these tonal peaks are unique to cases with sharp edges and a strong vena contraction, i.e., the square and beveled orifices. This suggests that higher turbulence intensity, reduced shear layer spacing, and synchronized vortex roll-up and shedding from both lips may be responsible for triggering these tonal noise components. Nevertheless, the threshold level of these influencing factors required to generate tonal peaks needs to be investigated in future work.

Figure 19 presents noise directivity patterns for the four cases. The overall sound pressure level (OASPL) is obtained by integrating the noise over the frequency range from 20 Hz to 20 kHz. In Fig. 19(a), the integration includes the entire frequency spectrum, incorporating

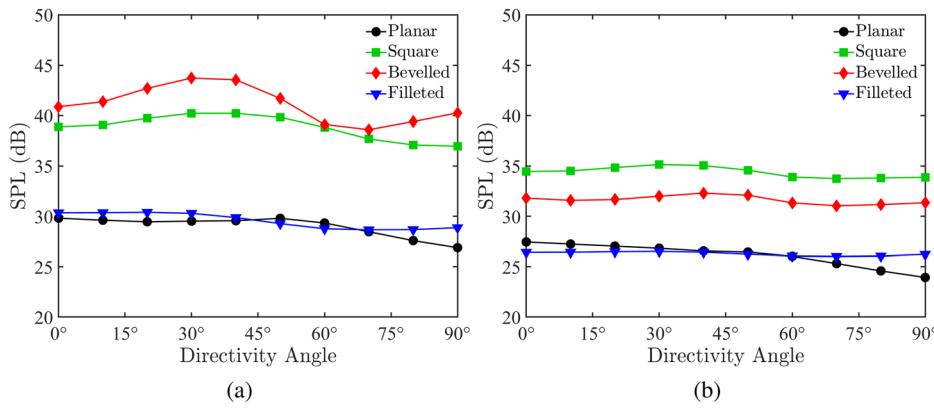


FIG. 19. Noise directivities of the planar nozzle, square, beveled, and filleted orifices. (a) Overall sound pressure level and (b) broadband overall sound pressure level with tones removed.

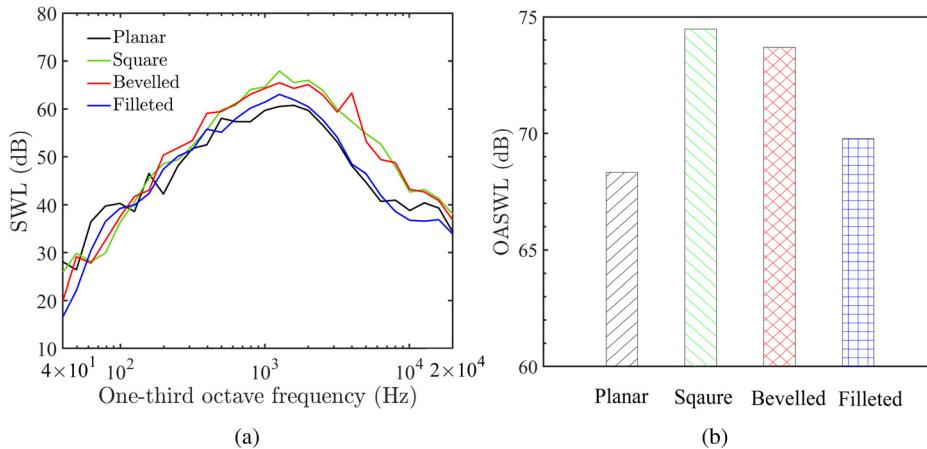


FIG. 20. Sound power levels of the four types of nozzles. (a) sound power spectra in one-third octave and (b) overall sound power level.

both broadband and tonal components, as shown in Fig. 18. In contrast, Fig. 19(b) presents the OASPL with the tonal components excluded from the integration. This shows the effect of nozzle type on the broadband noise. In Fig. 19(a), the OASPL for the beveled and square orifices is approximately 10 dB higher than that of the planar nozzle and the filleted orifice in all directions. The beveled orifice is the loudest (in terms of OASPL) due to the loud tone previously discussed. The planar nozzle and filleted orifice have similar OASPL levels from 0° to 60° . This is expected as both the planar nozzle and filleted orifice exhibit similar mean velocity profiles, turbulence intensities (in both the streamwise and transverse directions), centerline decay rates, and near-field spreading rates.

In Fig. 19(b), the OASPLs (excluding the tonal contributions) show that the broadband noise of the square orifice is the loudest configuration. This is different from the conclusion of the OASPL over the entire frequency range due to the large tone seen in the beveled orifice case. The broadband OASPL difference between the square orifice and the planar nozzle is approximately 7 dB, which is less than the OASPL when the tones are included in the integration [Fig. 19(a)]. This is due to the pronounced tonal noise peaks induced by the sharp-edge orifices, as shown previously in Fig. 18. It was shown previously that the sharp-edge cases (square and beveled orifices) result in stronger turbulence intensity compared to the planar nozzle and the filleted orifice. These instabilities in the shear layer not only generate tonal peaks in the noise spectra but also increase the broadband noise.

To assess the sound power level (SWL) emitted from the four nozzle types, receivers are placed on a sphere with a radius of 5 m surrounding the jet. The sphere is divided into 384 segments and the receivers are located at the center of the segments. The overall SWL (OASWL) is calculated by the following equations:

$$SWL_i = 10 \log_{10} \left(\sum_{j=1}^{384} \frac{p_{i,j}^2 A_j}{\rho c W_0} \right), \quad (9)$$

$$OASWL = 10 \log_{10} \left(\sum_{i=1}^{30} 10^{SWL_i/10} \right), \quad (10)$$

where SWL_i is the sound power level in the i th one-third octave band, $p_{i,j}$ is the sound pressure on the j th segment of the sphere in the i th

one-third octave band, A_j is the area of j th segment, ρ is the air density (1.225 kg/m^3), c is the sound speed (345 m/s), and W_0 is the reference sound power ($1 \times 10^{-12} \text{ W}$). Figure 20(a) presents the SWL spectra of the four cases, calculated using Eq. (9), over the frequency range from 40 Hz to 20 kHz. It can be observed that all four cases exhibit similar SWLs at frequencies below 500 Hz. However, the sharp-edge cases (beveled and square orifices) show higher SWLs than the other two configurations across most of the higher frequency range, up to 20 kHz. This phenomenon is attributed to the vena contraction effect, which accelerates the jet exit velocity up to $1.5U_b$ and increases turbulence intensity, as shown in Figs. 10 and 11, which enhances the strength of the noise sources. This also results in higher overall sound power levels (OASWLs) for the sharp-edge cases compared to the planar nozzle and filleted orifice, as shown in Fig. 20(b).

The ranking of the nozzle cases in terms of overall sound power level in Fig. 20(b) is similar to that previously detailed in the broadband OASPL with the tonal components removed [Fig. 19(b)]. While both sharp-edged cases have similar OASWL, the square orifice is 0.8 dB louder than the beveled orifice. This ranking is different compared to the overall sound pressure level in Fig. 19(a), where the beveled orifice was loudest. This difference is due to differences in the directivity of the tonal components of the sharp-edged cases. These tonal peaks are directional, and when the overall sound power is calculated over all the directivities, their relative contribution to the overall sound power level becomes less. As a result, when considering only the broadband components, the planar nozzle exhibits the lowest OASWLs (approximately 1.5 dB lower than that of the filleted nozzle). While similar, this difference can be attributed to the slight vena contraction that also occurs in the filleted nozzle, although it is much weaker than that observed in the sharp-edged cases.

V. CONCLUSION

This numerical study examined the influence of nozzle geometry on the near-field turbulence, jet dynamics, and aerodynamic noise of rectangular jets. Four configurations were analyzed. These were a planar nozzle, a square-edge orifice, a beveled-edge orifice, and a filleted-edge orifice. The mass flow rate was the same for each of the four nozzles, allowing the effect of nozzle geometry to be isolated. The conditions were a Reynolds number of $Re = 10^4$ and a Mach number

of $Ma \approx 0.17$. The DDES coupled with the FW-H acoustics solver showed good agreement with experimental data, validating the numerical framework for low-Mach number rectangular jets.

Results demonstrated that nozzle geometry significantly affects both flow development and acoustic emission. Sharp-edged nozzles (square and beveled) produce strong vena contractions, increasing the exit velocity (up to $1.5U_b$) and turbulence intensity, resulting in shorter potential cores compared to the filleted case. The filleted nozzle weakens this contraction, extending the potential core ($7.5h$ vs $4h$, for sharp-edge cases) and reducing turbulence. The planar nozzle exhibits a fully developed inlet profile and lower turbulence intensity than the sharp-edged cases. Sharp-edged nozzles generate stronger vorticity and higher spreading rates ($K_{1u} = 0.118$ vs 0.100 for the planar case). Enhanced vortex roll-up and shear-layer interaction in these cases increase turbulent mixing, rapidly broadening the jet half-width beyond $X/h > 4$.

Analysis of the turbulent velocity spectrum revealed a tonal peak at 4.3 kHz and its harmonics (8.5 and 12.7 kHz) for the beveled-edge nozzle. This tone was linked to vortex interactions between the two shear layers in the jet. For the beveled case, the vortices from both shear layers roll-up and propagate in phase between the upper and lower shear layers. Such in-phase vortex motion was only observed in the beveled case. The strong vena contraction in the beveled case resulted in stronger shear layers and reduced the distance between the two shear layers, increasing this interaction effect. The square orifice showed weaker peaks, while the planar and filleted nozzles exhibited broadband turbulent velocity spectra due to lower turbulence intensity and less coherent vortex roll-up. SPOD for the beveled case identified these tonal frequencies as coherent modes contained within the region $h < X < 4h$.

These tones in the turbulent velocity spectra propagate to the far field as acoustic waves. The beveled nozzle produces distinct tonal peaks at the same frequencies as in the turbulent velocity spectra. The square nozzle exhibits much milder tones, and the planar and filleted nozzles generate broadband noise. Sharp-edged nozzles (beveled and square) yield OASPLs approximately 10 dB higher than the others (planar and filleted), primarily due to the vena contraction effects that increase exit velocity by up to $1.5U_b$. The beveled nozzle is the loudest due to its strong tone, due to the shear layer interactions in the jet. When considering only the broadband noise, the sharp-edged nozzles remain approximately 7 dB louder compared to the filleted and planar cases. A similar trend is observed in OASWLs. Specifically, the OASWL of the square nozzle is about 0.8 dB higher than that of the

beveled nozzle, while the filleted nozzle has an OASWL approximately 1.5 dB higher than the planar case.

Overall, the findings demonstrate that nozzle geometry strongly influences both aerodynamic performance and noise generation at a constant mass flow rate. Sharp-edged nozzles enhance shear-layer mixing but produce higher noise levels, whereas planar and filleted designs offer quieter operation with reduced turbulent mixing. These results underscore the importance of nozzle geometry in balancing aerodynamic efficiency and acoustic performance. They also provide valuable guidance for optimizing nozzle profiles in applications such as aviation, high-speed transport, and HVAC systems, where minimizing noise while maintaining sufficient turbulent mixing, length of the potential core, and jet centerline velocity is essential.

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AUTHOR DECLARATIONS

Conflict of Interest

The authors have no conflicts to disclose.

Author Contributions

Yuan He: Conceptualization (equal); Methodology (equal); Writing – original draft (equal); Writing – review & editing (equal). **David Angland:** Conceptualization (equal); Methodology (equal); Supervision (equal); Writing – original draft (equal); Writing – review & editing (equal).

DATA AVAILABILITY

The data that support the findings of this study are available at the University of Southampton repository at <https://doi.org/10.5258/SOTON/D3759>.

APPENDIX A: ADDITIONAL DETAILS FOR THE PRECURSOR SIMULATION

The height of the channel is identical to the jet height (4 mm), and its length in the streamwise direction is 10 mm. To

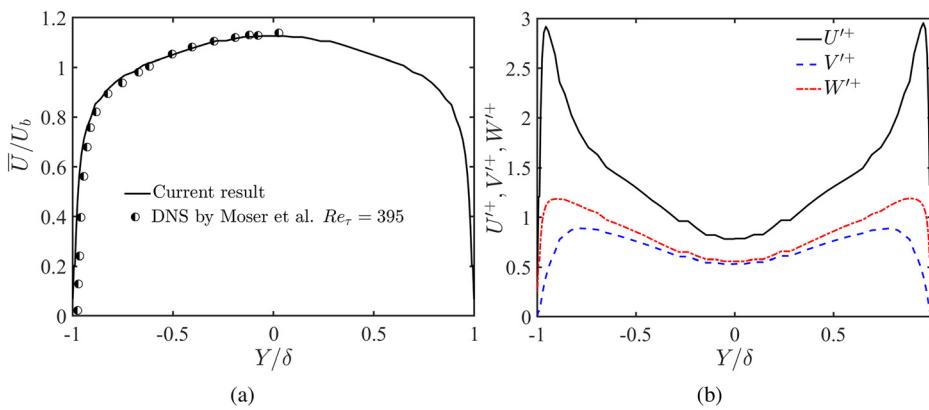


FIG. 21. Velocity profiles of the inlet velocity field for the channel flow case. (a) Dimensionless time-averaged velocity profile at the center of the channel. The experimental data are from Ref. 51; and (b) turbulence intensity profiles normalized by $u_\tau = \sqrt{\tau_w/\rho}$, where τ_w is wall shear stress and ρ is the density.

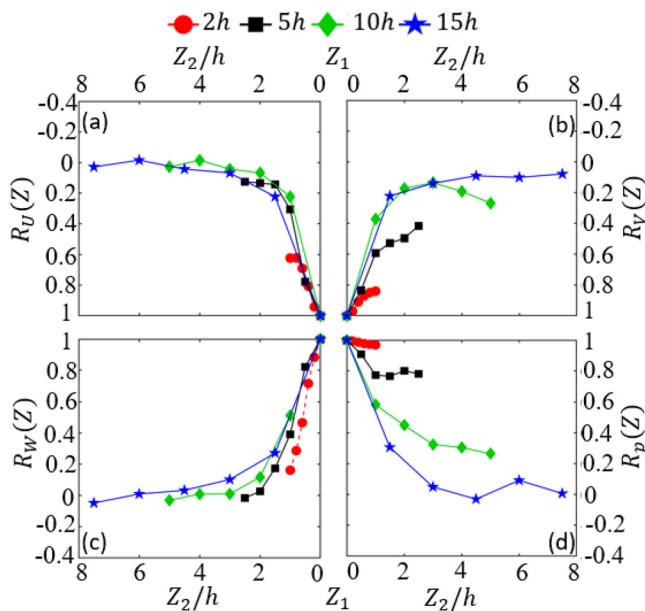


FIG. 22. Correlation coefficients of velocity and pressure with different spanwise lengths. (a) Correlation coefficient of streamwise velocity, U ; (b) correlation coefficient of transverse velocity, V (c) correlation coefficient of spanwise velocity, W ; and (d) correlation coefficient of pressure, p .

ensure the development of a fully turbulent and statistically steady channel flow, periodic boundary conditions were applied between the inlet and outlet of the channel. The resulting velocity profile is presented in Fig. 21, where the mean dimensionless streamwise velocity (\bar{U}/U_b) profile in Fig. 21(a) demonstrates excellent agreement with the DNS data for channel flow at $Re_\tau = 395$. Figure 21(b) depicts the turbulence intensity of the developed channel flow. The turbulence intensity profiles (U', V', W') are normalized by $u_\tau = \sqrt{\tau_w/\rho}$, where τ_w is wall shear stress and ρ is the air density.

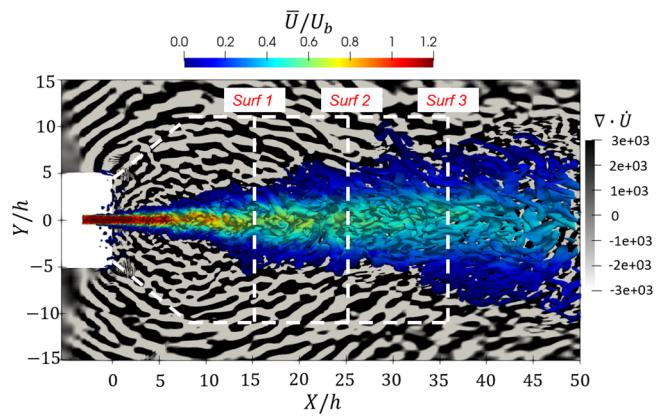


FIG. 23. The FW-H integral surface for the case with spanwise length of $10h$ is shown alongside the normalized Q criterion (0.01) and the divergence of velocity derivative $\nabla \cdot \dot{U}$.

APPENDIX B: ADDITIONAL DETAILS ON GRID RESOLUTION AND VALIDATION

To ensure that the spanwise variations are adequately captured within the chosen spanwise extent, the correlation between the mid-plane and the periodic plane should naturally decay to zero. To evaluate this, the correlation coefficient is calculated as follows:

$$R_\phi(Z) = \frac{\text{cov}(\phi_{Z_1}, \phi_{Z_2})}{\sigma(\phi_{Z_1}) \cdot \sigma(\phi_{Z_2})}, \quad (B1)$$

where $R_\phi(Z)$ is the correlation coefficient in the Z -direction, ϕ is the flow field variable, $\text{cov}(\phi_{Z_1}, \phi_{Z_2})$ is the covariance of ϕ at Z_1 and Z_2 , and $\sigma(\phi)$ represents the standard deviation of ϕ at these positions. The subscripts Z_1 and Z_2 denote different positions in the Z -direction.

To justify an adequate spanwise length of the simulations, the spanwise correlation coefficients were calculated for four cases with spanwise lengths of $2h$, $5h$, $10h$, and $15h$. Six monitors were positioned $35h$ downstream from the jet exit on the centerline,

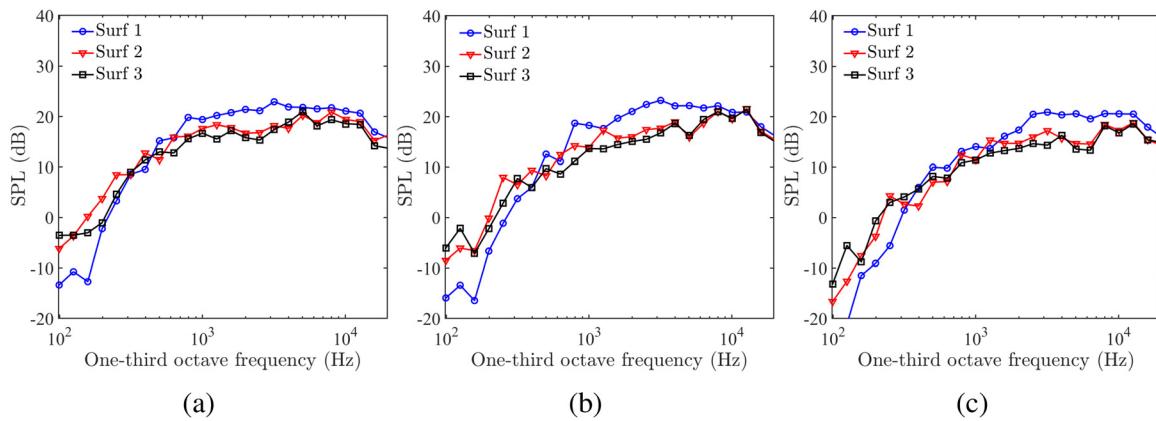


FIG. 24. Comparison of far-field noise spectra at (a) $\theta = 30^\circ$, (b) $\theta = 60^\circ$, and (c) $\theta = 90^\circ$ with different FW-H integral surfaces.

uniformly distributed from the mid-plane to the periodic plane for each case. The correlation coefficients $R_\phi(Z)$ for velocity and pressure are shown in Fig. 22. The monitor on the mid-plane is denoted as Z_1 , while Z_2 represents the positions of the remaining monitors along the spanwise direction. As shown in Fig. 22, $R_\phi(Z)$ decays as Z_2 approaches the periodic plane for all cases. However, for the cases with spanwise lengths of $2h$ and $5h$, the values of $R_\phi(Z)$ remain significantly above zero when Z_2 is located on the periodic plane.

Figure 23 shows the arrangement of the FW-H permeable integration surfaces placement study. The permeable surfaces fully enclose the turbulent wake of the jet until $x/h = 35$, ensuring the primary noise-generating regions are captured. The positions of the integral surface are carefully placed within the refined computational region. This refinement region can resolve frequencies up to approximately 20 kHz, assuming 20 points per wavelength. The receiver is placed on the mid-plane at the position ($r = 5$ m and $\theta = 60^\circ$) (see Fig. 3).

Figure 24 compares the noise spectra of three different permeable FW-H surfaces of the case with a spanwise length of $10h$. The results presented here correspond to a receiver at $\theta = 30^\circ$, $\theta = 60^\circ$ and $\theta = 90^\circ$. The results show that the noise spectra of case Surf 2 and Surf 3 are similar, whereas Surf 1 exhibits significant deviations in both the low and high-frequency ranges. This discrepancy can be attributed to the fact that Surf 2 and Surf 3 encompass more acoustic noise sources.

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